

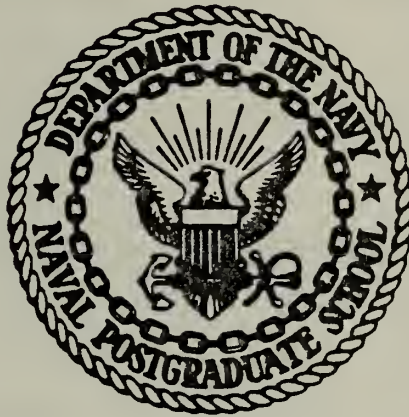
TIME DEPENDENT HOLOGRAPHIC INTERFEROMETRY  
AND FINITE-ELEMENT ANALYSIS  
OF HEAT TRANSFER WITHIN A  
RECTANGULAR ENCLOSURE

Gerald Paul Braun

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## Monterey, California



# THESIS

TIME DEPENDENT HOLOGRAPHIC INTERFEROMETRY  
AND FINITE-ELEMENT ANALYSIS  
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RECTANGULAR ENCLOSURE

by

Gerald Paul Braun

September 1976

Thesis Advisor:

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## 20. Abstract (cont'd)

encountered during this phase of research are presented with appropriate comments.



Time Dependent Holographic Interferometry and  
Finite-Element Analysis of Heat Transfer  
within a Rectangular Enclosure

by

Gerald Paul Braun  
Lieutenant, United States Navy  
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## ABSTRACT

In this thesis, the finite-element method was developed to numerically analyze heat transfer by laminar natural convection within a rectangular cavity, a classical fluid flow problem. A second auxiliary case study involving Couette flow was included to test the flexibility of this analysis technique.

Analyzing heat flows experimentally was also explored utilizing holographic interferometry. Specific problems encountered during this phase of research are presented with appropriate comments.



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## NOMENCLATURE

$C_p$	- specific heat of fluid
$D$	- width of the enclosure
$\Delta$	- area of element triangle
$g$	- gravitational acceleration
$Gr_L$	- Grashof number in L direction = $\frac{gBL^3(T_H-T_C)}{\nu^2}$
$L$	- height of the enclosure
$L_i$	- natural coordinates
$N_i$	- interpolation functions
$P$	- pressure (either wall or fluid)
$Pr$	- Prandtl number = $\frac{\nu}{\alpha}$
$Ra_D$	- Rayleigh number in D direction = $\frac{gBD^3(T_H-T_m)}{\nu}$
$Ra_L$	- Rayleigh number in L direction = $\frac{gBL^3(T_H-T_C)}{\nu}$
$T$	- temperature (either wall or fluid)
$t$	- time relative to beginning of solution
$T_C$	- temperature at cold wall
$T_H$	- temperature at hot wall
$T_m$	- mean temperature of fluid = $\frac{(T_H+T_C)}{2}$
$u$	- velocity in x-direction
$v$	- velocity in y-direction



- x      - independent coordinate in horizontal direction
- y      - independent coordinate in vertical direction
- $\alpha$      - thermal diffusivity of fluid =  $\frac{\kappa}{\rho C_p}$
- $\beta$       - coefficient of thermal expansion of fluid
- $\kappa$       - thermal conductivity of fluid
- $\mu$       - dynamic viscosity of fluid
- $\nu$       - kinematic viscosity of fluid =  $\frac{\mu}{\rho}$
- $\rho$       - fluid density
- $\Omega^{(e)}$  - domain of integration for element (e)



## I. INTRODUCTION

### A. HOLOGRAPHIC INTERFEROMETRY

The phenomenon of interference has had a considerable influence on the development of physics. Thomas Young's observation and explanation of the interference of the beams through two holes provided the basis for Fresnel's wave theory of light and the same experiment has been used as the foundation of modern coherence theory.

Derived from interference is the technique of interferometry, now one of the important methods of experimental physics. The father of visible-light interferometry was A. A. Michelson, who was awarded in 1907 the Nobel prize in physics for "his optical instruments of precision and the spectroscopic and metrological investigations he has executed with them." Applications to other spectral regions were more recent: the first use of interferometry in radio astronomy was reported in 1947, and infra-red interference spectroscopy was successfully employed some thirteen years later.

Ever since the wave nature of light was generally accepted, interferometry has been the primary method for making measurements with great accuracy. The very small





wavelength of light, on the order of  $5 \times 10^{-5}$  cm, and the fact that interferometric means are available for detecting changes of only a small fraction of this length, indicates the degree of accuracy which can be achieved. The widespread applications of the method attest to its general usefulness. Interferometry is used for testing optical components, optical gauging of machine tools, studying air flow in wind tunnels, and standardizing the fundamental units of length. Therefore it is understandable that any fundamental improvement or innovation in this interferometric technique would find many applications over a wide field.

Holographic interferometry is just such an innovation. Holography may be described as a photographic technique in which the amplitude and phase characteristics emanating from a coherent light source are recorded and later reproduced. This reproduction assumes the form of a three-dimensional image of the original subject. Holography has widened the scope of interferometry to such a degree that holographic interferometry is now considered a standard tool in engineering laboratories all over the world.

Conventional interferometry can be utilized to make measurements on highly polished surfaces of relatively simple shape. Holographic interferometry extends this



range by allowing measurements to be made on three-dimensional surfaces or arbitrary shape and condition. A roughly processed machine part can now be measured to optical tolerance. Furthermore, with the holographic technique a complex object can be examined interferometrically from many different perspectives, because of the three-dimensional nature of the hologram. A single interferometric hologram is equivalent to many observations with a conventional interferometer. This property is especially useful for observations of such things as fluid flow in a wind tunnel. A third departure of holographic interferometry from conventional interferometry is that an object can be interferometrically examined at two different times; one can detect with wavelength accuracy any changes undergone by an object over a period of time. The present object can thus be compared with itself as it was at an earlier time. This is a great advantage in many fields. For example, a large lens can be tested before and after mounting. Similarly, with the aid of pulsed lasers, a machine part can be interferometrically compared with itself statically as well as dynamically.

Methods of holographic interferometry include single- and double-exposure as well as pulsed laser interferometry.



In this thesis, only single-exposure holographic interferometry was considered since it corresponds to real-time interferometry, that is, a method which allows one to observe changes in a subject as they actually occur.

## B. CONCEPT AND HISTORY OF THE FINITE-ELEMENT METHOD

One must often resort to numerical procedures in order to obtain quantitative approximate solutions to linear and nonlinear problems in continuum mechanics. However, regardless of the initial assumptions and the methods used to formulate a problem, if numerical methods are employed in evaluating the results, the continuum is, in effect, approximated by a discrete model in the solution process. This observation suggests a logical alternative to the classical approach, namely, represent the continuum by a discrete model at the onset. One such approach, based on the idea of piecewise approximating continuous fields, is referred to as the finite-element method. Its simplicity and generality make it an attractive candidate for applications to a wide range of engineering problems.

Classically, the analysis of continuous systems often began with investigations of the properties of small differential elements of the continuum under investigation. Relationships were established among mean values of various





quantities associated with the infinitesimal elements, and partial differential equations or integral equations governing the behavior of the entire domain were obtained by allowing the dimensions of the elements to approach zero as the number of elements became infinitely large.

In contrast to this classical approach, the finite-element method begins with investigations of the properties of elements of finite dimensions. The equations describing the continuum may be employed in order to arrive at the properties of these elements, but the dimensions of the elements remain finite in the analysis, integrations are replaced by finite summations, and the partial differential equations of the continuous media are replaced, for example, by systems of algebraic or ordinary differential equations. The continuum with infinitely many degrees of freedom is thus represented by a discrete model possessing a finite number of degrees of freedom. Moreover, if certain completeness conditions are satisfied, then, as the number of finite elements is increased and their dimensions are decreased, the behavior of the discrete system converges to that of the continuous system. A significant feature of this procedure is that, in principle, it is applicable to the analysis of finite deformations of materially



nonlinear, nonhomogeneous bodies of any geometrical shape with arbitrary boundary conditions.

The practice of representing a structural system by a collection of discrete elements dates back to the early days of aircraft structural analysis, when wings and fuselages, for example, were treated as assemblages of stringers, skins, and shear panels. By representing a plane elastic solid as a collection of discrete elements composed of bars and beams, Hennikoff [1941] introduced his "framework method," a forerunner to the development of general discrete methods of structural mechanics. Topological properties of certain types of discrete systems were examined by Kron [1939], who developed systematic procedures for analyzing complex electrical networks and structural systems. Courant [1943] presented an approximate solution to the St. Venant torsion problem in which he approximated the warping function linearly in each of an assemblage of triangular elements and proceeded to formulate the problem using the principle of minimum potential energy. Courant's piecewise application of the Ritz method involves all the basic concepts of the procedure now known as the finite-element method. In 1954, Argyris and his collaborators began a series of papers in which they developed certain



generalizations of the linear theory of structures and presented procedures for analyzing complicated discrete structural configurations in forms easily adapted to the digital computer.

The formal presentation of the finite-element method together with the direct stiffness method for assembling elements was attributed to Turner, Clough, Martin, and Topp [1956], who employed the equations of classical elasticity to obtain properties of a triangular element for use in the analysis of plane stress problems. It was Clough [1960], who first used the term "finite elements" in a later paper devoted to plane elasticity problems.

Concepts of the method became more understandable after 1963 when Besseling [1969], Melosh [1970], Fraeys de Veubeke [1971], and Jones [1972] recognized that the finite-element method was a form of the Ritz technique and demonstrated its generality for handling elastic continuum problems. In 1965, the finite-element method received an even broader interpretation when Zienkiewicz and Cheung [1973] reported that it was applicable to all field problems which could be cast into variational form. During the late 1960's and early 1970's, while mathematicians were working on establishing errors, bounds, and convergence

The first part of the paper discusses the importance of the study and the objectives of the research.

The second part of the paper discusses the methodology used in the study and the data collection process.

The third part of the paper discusses the results of the study and the findings of the research.

The fourth part of the paper discusses the conclusions of the study and the implications of the findings.

The fifth part of the paper discusses the limitations of the study and the areas for future research.

The sixth part of the paper discusses the significance of the study and the contribution of the research.

The seventh part of the paper discusses the practical applications of the study and the recommendations for practice.

The eighth part of the paper discusses the ethical considerations of the study and the measures taken to ensure ethical standards.

The ninth part of the paper discusses the acknowledgments of the study and the contributions of the participants.

The tenth part of the paper discusses the references of the study and the sources of the information used.

The eleventh part of the paper discusses the appendices of the study and the additional information provided.

The twelfth part of the paper discusses the index of the study and the location of the information provided.



criteria for finite-element approximations, engineers and other appliers of this same method were also studying similar concepts for various problems in the area of solid mechanics.

Although a major portion of the literature written to date on the finite-element method deals with static and dynamic structural analysis, there has been a continuing steady increase in the number of applications in other fields. The goal of this thesis was to develop a computer program, utilizing the finite-element method, which could accurately analyze laminar natural convection within a vertical rectangular enclosure. The program should be able to properly analyze axisymmetric as well as two-dimensional flows.





## II. FUNDAMENTAL THEORY OF FINITE-ELEMENT ANALYSIS

In this section the fundamental theory on which the thesis was based is presented. Highlighted topics include the variational principle, some basic concepts of finite-element analysis and the Ritz technique, and finally the method of weighted residuals featuring the Galerkin criterion. The variational principle and the Galerkin method are looked at in detail in regards to the derivation of finite-element equations.

The finite-element method envisions a solution region as built up of many small, interconnected subregions or elements. Such a model of a problem gives a piecewise approximation to the governing equations. The basic premise of the finite-element method is that a solution region can be analytically modeled or approximated by replacing it with an assemblage of discrete elements. These finite-element discretization procedures reduce the problem to one of a finite number of unknowns by dividing the solution region into elements and by expressing the unknown field variable in terms of assumed approximating or interpolation functions within each element. The interpolation functions



are defined in terms of the value of the field variables at specified points called nodes or nodal points. Nodes usually lie on the element boundaries where adjacent elements are considered to be connected. In addition to boundary nodes, an element may also have a few interior nodes (although this was not the case in the choice of linear and quadratic triangular elements utilized in this thesis). The nodal values of the field variable and the interpolation functions for the elements completely define the behavior of the field variable within the elements. For the finite-element representation of a particular problem, the nodal values of the field variable become the new unknowns. Once these unknowns are found, the chosen interpolation functions define the field variable throughout the assemblage of elements.

Clearly, the nature of the solution and the degree of approximation depend not only on the size and number of the elements used, but also on the interpolation functions selected. As one would expect, functions cannot be arbitrarily chosen since certain compatibility conditions must be satisfied. Often such functions are selected so that the field variable and/or its derivatives are continuous across adjoining element boundaries. Another important



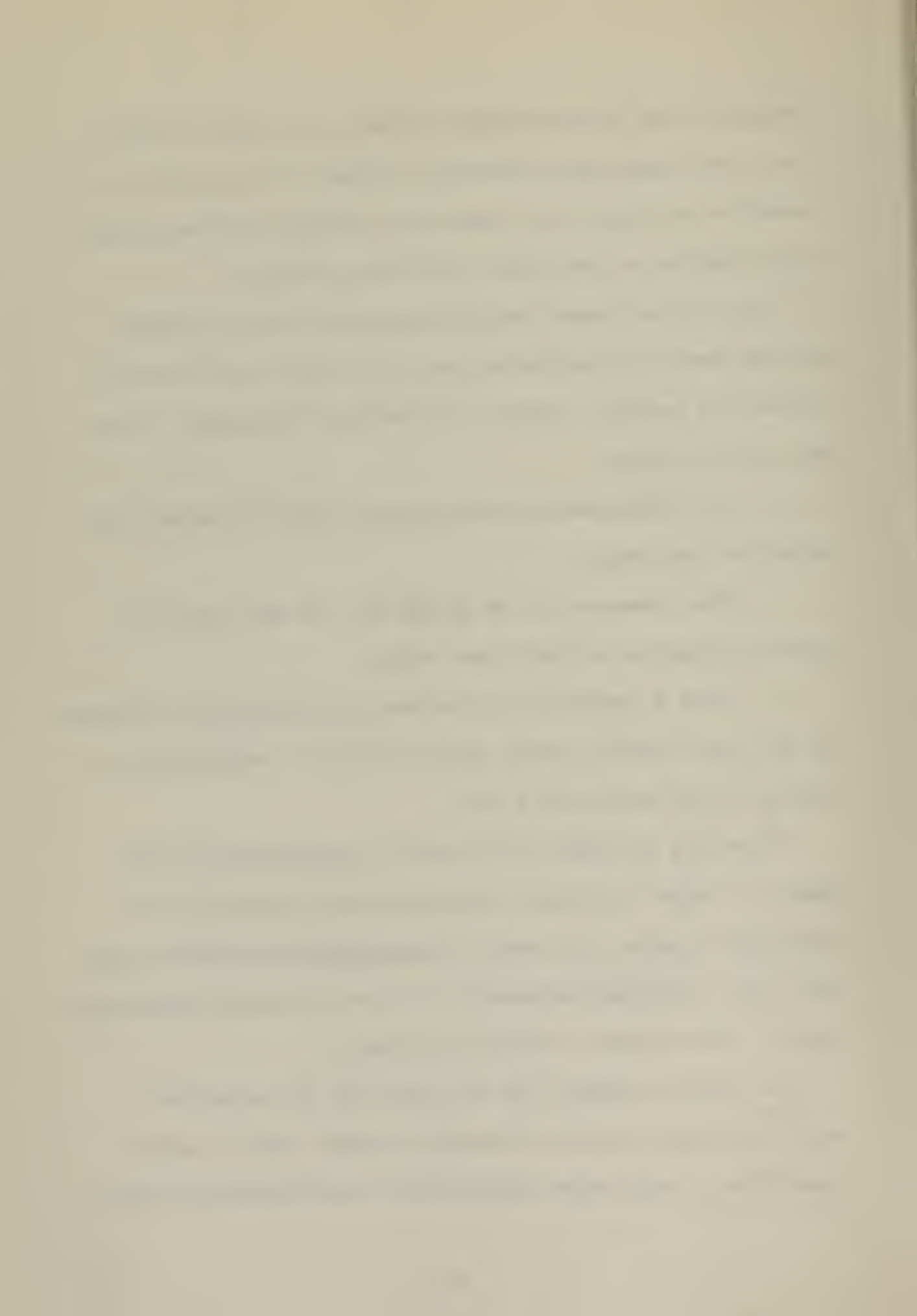
feature of the finite-element method which sets it apart from other approximate numerical methods is its ability to formulate solutions for individual elements before putting them together to represent the entire problem.

The finite-element method has gained much popularity and has been utilized extensively in recent years because it has, in general, several outstanding advantages. These are the following:

1. Non-homogeneous configurations may be treated with relative simplicity.
2. The elements can be graded in size and shape to follow boundaries of arbitrary shape.
3. Once a computer program has been developed, problems of the same variety can be solved simply by supplying the computer with appropriate data.

There are at least three distinct approaches one may employ in order to obtain finite-element equations of a particular system. In order of increasing versatility they are: (1) the direct approach, (2) the variational principle, and (3) the weighted residuals approach.

The direct approach can be used only for relatively simple problems in which discrete elements may be easily identified. Once these elements have been selected, direct





physical reasoning is introduced to establish the element equations in terms of pertinent variables. The final step is then to combine the element equations to form the governing equations of the complete system.

A detailed explanation of the remaining two approaches will be given in Subsections A, B and C to follow.

Whichever one of these three particular approaches is utilized, the finite-element method follows a systematic step-by-step process when applied to continuum problems. They are:

1. Discretize the continuum.

The entire flow region under study is divided into a series of subregions or elements assumed to be interconnected at a finite number of nodal points; thus a program originally exhibiting an infinite number of degrees of freedom is made finite. The elements used can be triangular, rectangular, or almost any shape. Also, information must be fed into a computer giving global coordinates of the nodes and topology of the system.

Finally, selection of which field variables are to be used to satisfactorily describe solution domain must be indicated at this point in the process.



## 2. Select interpolation functions, $N_i(e)$

From the nodal values one represents the value of the field variable over the element by means of interpolation functions. Often, although not always, polynomials are selected as these functions because they are easy to integrate and differentiate. The number of nodes and the order of the interpolation polynomials are interrelated. The field variable itself may be a scalar, a vector, or a higher-order tensor.

## 3. Find the element properties.

Essentially, the problem is solved at the element level. The matrix equations expressing the properties of the individual elements are determined. This can be accomplished by any one of the three approaches previously mentioned: the direct method, the variational principle, or the weighted residual method. The approach used depends entirely on the nature of the particular problem.

## 4. Assemble the element properties to obtain the system equations.

In this step, one combines the matrix equations expressing the behavior of the elements to form the matrix equations expressing the behavior of the entire solution region or system. The matrix equations for the system



exhibit the same form as the equations for an individual element except that they contain many more terms because they include all the nodes. The basis for this assembly procedure stems from the fact that, at a node where elements are interconnected, the value of the field variable is the same for each element sharing that node.

5. Solve the system equations.

From the previous step, a set of simultaneous equations are derived which can now be solved to obtain the unknown nodal values of the field variable. If these equations are linear, a number of standard solution techniques may be employed; if the equations are nonlinear, their solution is more difficult to obtain, but several alternative approaches do exist that lead to satisfactory results.

6. Make additional computations if desired.

It may be desired to use the solution of the system equations to calculate other important parameters, i.e., from the nodal values of the pressure, one might wish to calculate velocity distributions.

It is worth making mention of the fact that several of the steps in the above process are essentially the same regardless of the type of problem (this thesis was devoted





to the fluid mechanics problem). Thus, only steps three (3) and six (6) might differ for any given situation, in that the equations describing the elements could vary. The other steps would be the same. This generality of the finite-element method is, without doubt, one of its greatest strengths.

#### A. VARIATIONAL PRINCIPLE

Often, continuum problems have different but yet equivalent formulations, such as a differential formulation and a variational formulation. In the differential case, the problem is to integrate a differential equation or a system of differential equations subject to given boundary conditions. In the classical variational formulation, the problem is to find the unknown function or functions which extremize (maximize, minimize) or make stationary a functional or system of functionals subject to the same specified boundary conditions. The two problem formulations are equivalent because the functions that satisfy the differential equations and their boundary conditions also extremize or make stationary the functionals. This equivalence is apparent from the calculus of variations, which shows that the functionals are extremized or made stationary only when





one or more Euler equations and their boundary conditions are satisfied. Consequently, these equations are precisely the governing differential equations of the problem. To illustrate this duality concept, Appendix B provides a brief review and introduction to some basic ideas of the calculus of variations.

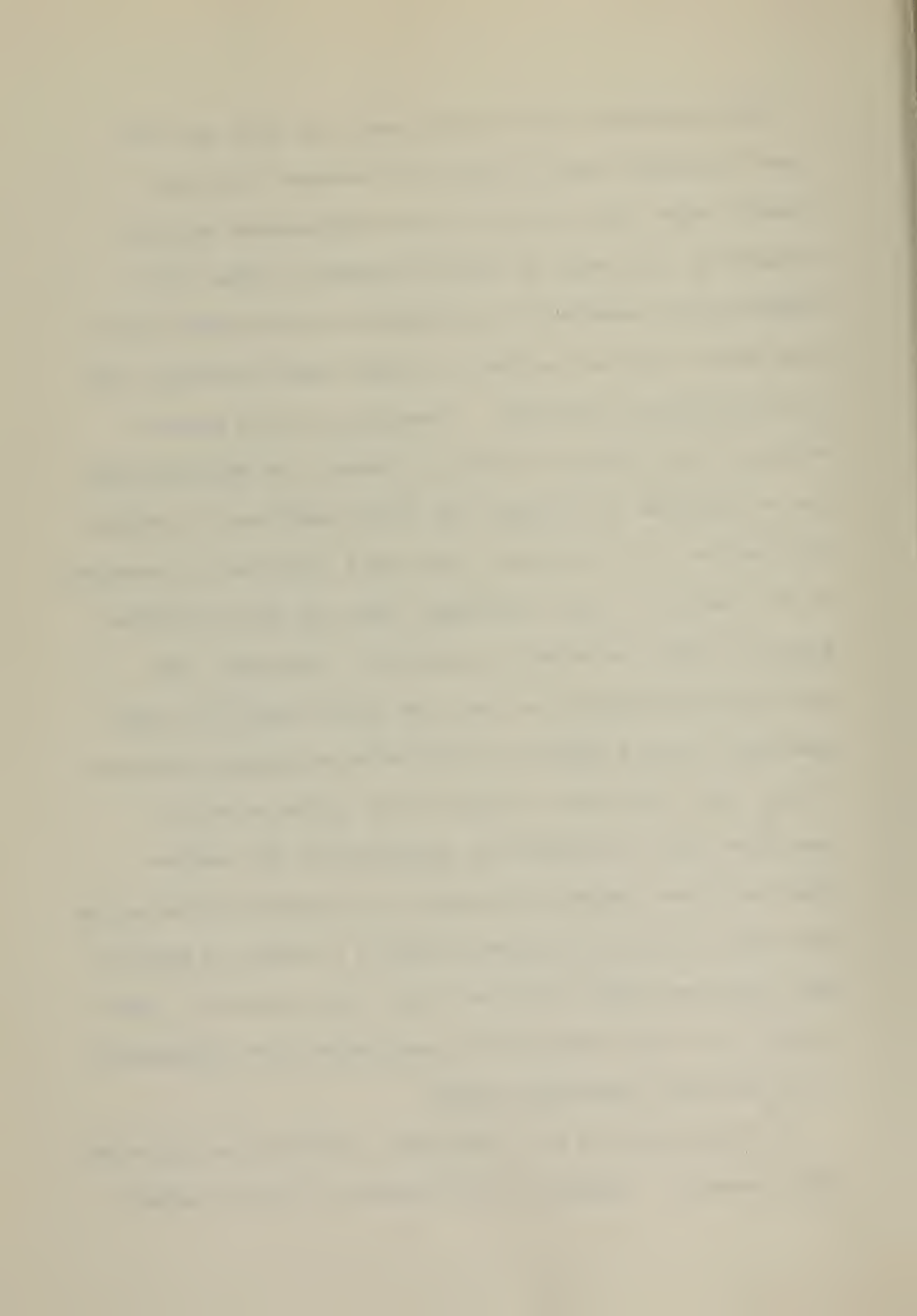
## B. FINITE-ELEMENT METHOD AND THE RITZ TECHNIQUE

The Ritz technique is basically a procedure for transforming a continuous medium into an approximated lumped parameter system. A more qualitative definition would be that the Ritz method consists of assuming the form of the unknown adjustable parameters. From this family of trial or coordinate functions, that particular function which renders the functional stationary is then selected. The procedure is to substitute the trial functions into the functional and thereby express the functional in terms of the adjustable parameters. This functional is then differentiated with respect to each parameter, and the resulting equation is set equal to zero. If there are  $n$  unknown parameters, there will be  $n$  simultaneous equations to be solved for these parameters. By this means, the approximate solution is chosen from the family of assumed solutions.



This procedure does nothing more than give one the "best" solution from the family of assumed solutions. Clearly, then, the accuracy of the approximate solution depends on the choice of trial functions. These trial functions are required to be defined over the whole solution domain and must satisfy at least some and usually all of the boundary conditions. Sometimes, if the general nature of the desired solution is known, the approximation can be improved by choosing the trial functions to reflect this nature. If, by chance, the exact solution is contained in the family of trial solutions, then the Ritz technique gives the exact solution as expected. Generally, the approximation improves as the size of the family of trial functions and the number of adjustable parameters increase. If the trial functions are part of an infinite set of functions that are capable of representing the unknown function to any degree of accuracy, the process of including more and more trial functions leads to a series of approximate solutions which converge to the true solution. Often a family of trial functions is constructed from polynomials of successively increasing degree.

To illustrate the Ritz technique, consider the following simple example. Suppose it is desired to find the general



function  $\phi(x)$  satisfying

$$\frac{d^2\phi}{dx^2} = -f(x)$$

with boundary conditions of  $\phi(a)=A$  and  $\phi(b)=B$  specified.

It is assumed that  $f(x)$  is a continuous function in the closed interval  $[a,b]$ . This problem is equivalent to finding the function  $\phi(x)$  that minimizes the functional

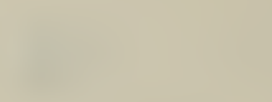
$$I(\phi) = \int_b^a \left[ \frac{1}{2} \left( \frac{d\phi}{dx} \right)^2 - f(x)\phi(x) \right] dx$$

which is of the form  $I(\phi) = \int_{x_1}^{x_2} F(x, \phi, \phi_x, f(x)) dx$

Ignoring the fact that this problem possesses an exact solution, we will attempt to find an approximate solution. According to the Ritz method, the desired solution can be assumed to be approximately represented in  $[a,b]$  by a combination of selected trial functions of the form

$$\phi(x) = C_1 \psi_1(x) + C_2 \psi_2(x) + \dots + C_n \psi_n(x) , \quad a \leq x \leq b$$

where the  $n$  constants  $C_i$  are the adjustable parameters to be determined. The trial functions should be selected so that the expression for  $\phi(x)$  satisfies the boundary conditions regardless of the choice of the constants  $C_i$ . Using



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polynomials is a simple and convenient way of constructing the trial functions. Therefore

$$\phi(x) \approx (x-a)(x-b)(C_1 + C_2x + C_3x^2 + \dots + C_nx^{n-1})$$

is a possible series of trial functions. When this approximate expression for  $\phi(x)$  is substituted into the functional to be minimized, and after the integration has been carried out, the functional is of the form

$$I = I(C_1, C_2, \dots, C_n).$$

Since the  $C_i$  are required to be chosen such that they minimize  $I$ , employing differential calculus, the following partial differential equations are formulated

$$\frac{\partial I}{\partial C_1} = 0, \frac{\partial I}{\partial C_2} = 0, \dots, \frac{\partial I}{\partial C_n} = 0$$

These  $n$  equations are then solved for the  $n$  parameters  $C_i$ , and the accuracy of the approximate solution depends on the number of  $C$ 's used in the trial function. Generally, as  $n$  increases the accuracy improves. To assess the improvement in accuracy as more  $C$ 's are utilized, the problem is solved repeatedly by taking successively more terms in the approximation, that is

January 1st 1880. The weather was very cold and the wind was very strong. The snow was very deep and the ice was very thick. The water was very cold and the air was very dry.

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$$\phi_1(x) \approx (x-a)(x-b)C_1$$

$$\phi_2(x) \approx (x-a)(x-b)(C_1+C_2x)$$

$$\phi_3(x) \approx (x-a)(x-b)(C_1+C_2x+C_3x^2)$$

and so on. By comparing the results at the end of each calculation, the effect on accuracy of adding more terms can be estimated.

The finite-element method and the Ritz technique are essentially equivalent. Each method uses a set of trial functions as the starting point for obtaining an approximate solution; both methods take linear combinations of these trial functions; and both methods seek the combination of trial functions that renders a given functional stationary. The major difference between these approximating methods stems from the fact that the assumed trial functions in the finite-element method are not defined over the entire solution domain, and they must satisfy not just any boundary conditions, but only certain continuity conditions and then only sometimes. Since the Ritz technique uses functions construed over the whole domain, it can be employed only for domains of relatively simple geometric shape. Also, these trial functions associated with the Ritz method are required to satisfy at least some and usually all of the

THE HISTORY OF THE

REIGN OF

CHARLES THE FIRST

BY JOHN BURNET, ESQ. OF THE MIDDLE TEMPLE, ESQ.

IN TWO VOLUMES. THE FIRST OF WHICH CONTAINS THE HISTORY OF THE REIGN OF CHARLES THE FIRST, FROM HIS MARRIAGE TO HIS DEATH. THE SECOND OF WHICH CONTAINS THE HISTORY OF THE REIGN OF CHARLES THE SECOND, FROM HIS RESTORATION TO HIS DEATH.

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boundary conditions. In the finite-element method the same geometric limitations exist, but only for the elements. Due to the fact that elements with simple shapes can be assembled to represent exceedingly complex geometries, the finite-element method is a far more versatile tool than the Ritz technique. From a strict mathematical standpoint, the finite-element method is a special case of the Ritz technique only when the piecewise trial functions obey certain continuity and completeness conditions that are stipulated over just the element alone.

#### C. METHOD OF WEIGHTED RESIDUALS (GALERKIN'S METHOD)

The third and final approach to the finite-element method involves a procedure that is more generalized and straightforward than either of its two predecessors.

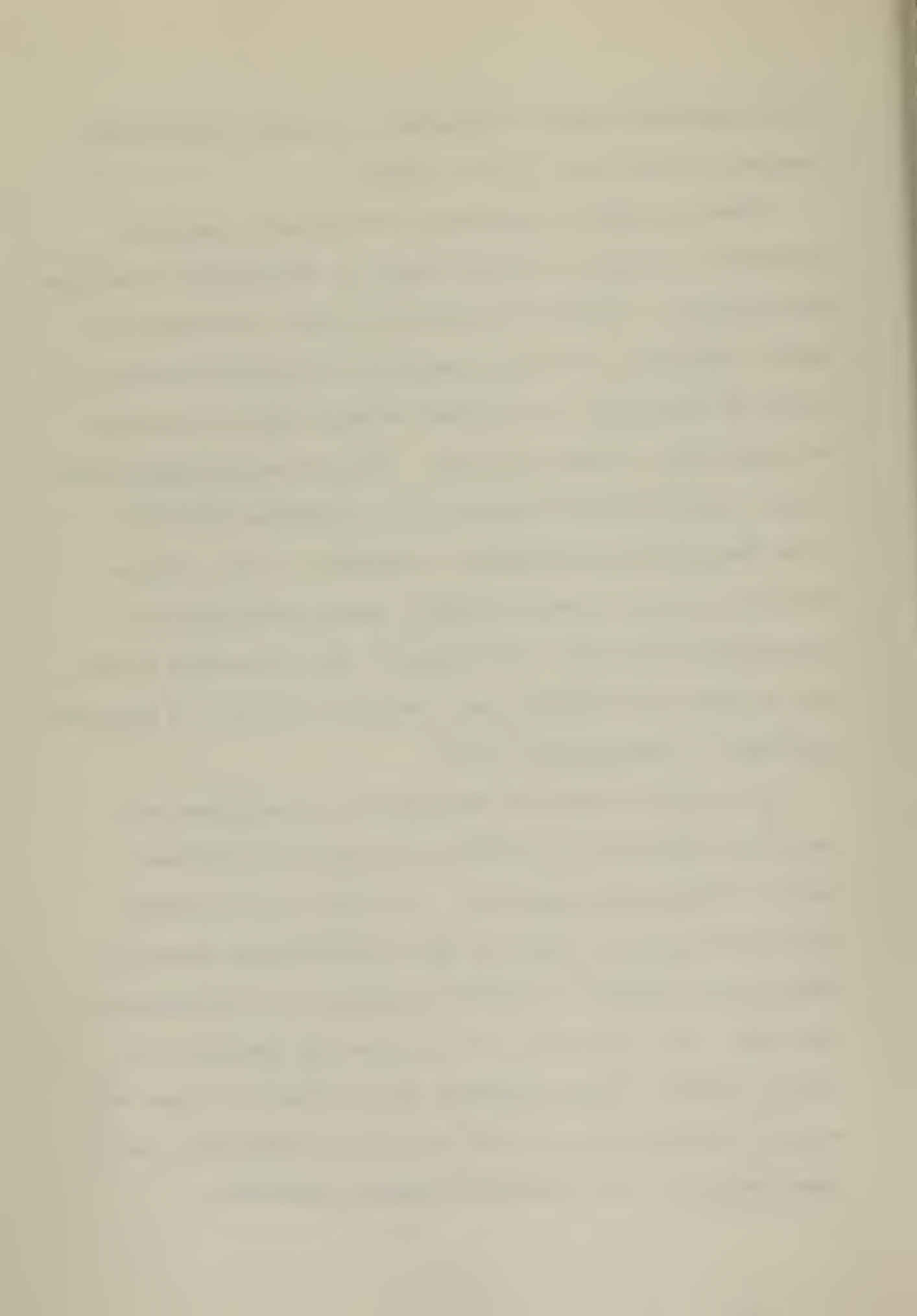
The relationship between the well-known Ritz technique and the finite-element method enables one to view the finite-element discretization procedure as simply another means for finding approximate solutions to variational problems. In fact, these finite-element equations were shown to be derived by requiring that a given functional be stationary. This broad variational interpretation is the one most widely used to derive element equations, and it is the



most convenient approach whenever a classical variational statement exists for a given problem.

However, applied scientists and engineers encounter practical problems for which classical variational principles are unknown. In these cases finite-element techniques are still applicable, but more generalized procedures characteristic of the method of weighted residuals must be employed to derive the element equations. Through certain generalizations, finite-element equations may be derived directly from the governing differential equations of the problem without reliance on any classical, quasi-variational, or restricted variational "principles." This procedure allows one to apply the finite-element method to almost all practical problems of mathematical physics.

The method of weighted residuals is a technique for obtaining approximate solutions to linear and nonlinear partial differential equations. It offers still another means with which to formulate the finite-element equations. Applying the method of weighted residuals involves basically two steps. The first step is to assume the general functional behavior of the dependent field variable in some way so as to approximately satisfy the given differential equation along with its associated boundary conditions.





Substitution of this approximation into the original differential equation and boundary conditions then results in some error called a residual. This residual is required to vanish in some average sense over the entire solution domain. The second step entails solving the equation(s) resulting from step one and thereby specializing the general functional form to a particular function, which in turn becomes the approximate solution sought.

To be more specific, the following typical problem is offered. Suppose it is desired to find an approximate functional representation for a general field variable  $\phi$  governed by the differential equation

$$\mathcal{L}(\phi) - f = 0 \quad (2.1)$$

in the domain  $D$  bounded by the surface  $\Sigma$ .  $\mathcal{L}$  is a linear or nonlinear differential operator and the function  $f$  is a known function of the independent variables. Also, proper boundary conditions are assumed to be prescribed on  $\Sigma$ . The method of weighted residuals is now applied in two steps. First, the unknown exact solution  $\phi$  is approximated by  $\hat{\phi}$ , where either the functional behavior of  $\hat{\phi}$  is completely specified in terms of unknown parameters, or the functional dependence on all but one of the independent variables is given while the functional dependence on the





remaining independent variable is left unspecified. Thus the dependent variable is approximated by

$$\phi \approx \hat{\phi} = \sum_{i=1}^m N_i C_i \quad (2.2)$$

where the  $N_i$  are the assumed functions and the  $C_i$  are either the unknown parameters or unknown functions of one of the independent variables. The  $m$  functions  $N_i$  are usually chosen to satisfy the global boundary conditions of the system in question. When  $\hat{\phi}$  is substituted into equation 2.1, it is unlikely that this equation will not be satisfied, that is,

$$\mathcal{L}(\hat{\phi}) - f \neq 0$$

but in fact,

$$\mathcal{L}(\hat{\phi}) - f = e$$

where  $e$  is the residual or error that results from approximating  $\phi$  by  $\hat{\phi}$ . The method of weighted residuals seeks to determine the  $m$  unknowns  $C_i$  in such a way that the error  $e$  over the entire solution domain is small. This is accomplished by forming a weighted average of the error and specifying that this weighted average vanish over the solution domain. In other words,  $m$  linearly independent weighting functions,  $W_i$ , are chosen such that

$$\int_D [\mathcal{L}(\hat{\phi}) - f] W_i dD = \int_D e W_i dD = 0, \quad i=1, 2, \dots, m \quad (2.3)$$



The form of the error distribution principle expressed in equation 2.3 depends on the choice of weighting functions. Once these are specified, equation 2.3 represents of a set of  $m$  equations, which may be either algebraic or ordinary differential. The second step is to solve for the  $C_i$ 's and hence obtain an approximate representation of the unknown general field variable  $\phi$  via equation 2.2. There are many linear problems and even some nonlinear problems for which it can be shown that, as  $m \rightarrow \infty$ ,  $\hat{\phi} \rightarrow \phi$ , but, in general, studies of convergence and error bounds are scarce.

Due to the broad choice of weighting functions or error distribution principles than can be used, a variety of weighted residual techniques are likewise available. The error distribution principle most often utilized to derive finite-element equations in the field of aeronautics is known as the Galerkin criterion, or Galerkin's method. Here, the weighting functions are chosen to be the same as the approximating functions employed to represent  $\phi$ , that is,  $W_i = N_i$  for  $i=1,2,\dots,m$ . Therefore Galerkin's method requires that

$$\int_D [\mathcal{L}(\hat{\phi}) - f] N_i dD = 0 \quad (2.4)$$

In the preceding section pertaining to the Ritz technique,



it was assumed that the entire solution domain was being dealt with. However, because equation 2.1 holds for any point in this region, it also holds for any collection of points defining an arbitrary subdomain or element of the whole domain. Consequently, attention may be focused directly on an individual element by means of a local approximation analogous to equation 2.2, but being defined as valid for only one element at a time. Now the finite-element representations of a general field variable become available. The functions  $N_i$  become what are known as the interpolation functions  $N_i^{(e)}$  defined over the element, and the  $C_i$  are the undetermined parameters, which may be the nodal values of the field variable or its derivatives. Then, from Galerkin's method, the equations governing the behavior of an element of the solution domain may be written as

$$\int_D \left[ \mathcal{L}(\phi^{(e)}) - f^{(e)} \right] N_i^{(e)} dD^{(e)} = 0, \quad i=1,2,\dots,r \quad (2.5)$$

where, as before, the superscript (e) restricts the range to one element, and

$$\phi^{(e)} = \left[ N^{(e)} \right] \{ \phi \}^{(e)}$$

$f^{(e)}$  = forcing function defined over element (e)

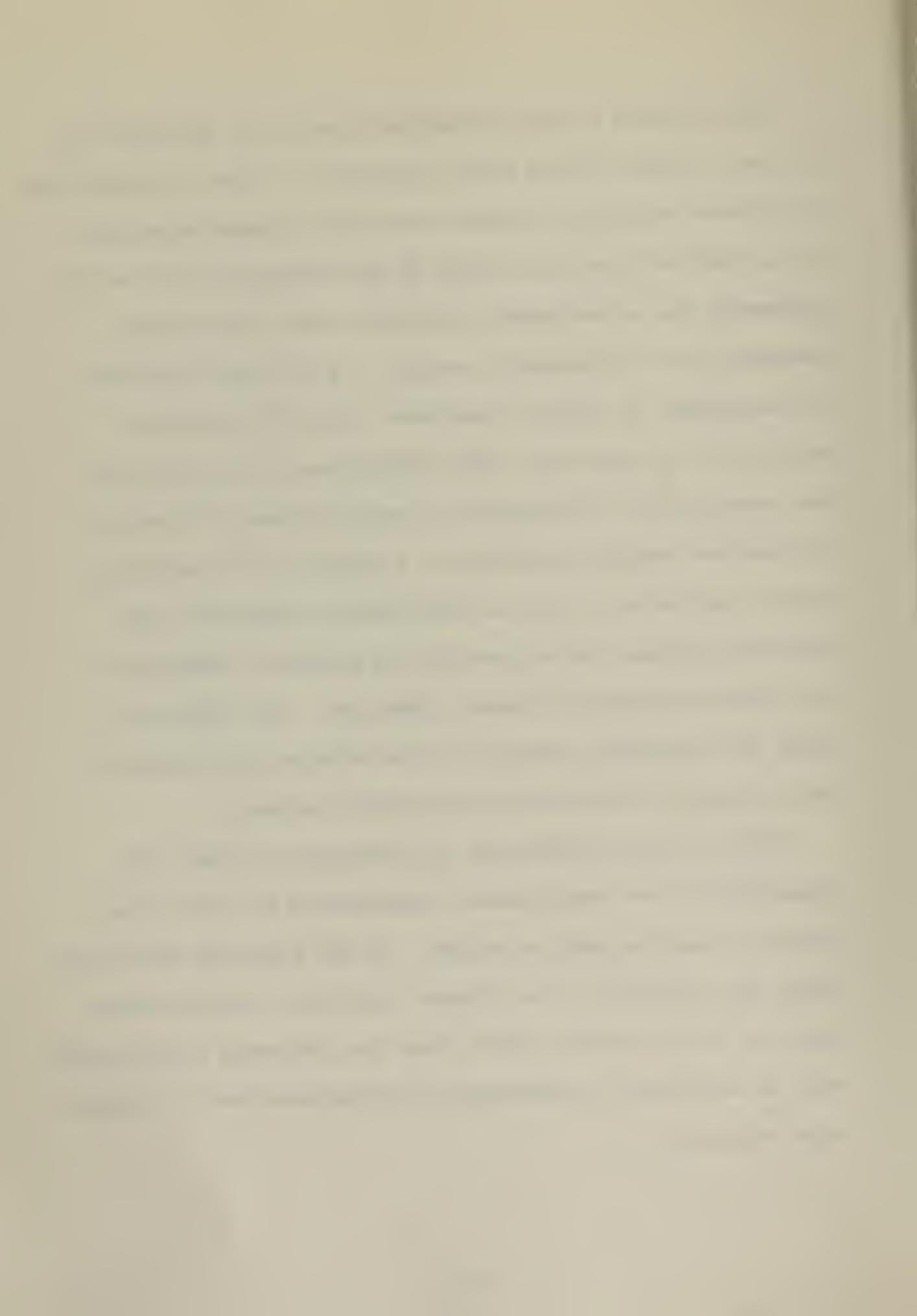
$r$  = number of unknown parameters assigned to the element.





There exists a set of equations similar to equation 2.5 for each element of the whole assemblage. Prior to assembling the system equations from the individual element equations, it is required that the choice of approximating functions  $N_i$  guarantee the interelement continuity along the boundary necessary for the assembly process. If the field variable is continuous at element interfaces, then  $C^0$  continuity exists; if, in addition, first derivatives of the variable are continuous,  $C^1$  continuity is said to occur; if second derivatives are also continuous, a region of  $C^2$  continuity exists; and so on. This is the standard definition and notation utilized for expressing the degree of continuity of a field variable at element junctions. The higher the order of continuity required in the solution, the narrower one's choice of interpolation functions becomes.

With the above definition of continuity in mind, the compatibility and completeness requirements for such interpolation functions may be stated. If the functions appearing under the integrals in the element equations contain derivatives up to the  $(n+1)$ th order, then the following stipulations must be satisfied for assurance of convergence as the element size decreases.



Compatibility requirement: At element interfaces,  $C^n$  continuity must exist.

Completeness requirement: Within an element,  $C^{n+1}$  continuity must exist. These requirements hold regardless of whether the element equations (integral expressions) were derived using the variational technique or the Galerkin method. For this thesis,  $n$  was taken to have a value of zero.

Integration by parts is a convenient way to introduce the natural boundary conditions that must be satisfied on some portion of the system exterior or boundary. Although the boundary terms containing these imposed conditions appear in the equations for each element, during the assembly of the element equations only the boundary elements give nonvanishing contributions. After the assembly process has been completed, the fixed boundary conditions (i.e., specified velocity, pressure or temperature) are conveniently introduced to help simplify the final matrix form of the finite element equation.



### III. ANALYSIS OF CONVECTIVE HEAT TRANSFER BETWEEN PARALLEL PLATES

The transfer of heat energy across a fluid layer is accomplished, in general, through the mechanisms of conduction, convection and radiation. This last phenomenon is usually a function of the fluid enclosed between the surfaces and the nature, temperature and configuration of the enclosing boundaries. Radiation takes place independently of the conduction and convection as long as there is no absorption by the fluid, and therefore under these conditions it can be considered separately. The phenomena of conduction and convection are closely interdependent and are usually analyzed together. Buoyancy forces result from differences in density within the fluid and are caused by heat transfer to or from this fluid. Natural convection may then be thought of as fluid motion of the system due to the activation of these buoyance forces. In a two-dimensional plane, such heat transfer across a vertical, enclosed fluid layer is a function of the Grashof number, the Prandtl number and the fluid layer height-to-width ratio ( $L/D$ ).





Natural convection plays a very important role in materials processing at high temperatures where agitation by other means is impracticable, or where the existence of temperature gradients is an inherent characteristic of the system.

The steady convective motion of a lubricating fluid contained within a long, rectangular enclosure was investigated. Holographic interferometry and numerical approximation were the experimental and theoretical analysis tools, respectively.

The two vertical walls of the enclosure were held at different temperatures, and the top and bottom were deemed perfect insulators (Figure 1). It was considered that the length of the enclosure (7 inches) was sufficiently long in the direction normal to the plane of Figure 1 for the motion to be assumed two-dimensional. Another assumption made was that the fluid motion was laminar. Experimental evidence indicates that such an assumption is valid provided the Rayleigh number based on cavity height is less than about  $10^8$  ( $Ra_L$  in this study was calculated to be  $1.018 \times 10^7$ ). Using this value and a value of the Prandtl number of  $1.0755 \times 10^4$ , determined from the ratio of kinematic viscosity to thermal diffusivity of the fluid, a system





Grashof number of 946.4 was calculated. The temperatures of the vertical walls  $x=0$  and  $x=D$  were defined to be  $T_H$  and  $T_C$  respectively. If  $(T_H - T_C)$  in degrees Fahrenheit is sufficiently small with respect to  $T_C$ , the Boussinesq approximation may be introduced which neglects density variations in inertia terms of the equations of motion, but retains it in the buoyancy term. One final assumption was made that all other relevant thermodynamic and transport properties were independent of temperature and that compressibility and viscous dissipation effects were negligible.

The problem now was to find the time and spatial dependence of the velocities and the temperatures within the system.

The governing differential equations expressing conservation of mass, momentum (both in x- and y-directions) and energy were

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (3.1)$$

$$\begin{aligned} \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \frac{\mu}{\rho} \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right) - \frac{1}{\rho} \frac{\partial P}{\partial x} \\ + gB(T - T_m) \end{aligned} \quad (3.2)$$

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = \frac{\mu}{\rho} \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right) - \frac{1}{\rho} \frac{\partial P}{\partial y} \quad (3.3)$$



$$\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} = \alpha \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) \quad (3.4)$$

The solution to the foregoing set of dynamic equations must satisfy the following boundary conditions on the walls,

$u_0 = v_0 = 0$ ,      no velocity on any of the four walls

$T = T_H$  or  $T_C$       given on the two vertical walls

$P = P_{ATMOS}$ ,      also given on the two vertical walls.



#### IV. THEORETICAL RESULTS

##### A. FINITE-ELEMENT ANALYSIS OF THE CONTINUUM

###### 1. Discretization of the Continuum

Since the fundamental premise of the finite-element method is that a continuum or solution domain of arbitrary shape can be accurately modeled by an assemblage of simple shapes, most finite elements are geometrically simple also. This statement especially pertains to the choice of the triangular-shaped element which would represent the unknown system parameters in this study, that is, the velocity, temperature and pressure. The main reason behind this choice was the fact that the three-node flat triangular element is the simplest two-dimensional element available, and hence an assemblage of triangles could always depict a two-dimensional domain with any number of straight sides. The solution domain in this problem was the vertical rectangular enclosure, a relatively simple-shaped continuum which posed no problem for the triangular elements. Twelve (12) elements were utilized to represent the 8.5 inch by 1.875 inch area. They were interconnected to each other and the boundary at a total of thirty-five (35) nodal points, of which twelve (12) were corner nodes (Figure 2).





Arriving at this figure of thirty-five nodes was not an arbitrary process. Once pressure was chosen to be linearly approximated, system velocities and temperature were required to assume polynomial approximation of one degree higher, or quadratic, if the highest solution accuracy was to be achieved. For triangular elements, a complete  $n$ th-order polynomial requires  $\frac{1}{2}(n+1)(n+2)$  nodes for its specification. Therefore, a 1st order, or linearly approximated, polynomial is associated with a three-node triangle; and a quadratic polynomial relates to a six-node triangle.

The three-node elements, with their nodes on the corners, may be thought of as being superimposed onto the six-node elements. Such elements contain, in addition to the corner nodes, nodes located at the midpoint of each side of the triangle. Twelve triangular elements of the six-node variety may be interconnected to form the solution domain shown in Figure 2; this domain possessing exactly thirty-five nodes.

Each element (6-node and 3-node) specifies uniquely a complete polynomial of the order necessary to give  $C^0$  continuity, and hence satisfy the completeness and compatibility requirements for elemental assemblage.



Next, the distinction between local and global node-numbering had to be made. Since each element in the triangular mesh had six nodes, the local nodes were identified as such by starting in the upper left hand corner of each element and numbering counterclockwise around the element. The global node system is a method for uniting these independent elements along with their nodes into one distinct entity. Figure 3 summarizes the relation between local and global numbering for four (4) such elements. This figure defines the system topology or the connectivity of the system.

## 2. Selection of the Interpolation Functions

In the preceding subsection it was mentioned that linear approximation was used for values representing nodal pressures, while both velocity and temperature varied in a quadratic fashion within the elements. Such a relationship was based on the governing equations of the system, in which the highest order of partial differential equations involving pressure was one, while partial derivatives of  $u$ ,  $v$  and  $T$  existed up to second order. Therefore, choosing linear pressures required the remaining three nodal parameters or field variables to take on quadratic approximation.



The functions employed to represent the behavior of these field variables within an element are known as interpolation or approximating functions. Their order within an element depends on the number of degrees of freedom assigned to that element. In this study, two different polynomial series were selected as the first and second order interpolation functions. Associated with these series were coefficients made up of generalized coordinates, that is, independent parameters which specified the magnitude of the prescribed distribution for each field variable (u, v, P, T). These polynomials were represented as follows

$$P(x,y)^{(e)} = C_1^{(e)} + C_2^{(e)}x + C_3^{(e)}y \quad (4.1)$$

for the linear pressure terms, and

$$\begin{aligned} \phi(x,y)^{(e)} = & C_1^{(e)} + C_2^{(e)}x + C_3^{(e)}y + C_4^{(e)}x^2 + \\ & C_5^{(e)}xy + C_6^{(e)}y^2 \end{aligned} \quad (4.2)$$

with  $\phi$  being a generalized quadratic field variable (either u, v, or T in this case, and the superscript 'e' standing for element.





The next step in the process was to solve for the generalized coordinate  $C_i^{(e)}$  in terms of the as yet unknown field variables. This gave the desired interpolation, but the form of the resulting equations was not convenient. As a final step then, the equations were rearranged until they appeared as

$$P(x,y)^{(e)} = N_1^P(x,y)P_1 + N_2^P(x,y)P_2 + N_3^P(x,y)P_3 = \left[ N^P \right] \left\{ P \right\} \quad (4.3)$$

and

$$\phi(x,y)^{(e)} = N_1^\phi(x,y)\phi_1 + N_2^\phi(x,y)\phi_2 + N_3^\phi(x,y)\phi_3 + N_4^\phi(x,y)\phi_4 + N_5^\phi(x,y)\phi_5 + N_6^\phi(x,y)\phi_6 = \left[ N^\phi \right] \left\{ \phi \right\} \quad (4.4)$$

where  $N_i^u$ ,  $N_i^v$ , and  $N_i^T$  were the specific interpolation functions in equation (4.4) for this study and were all equal in form, i.e.,  $N_i^u = N_i^v = N_i^T = N_i$ .

### 3. Determination of the Elemental Properties

In this thesis, the Galerkin method was utilized to determine the element properties. This procedure applied at a general node  $i$  of an isolated element becomes, in view of equations 3.1-3.4,





$$\int_{\Omega^{(e)}} H_i \left( \frac{\partial u^{(e)}}{\partial x} + \frac{\partial v^{(e)}}{\partial y} \right) dx dy = 0 \quad (4.5)$$

$$\int_{\Omega^{(e)}} W_i \left[ \frac{4}{\rho} \left( \frac{\partial^2 u^{(e)}}{\partial x^2} + \frac{\partial^2 u^{(e)}}{\partial y^2} \right) - \frac{1}{\rho} \frac{\partial P^{(e)}}{\partial x} + gB(T^{(e)} - T_m) \right. \\ \left. - u^{(e)} \frac{\partial u^{(e)}}{\partial x} - v^{(e)} \frac{\partial u^{(e)}}{\partial y} - \frac{\partial u^{(e)}}{\partial t} \right] dx dy = 0 \quad (4.6)$$

$$\int_{\Omega^{(e)}} W_i \left[ \frac{4}{\rho} \left( \frac{\partial^2 v^{(e)}}{\partial x^2} + \frac{\partial^2 v^{(e)}}{\partial y^2} \right) - \frac{1}{\rho} \frac{\partial P^{(e)}}{\partial y} - u^{(e)} \frac{\partial v^{(e)}}{\partial x} \right. \\ \left. - v^{(e)} \frac{\partial v^{(e)}}{\partial y} - \frac{\partial v^{(e)}}{\partial t} \right] dx dy = 0 \quad (4.7)$$

$$\int_{\Omega^{(e)}} W_i \left[ \alpha \left( \frac{\partial^2 T^{(e)}}{\partial x^2} + \frac{\partial^2 T^{(e)}}{\partial y^2} \right) - u^{(e)} \frac{\partial T^{(e)}}{\partial x} \right. \\ \left. - v^{(e)} \frac{\partial T^{(e)}}{\partial y} - \frac{\partial T^{(e)}}{\partial t} \right] dx dy = 0 \quad (4.8)$$

where  $W_i(x,y)$  and  $H_i(x,y)$  are the weighting or interpolation functions, which were taken as

$$W_i = N_i \text{ and } H_i = N_i^P = L_i \text{ (natural coordinates).}$$

The inertia terms in equations 4.6, 4.7 and 4.8 considerably increased the degree of difficulty of this fluid flow problem when compared to an incompressible viscous flow without inertia. This is because the above mentioned equations are nonlinear, thereby forcing an iterate procedure to be introduced and repeated until the  $u_{n+1}^{(e)}$ ,  $v_{n+1}^{(e)}$ , and  $T_{n+1}^{(e)}$



values converged to the previous  $u_n^{(e)}$ ,  $v_n^{(e)}$ , and  $T_n^{(e)}$  solutions. The subscript  $n$  runs from zero to some positive number at which the field variable passes a convergence test.

Integrating each term of equations 4.5-4.8 by parts, and making use of the approximations of equations 4.3 and 4.4, the following results on an elemental level were obtained

$$\begin{aligned} \int_{\Omega^{(e)}} (N_i^P \frac{\partial [N]}{\partial x} dx dy) \{ u \} \\ + \int_{\Omega^{(e)}} (N_i^P \frac{\partial [N]}{\partial y} dx dy) \{ v \} = 0 \end{aligned} \quad (4.9)$$

$$\begin{aligned} \int_{\Omega^{(e)}} \frac{1}{\rho} \left( \frac{\partial N_i}{\partial x} \frac{\partial [N]}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial [N]}{\partial y} \right) dx dy \{ u \} \\ - \frac{1}{\rho} \int_{\Omega^{(e)}} \frac{\partial N_i}{\partial x} [N^P] dx dy \{ P \} - gB [N_i] \{ T \} \\ - \int_{\Omega^{(e)}} (N_i [N] dx dy) \left\{ \frac{\partial u}{\partial t} \right\} = \left\{ gBT_m \right\} + \int_C N_i^* X ds \end{aligned} \quad (4.10)$$

$$\begin{aligned} \int_{\Omega^{(e)}} \frac{1}{\rho} \left( \frac{\partial N_i}{\partial x} \frac{\partial [N]}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial [N]}{\partial y} \right) dx dy \{ v \} \\ - \frac{1}{\rho} \int_{\Omega^{(e)}} \frac{\partial N_i}{\partial y} [N^P] dx dy \{ P \} - \int_{\Omega^{(e)}} (N_i [N] dx dy) \left\{ \frac{\partial v}{\partial t} \right\} \\ = \int_C N_i^* Y ds \end{aligned} \quad (4.11)$$



$$\int_{\Omega^{(e)}} \alpha \left( \frac{\partial N_i}{\partial x} \frac{\partial [N]}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial [N]}{\partial y} \right) dx dy \{ T \} - \int_{\Omega^{(e)}} (N_i [N] dx dy) \left\{ \frac{\partial T}{\partial t} \right\} = \int_C N_i Z^* ds \quad (4.12)$$

where  $N_i X^* ds$ ,  $N_i Y^* ds$  and  $N_i Z^* ds$  are simply lumped-sum contour integrals that introduce the natural boundary conditions for  $u$ ,  $v$  and  $T$  respectively. These integral values were labeled QX, QY, and QZ in the computer program. The last term on the left hand side of equations 4.10-4.12 represents the transient nature of the system.

Finally, the element matrix equations were written by inspection from equations 4.9-4.12 and were of the general form

$$[K]^{(e)} \{ \emptyset \}^{(e)} - [K_t]^{(e)} \{ \dot{\emptyset} \}^{(e)} = \{ R \}^{(e)} \quad (4.13)$$

where the square matrices  $[K]^{(e)}$  AND  $[K_t]^{(e)}$  are known as stiffness matrices, the column vectors  $\{ \emptyset \}^{(e)}$  and  $\{ \dot{\emptyset} \}^{(e)}$  are the nodal field variable and time derivative vectors, respectively. The column vector  $\{ R \}^{(e)}$  signifies the resultant nodal force vector for the element. In the actual computer program, the following identities were used

$$[K] = [TM] , \quad [K_t] = [CD] , \quad \{ \emptyset \} = \{ X \} , \quad \{ \dot{\emptyset} \} = \{ \dot{X} \} ,$$

and  $\{ R \} = \{ RHS \}$





and the element matrix equations were

$$\begin{array}{c}
 (3r+s) \times (3r+s) \quad (3r+s) \times 1 \\
 \left[ \begin{array}{c|c|c|c}
 r[K_1] & [0] & -\frac{1}{\rho}[K_2]^T & g\beta[I] \\
 \hline
 [0] & r[K_1] & -\frac{1}{\rho}[K_3]^T & [0] \\
 \hline
 -[K_2] & -[K_3] & [0] & [0] \\
 \hline
 [0] & [0] & [0] & \alpha[K_1]
 \end{array} \right] \begin{Bmatrix} \{u\}^e \\ \{v\}^e \\ \{p\}^e \\ \{T\}^e \end{Bmatrix} - \\
 \\
 \left[ \begin{array}{c|c|c|c}
 [c_D] & [0] & [0] & [0] \\
 \hline
 [0] & [c_D] & [0] & [0] \\
 \hline
 [0] & [0] & [0] & [0] \\
 \hline
 [0] & [0] & [0] & [c_D]
 \end{array} \right] \begin{Bmatrix} \{\dot{u}\}^e \\ \{\dot{v}\}^e \\ \{\dot{p}\}^e \\ \{\dot{T}\}^e \end{Bmatrix} = \begin{Bmatrix} \{AX\}^e \\ \{QY\}^e \\ \{QZC\}^e \\ \{QZ\}^e \end{Bmatrix} \\
 (3r+s) \times (3r+s) \quad (3r+s) \times 1 \quad (3r+s) \times 1
 \end{array}$$

where,  $\{AX\}^e = \{g\beta T_m + QX\}$



In the above assemblage, the individual matrix notation utilized was

$$[K_1] = K_1(i,j) = \int_{\Omega^e} \left( \frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) dx dy$$

$$[K_2]^T = K_2^T(i,j) = \int_{\Omega^e} \left( \frac{\partial N_i}{\partial x} N_j^P \right) dx dy$$

$$[K_3]^T = K_3^T(i,j) = \int_{\Omega^e} \left( \frac{\partial N_i}{\partial y} N_j^P \right) dx dy$$

$$[K_2] = K_2(i,j) = \int_{\Omega^e} \left( \frac{\partial N_j}{\partial x} N_i^P \right) dx dy$$

$$[K_3] = K_3(i,j) = \int_{\Omega^e} \left( \frac{\partial N_j}{\partial y} N_i^P \right) dx dy$$

$$[CD] = CD(i,j) = \int_{\Omega^e} N_i N_j dx dy$$

Also,  $gBT_m$  is the  $u$  velocity forcing function, and  $r$  and  $s$  are the number of nodes where velocity (or temperature) and pressure are interpolated at, respectively. In this study,  $r=6$  and  $s=3$ , therefore the element matrices were  $21 \times 21$  and the element column vectors  $21 \times 1$ .

Once the matrix equations were compiled or assembled on the element level, assembling these properties to obtain the system equations in matrix form also was a relatively simple operation for the digital computer. In essence, the large square matrices were derived by systematically adding



together the contributions of each individual element matrix, and inserting prescribed nodal variables or boundary conditions where applicable. As was brought out in the theoretical section of this thesis, the final assembly now became a system of ordinary differential equations resembling the same format as equation 4.13, i.e.

$$\left[ K \right] \left\{ \phi \right\} - \left[ K_t \right] \left\{ \dot{\phi} \right\} = \left\{ R \right\} \quad (4.14)$$

The problem solution was completed when these equations were solved for the nodal parameters  $\left\{ \phi \right\}$ , subject to the discretized initial conditions.

## B. DERIVATION OF ELEMENT MATRICES

Derivation of various element matrices, referred to previously as simply area integrals over the solution domain, will be discussed in this section. The evaluation of each matrix will be in terms of natural coordinates, that is, weighting functions relating the coordinates of the end nodes to the coordinate of any interior point belonging to the element. The weighting functions are not independent of one another, since their sum must equal unity, i.e.

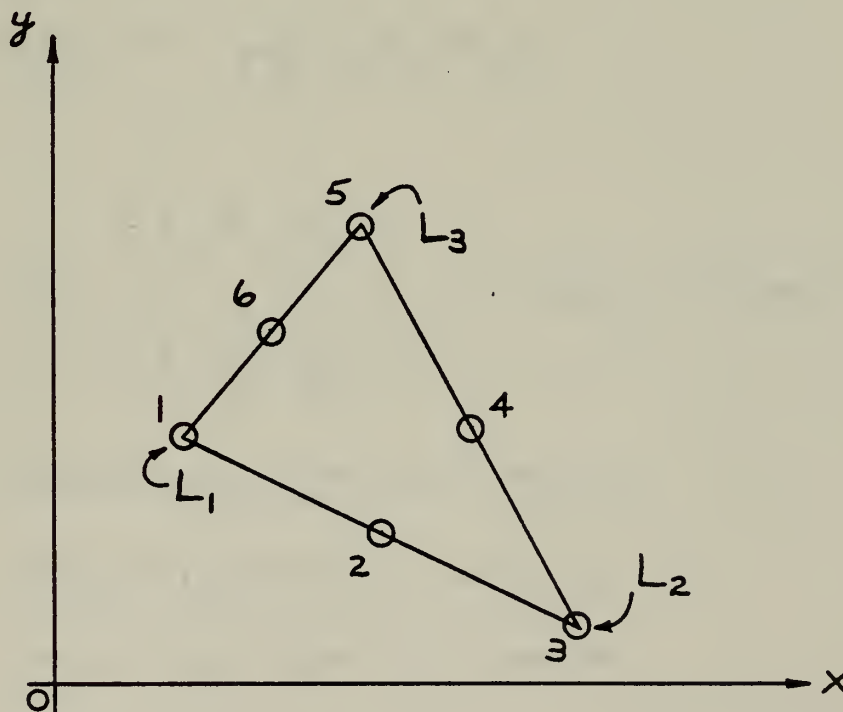
$$\sum_{i=1}^n L_i = 1 \quad (4.15)$$



where  $n$  is the number of external nodes of the element. This expression can be interpreted to mean that one and only one coordinate is associated with node  $i$ , having a unit value there and a zero value at every other node. As was previously mentioned in other sections, a general triangular shaped element, such as sketched below, was employed. Then by equation 4.15

$$L_1 + L_2 + L_3 = 1$$

A cartesian coordinate system is used since the fluid flow is assumed to be two-dimensional. Similar results could be derived using cylindrical coordinates for an axisymmetrically shaped element. The original Cartesian coordinates of a







point in the element can now be linearly related to the new natural coordinates by the equations

$$x = L_1 x_1 + L_2 x_2 + L_3 x_3 \quad (4.16)$$

and

$$y = L_1 y_1 + L_2 y_2 + L_3 y_3 \quad (4.17)$$

Solving for the natural coordinates in terms of the Cartesian coordinates gives

$$L_1(x,y) = \frac{1}{2\Delta}(a_1 + b_1 x + c_1 y) \quad (4.18a)$$

$$L_2(x,y) = \frac{1}{2\Delta}(a_2 + b_2 x + c_2 y) \quad (4.18b)$$

and finally

$$L_3(x,y) = \frac{1}{2\Delta}(a_3 + b_3 x + c_3 y) \quad (4.18c)$$

where

$$2\Delta = \begin{vmatrix} 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix} = 2 \text{ (area of triangle 1-2-3)}$$

$$a_1 = x_2 y_3 - x_3 y_2, \quad b_1 = y_2 - y_3, \quad c_1 = x_3 - x_2$$

$$a_2 = x_3 y_1 - x_1 y_3, \quad b_2 = y_3 - y_1, \quad c_2 = x_1 - x_3$$

$$a_3 = x_1 y_2 - x_2 y_1, \quad b_3 = y_1 - y_2, \quad c_3 = x_2 - x_1$$



The interpolation functions  $N_i$  for the linear pressures in terms of natural coordinates are merely

$$N_1^P = L_1, N_2^P = L_2, N_3^P = L_3$$

but those interpolation functions that relate to the velocities and temperatures stemming from quadratic approximation possess the form

$$\begin{aligned} N_1 &= 2L_1^2 - L_1 \\ N_2 &= 4L_1L_2 \\ N_3 &= 2L_2^2 - L_2 \\ N_4 &= 4L_2L_3 \\ N_5 &= 2L_3^2 - L_3 \\ N_6 &= 4L_1L_3 \end{aligned} \tag{4.19}$$

Another way of envisioning  $L_i(x,y)$  for the triangular element is to consider it a ratio of areas. Figure 4 shows how the natural coordinates, often called area coordinates, are related to areas. In this figure, when the point  $(x_p, y_p)$  is located on the boundary of the element, one of the area segments vanishes and hence the appropriate area coordinate along that particular boundary is identically zero. For example, if  $(x_p, y_p)$  is on line 1-2, then



$$L_3 = \frac{A_3}{\Delta} = 0 \quad \text{since } A_3 = 0$$

There is also a convenient analytical method for integrating area coordinates over the area of a triangular element and involves the formula

$$\int_{A(e)} L_1^\alpha L_2^\beta L_3^\gamma dA(e) = \frac{\alpha! \beta! \gamma!}{(\alpha + \beta + \gamma + 2)!} 2\Delta$$

A summation of values derived using this formula is presented in Table 4.1 for  $(\alpha + \beta + \gamma) \leq 4$ .

$$\frac{1}{\Delta} \int_{A(e)} L_1^\alpha L_2^\beta L_3^\gamma dA(e) = \frac{A}{B}$$

$\alpha + \beta + \gamma$	$\alpha$	$\beta$	$\gamma$	A	B
0	0	0	0	1	1
1	1	0	0	1	3
2	2	0	0	2	12
2	1	1	0	1	12
3	3	0	0	6	60
3	2	1	0	2	60
3	1	1	1	1	60
4	4	0	0	12	180
4	3	1	0	3	180
4	2	2	0	2	180
4	2	1	1	1	180

Table 4.1





With this preliminary work finished, the actual derivation of the element matrices may now begin. In all, five matrices will be completely evaluated while one matrix,  $[K_1]$ , will have only two of its terms derived, due to the extensive amount of time and paper needed to evaluate  $[K_1]$  in total. Beginning with this above-mentioned matrix as it appeared in Subsection A,

$$K_1(i,j) = \nu \int_{\Omega^{(e)}} \left( \frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) dx dy \quad (4.20)$$

where  $\Omega^{(e)}$  is the elemental area representing the solution domain, and  $i$  and  $j$  both vary from one to six. Since  $[K_1]$  is an array multiplying the nodal variables of velocity and temperature, it must be correlated with the quadratic interpolation functions of equation 4.19. For the point (1,1), equation 4.20 becomes

$$K_1(1,1) = \nu \int_{\Omega^{(e)}} \left( \frac{\partial N_1}{\partial x} \frac{\partial N_1}{\partial x} + \frac{\partial N_1}{\partial y} \frac{\partial N_1}{\partial y} \right) dx dy \quad (4.21)$$

where

$$\frac{\partial N_1}{\partial x} = \frac{\partial L_1}{\partial x} (4L_1 - 1) = \frac{b_1}{2\Delta} (4L_1 - 1)$$

and

$$\frac{\partial N_1}{\partial y} = \frac{\partial L_1}{\partial y} (4L_1 - 1) = \frac{c_1}{2\Delta} (4L_1 - 1)$$

substituting these values into equation 4.21



$$\begin{aligned}
K_1(1,1) &= \nu \int_{\Omega^{(e)}} \left[ \frac{b_1^2}{4\Delta^2} (4L_1 - 1)^2 + \frac{c_1^2}{4\Delta^2} (4L_1 - 1)^2 \right] dx dy \\
&= \frac{\nu(b_1^2 + c_1^2)}{4\Delta^2} \int_{\Omega^{(e)}} (16L_1^2 - 8L_1 + 1) dx dy
\end{aligned}$$

employing Table 4.1 for these three cases above in which  $(\alpha + \beta + \gamma) = 2, 1$ , and  $0$  respectively; plus the relationship that  $\int_{\Omega^{(e)}} dx dy = \Delta$

$$K_1(1,1) = \frac{\nu(b_1^2 + c_1^2)}{4\Delta^2} (16 \cdot \frac{2}{12} - 8 \cdot \frac{1}{3} + 1) \Delta$$

or finally

$$K_1(1,1) = \frac{\nu(b_1^2 + c_1^2)}{4\Delta}$$

Next, consider the point  $(2,4)$  where

$$K_1(2,4) = \nu \int_{\Omega^{(e)}} \left( \frac{\partial N_2}{\partial x} \frac{\partial N_4}{\partial x} + \frac{\partial N_2}{\partial y} \frac{\partial N_4}{\partial y} \right) dx dy \quad (4.22)$$

where

$$\begin{aligned}
\frac{\partial N_2}{\partial x} &= 4 \left[ L_1 \left( \frac{b_2}{2\Delta} \right) + L_2 \left( \frac{b_1}{2\Delta} \right) \right], \\
\frac{\partial N_2}{\partial y} &= 4 \left[ L_1 \left( \frac{c_2}{2\Delta} \right) + L_2 \left( \frac{c_1}{2\Delta} \right) \right], \\
\frac{\partial N_4}{\partial x} &= 4 \left[ L_2 \left( \frac{b_3}{2\Delta} \right) + L_3 \left( \frac{b_2}{2\Delta} \right) \right], \\
\frac{\partial N_4}{\partial y} &= 4 \left[ L_2 \left( \frac{c_3}{2\Delta} \right) + L_3 \left( \frac{c_2}{2\Delta} \right) \right]
\end{aligned}$$



substituting these four values into equation 4.22

$$K_1(2,4) = \nu \int_{\Omega(e)} \left\{ 16 \left[ L_1 L_2 \left( \frac{b_2}{2\Delta} \right) \left( \frac{b_3}{2\Delta} \right) + L_2^2 \left( \frac{b_1}{2\Delta} \right) \left( \frac{b_3}{2\Delta} \right) \right. \right. \\ \left. \left. + L_1 L_3 \left( \frac{b_2}{2\Delta} \right)^2 + L_2 L_3 \left( \frac{b_1}{2\Delta} \right) \left( \frac{b_2}{2\Delta} \right) + L_1 L_2 \left( \frac{c_2}{2\Delta} \right) \left( \frac{c_3}{2\Delta} \right) \right. \right. \\ \left. \left. + L_2^2 \left( \frac{c_1}{2\Delta} \right) \left( \frac{c_3}{2\Delta} \right) + L_1 L_3 \left( \frac{c_2}{2\Delta} \right)^2 + L_2 L_3 \left( \frac{c_1}{2\Delta} \right) \left( \frac{c_2}{2\Delta} \right) \right] \right\} dx dy$$

simplifying again, through the use of Table 4.1

$$K_1(2,4) = \frac{\nu}{3\Delta} (b_2 b_3 + c_2 c_3 + 2b_1 b_3 + 2c_1 c_3 + b_1 b_2 + c_1 c_2 + b_2^2 + c_2^2)$$

As can be seen, terms in the  $K_1$  matrix can be quite lengthy and require considerable time to derive. On a more positive note though, this square matrix is symmetric and thus only half the terms need be calculated manually.

Next, attention will be focused on the  $K_2$  and  $K_3$  matrices. These two arrays may be considered together since the only difference between the two of them is that  $b$  values are associated with  $[K_2]$  and  $c$  values with  $[K_3]$ . Otherwise, they are identical. These two matrices were given as

$$K_2(i,j) = \int_{\Omega(e)} \left( \frac{\partial N_i}{\partial x} N_j^P \right) dx dy \quad (4.23)$$

and

$$K_3(i,j) = \int_{\Omega(e)} \left( \frac{\partial N_i}{\partial y} N_j^P \right) dx dy \quad (4.24)$$



Taking the (3,6), equation 4.23 becomes

$$K_2(3,6) = \int_{\Omega^{(e)}} \left( \frac{\partial N_6}{\partial x} N_3^P \right) dx dy$$

where

$$\frac{\partial N_6}{\partial x} = 4 \left[ L_1 \frac{\partial L_3}{\partial x} + L_3 \frac{\partial L_1}{\partial x} \right] \text{ and } N_3^P = L_3, \text{ then}$$

substituting above

$$\begin{aligned} K_2(3,6) &= 4 \int_{\Omega^{(e)}} \left[ L_1 \left( \frac{b_3}{2\Delta} \right) + L_3 \left( \frac{b_1}{2\Delta} \right) \right] L_3 dx dy \\ &= \frac{2}{\Delta} \int_{\Omega^{(e)}} (L_1 L_3 b_3 + L_3^2 b_1) dx dy \end{aligned}$$

once again, using Table 4.1

$$K_2(3,6) = \frac{1}{6} (2b_1 + b_3)$$

and consequently

$$K_3(3,6) = \frac{1}{6} (2c_1 + c_3)$$

Following the same procedure throughout each of these 3x6 matrices, complete  $[K_2]$  and  $[K_3]$  are

$$[K_2] = \frac{1}{6} \begin{bmatrix} b_1 & b_1+2b_2 & 0 & b_2+b_3 & 0 & b_1+2b_3 \\ 0 & 2b_1+b_2 & b_2 & b_2+2b_3 & 0 & b_3+b_1 \\ 0 & b_1+b_2 & 0 & 2b_2+b_3 & b_3 & 2b_1+b_3 \end{bmatrix}$$





and also

$$\begin{bmatrix} K_3 \end{bmatrix} = \frac{1}{6} \begin{bmatrix} c_1 & c_1+2c_2 & 0 & c_2+c_3 & 0 & c_1+2c_3 \\ 0 & 2c_1+c_2 & c_2 & c_2+2c_3 & 0 & c_3+c_1 \\ 0 & c_1+c_2 & 0 & 2c_2+c_3 & c_3 & 2c_1+c_3 \end{bmatrix}$$

The next two matrices to be derived, that is  $\begin{bmatrix} K_2 \end{bmatrix}^T$  and  $\begin{bmatrix} K_3 \end{bmatrix}^T$ , can simply be written down by inspection of the two above arrays. Thus

$$\begin{bmatrix} K_2 \end{bmatrix}^T = \frac{1}{6} \begin{bmatrix} b_1 & 0 & 0 \\ b_1+2b_2 & 2b_1+b_2 & b_1+b_2 \\ 0 & b_2 & 0 \\ b_2+b_3 & b_2+2b_3 & 2b_2+b_3 \\ 0 & 0 & b_3 \\ b_1+2b_3 & b_3+b_1 & 2b_1+b_3 \end{bmatrix}$$

while its counterpart is then



$$[K_3]^T = \frac{1}{6} \begin{bmatrix} c_1 & 0 & 0 \\ c_1+2c_2 & 2c_1+c_2 & c_1+c_2 \\ 0 & c_2 & 0 \\ c_2+c_3 & c_2+2c_3 & 2c_2+c_3 \\ 0 & 0 & c_3 \\ c_1+2c_3 & c_3+c_1 & 2c_1+c_3 \end{bmatrix}$$

Finally, the last elemental matrix to be analyzed is the one associated with the time-dependent nodal parameters. In subsection A this matrix was given as  $[CD]$ . For convenience here, let  $[CD] = [K_t]$ , then

$$K_t(i,j) = \int_{\Omega^{(e)}} N_i N_j dx dy \quad (4.25)$$

with both  $i$  and  $j$  running from one to six. Consider, for example, point (1,5)

$$K_t(1,5) = \int_{\Omega^{(e)}} N_1 N_5 dx dy$$

substituting from equation 4.19

$$\begin{aligned} K_t(1,5) &= \int_{\Omega^{(e)}} (2L_1^2 - L_1)(2L_3^2 - L_3) dx dy \\ &= \int_{\Omega^{(e)}} (4L_1^2 L_3^2 - 2L_1 L_3^2 - 2L_1^2 L_3 + L_1 L_3) dx dy \end{aligned}$$



then from Table 4.1, using  $(\alpha + \beta + \gamma)$  four separate times, this term reduces rather easily to

$$K_t(1,5) = - \frac{\Delta}{180}$$

Factoring out a constant of  $\frac{\Delta}{180}$ , the total  $K_t$  matrix takes on the form

$$[K_t] = [CD] = \frac{\Delta}{180} \begin{bmatrix} 6 & 0 & -1 & -4 & -1 & 0 \\ 0 & 32 & 0 & 16 & -4 & 16 \\ -1 & 0 & 6 & 0 & -1 & -4 \\ -4 & 16 & 0 & 32 & 0 & 16 \\ -1 & -4 & -1 & 0 & 6 & 0 \\ 0 & 16 & -4 & 16 & 0 & 32 \end{bmatrix}$$

Which is also a symmetric matrix, thereby allowing faster derivation of the individual terms with less chance of numerical error.

### C. STRUCTURE OF COMPUTER PROGRAMS FOR FLOW ANALYSIS

A total of three computer programs analyzing two distinct test cases of fluid flow problems were employed in this thesis. The first was a steady state analysis of Couette flow. This involved determining the solution of the velocity profiles (linear and nonlinear) in a shear- and pressure-induced flow between flat parallel plates.





The upper plate slides in the positive x-direction with a constant velocity  $u$ , while the lower plate remains stationary. There is no velocity component normal to the plates, that is,  $v=0$  in the y-direction. The governing equations for this particular fluid flow are

Continuity:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (4.26)$$

Momentum:

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial P}{\partial x} + \nu \nabla^2 u \quad (4.27)$$

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial P}{\partial y} + \nu \nabla^2 v \quad (4.28)$$

For determination of the velocity profile involving only linear terms, the left hand side of equations 4.27 and 4.28 are set equal to zero (no inertia terms).

A physical representation of the Couette flow analyzed is shown in Figure 5. Node and element numbering is the same as in Figure 2. A pressure gradient of -3 units is directed along the x-axis, i.e.,  $\frac{dP}{dx} = -3$ .

The two remaining programs formed the majority of the theoretical portion of this study. They were devised to carry out the calculations for the analysis of two-dimensional or axisymmetric natural convection heat transfer problems.



The first is the lesser-complex steady state approach, whereby all transient conditions characteristic of the system are assumed to have died out, leaving only those steady state parameters remaining to be solved for. Once the finite element matrix equations describing the system are correctly assembled, a library subroutine (LEQT2F) functioning as a linear equation solver is called and the desired nodal parameters can be calculated. The second program takes into account the previously-neglected time dependence of the system by introducing a type of finite difference integration scheme to solve the transient portion of the governing equations. This integration technique must be an iterative procedure in order to circumvent the problem of nonlinearity similar to that resulting from the addition of inertia terms. Furthermore, the integral is solved at successive time steps, with time being increased until the value of the field variable converges, within tolerance, to that of its steady state counterpart. This segment of the total equation is then algebraically combined with the remaining steady state solution to yield values of nodal field variables with improved accuracy. Gravity acting in the longitudinal direction was taken into account for both the two-dimensional and axisymmetric flows.



The flow region to be studied is first defined, followed by the setting up of coordinate axes (Figure 1). The location of the origin of these axes is in most cases arbitrary, except that for a problem involving axial symmetry the x-axis must coincide with the system axis of symmetry. The flow region is then divided into a mesh of triangular elements, and the nodal points are numbered in the sequence previously described. Once the setting-up of the solution domain is completed, computer analysis of the system with its included boundary conditions can be initiated.

Structure of the steady state fluid mechanics problem will be discussed in detail, primarily because it comprises one entire program and with the addition of the transient stiffness matrix elements, accounts for the main program in the time-dependent study.

The program was coded in FORTRAN IV language and begins with a series of DIMENSION statements, which set up the arrays needed in the calculations. As indicated in statements 0260-0310, storage has been allocated for problems with up to 117 nodes; however larger problems can be considered by simply increasing the dimensions of these matrices. The limit of problem size is dictated by the





core storage of the available machine. Next, the program calls for a declaration of the type of program to be solved (either two-dimensional or axisymmetric); then the appropriate problem label is printed (statements 0360-0410). Before proceeding to input the data describing the finite element mesh, all the matrix arrays must be initialized by setting all terms in these arrays equal to zero.

Statements 0930-1150 read into the program the node numbers and the coordinates of the nodes for the complete finite element mesh. Also, the system topology, the element numbers, and the numbers of the six nodes associated with each element are read. Beginning with statement 1300, the velocity, pressure and temperature conditions within the solution domain are inserted. Correspondingly, conditions are specified for the QX, QY, QZC and QZ indices where the nodal field variables are unknown. Since the solution obtained by the program depends intimately on the body of data, the program is queried to print out all data that have been input. This enables the programmer to check for input data errors. Statement 2970 marks the completion of these steps; the program is now ready to commence work on a particular fluid mechanics problem.





The loop to begin calculating the various element matrices starts with statement 3010. Once the element matrices  $[TM\$]$  are computed for one element, they are assembled into the master or system stiffness matrix  $[TM]$  by the code followed in statements 7740-7790. The non-linear terms appearing in the velocity and temperature expressions of the governing equations are formulated in statements 4110-5900. An iterative process compares each of these terms with a corresponding quantity in the linear symmetric  $[TM\$]$  matrix until they converge in value. It is this comparison value that is then assembled with all similar values of the other elements to form the system  $[TM]$  matrix, which now exhibits or reflects the non-linearity.

Since each element in the triangular mesh has six (6) nodes, the local node numbers are  $I\$ = 1, 2, \dots, 6$ . The global node numbers for the element are recovered from the parameter  $NODE(K, I\$)$ , which was read as input data for the element; that is, for element  $K$ , the node numbers  $N(1)=NODE(K,1)$ ,  $N(2)=NODE(K,2)$ , etc. were introduced. Then the code in statements 7740-7790 loads the terms of the elemental matrices into their proper locations in the system matrices. Each time that a term of an element



matrix is placed in a location in the system matrix where another term has already been inserted, this new term is added to whatever value is there. A similar loading process takes place for the right-hand-side column vector in statements 7810-7940.

After all the elements have been processed in this fashion, the assembled system equations are ready to be modified to account for the boundary conditions or phenomena. This is done by statements 7980-8060. Thus, at the conclusion of statement 8060, the system equations possess the form

$$\begin{bmatrix} \text{TM} \end{bmatrix} \begin{Bmatrix} \text{X} \end{Bmatrix} = \begin{Bmatrix} \text{RHS} \end{Bmatrix}$$

$$\text{where } \begin{Bmatrix} \text{RHS} \end{Bmatrix} = \begin{Bmatrix} \text{Kq} \end{Bmatrix} = \begin{pmatrix} \text{QX} + \text{gBTm} \\ \text{QY} \\ \text{QZC} \\ \text{QZ} \end{pmatrix}, \text{ and } \begin{Bmatrix} \text{x} \end{Bmatrix} = \begin{pmatrix} \text{u} \\ \text{v} \\ \text{P} \\ \text{T} \end{pmatrix}$$

Not all of the components of the column vector  $\begin{Bmatrix} \text{RHS} \end{Bmatrix}$  are known because the  $Q$  values at nodes where velocity, pressure or temperature is specified are unknown; that is, at each node  $i$ , either  $u_i$ ,  $v_i$ , or  $T_i$  is known on the one side, or  $Kq_i$  is known on the other. A similar relationship exists at the corner nodes for pressure and  $QZC$  values. The only  $Q$ 's that can be specifically labeled as



heat fluxes are the QZ's, since they directly relate to temperature parameters within the system.

The only thing that remains to be done now is to call a compatible linear equation solver to produce the nodal variables sought. In this case, LEQT2F was chosen because of its speed and accuracy.

The following is a list of the symbols and descriptions utilized in coding the above program:

<u>Symbol</u>	<u>Description</u>
NCASE	interger which specifies the type of problem to be solved: NCASE=1, 2-D plane problem NCASE=2, axisymmetric problem
NN	number of nodes in solution domain
NNCN	number of corner nodes
NE	number of elements
XC(I),YC(I)	global coordinates of node I
NODE(J,I)	J=1,2,...NE; I=1,2,...6 node numbers associated with element J
NVS(I)	node number where velocity or temperature is specified
NPS(I)	node number where pressure is specified
VELU	specified nodal u velocity
VELV	specified nodal v velocity
PNP	specified nodal pressure





<u>Symbol</u>	<u>Description</u>
TNT	specified nodal temperature
NQS(I)	node number where a Q value is specified; QX and QY are specified only at internal nodes, while QZC and QZ may be specified at either external or internal nodes
QXNS	specified nodal value of QX
QYNS	specified nodal value of QY
QZCNS	specified nodal value of QZC
QZNS	specified nodal value of heat flux QZ
XC\$(I),YC\$(I)	local coordinates of node I
TM\$	element stiffness matrix
TM	system stiffness matrix
DEL	area of a triangular element

The program output begins with a statement declaring the type of problem to be solved - either nonlinear two-dimensional or axisymmetrical. Next, all input data are printed and labeled for easy identification. To ensure the validity of the solution, the printed input data should be carefully checked against the intended input. A statement following these printed data identifies which nodal parameters are associated with which system nodes (re-membering specifications of the finite element analysis called for a value of both velocities along with a temperature



at each node, while a pressure value could be defined only at the corner nodes). The complete continuum solution follows in the form of a numbered list, in which the integer appearing at the far left of this list designates the node number, or multiple of it in cases above  $I=35$ , and the figure on the right representing the value of the nodal variable in double precision.

#### D. NUMERICAL RESULTS

Complete numerical listings of the field variables for both the Couette flow problem and the steady state heat transfer problem are shown in the two computer program outputs.

The velocity profile for the linear Couette flow, i.e. node numbers 1-5, 6-10, etc., revealed that the finite-element method of analysis agreed with the exact solution of this shear-type flow out to the sixth decimal place. This is evident by the fact that all five of the FEM points lie exactly on the smooth curve depicting the exact solution in Figure 15.

The approximate steady state isotherms of Figure 16 are directly related to the nonlinear temperatures (node numbers 83-117 of the second set of nodal variables) in



the heat transfer problem. These isotherms, or constant nondimensional temperature lines, vary in value from +1.0 on the hot temperature wall, to -1.0 on the cold temperature wall. The equation used for deriving these values at all thirty-five nodal points within the solution domain was

$$\theta = \frac{T - T_M}{T_H - T_M} \quad (4.29)$$

where  $T$  is the nodal temperature and  $T_M$  is the mean temperature of the fluid defined at  $T_M = \frac{(T_H + T_C)}{2}$ .

The general shape and relative location of the various isotherms within the rectangular enclosure are somewhat similar to those of comparable heat transfer flows involving different Prandtl numbers, Grashof numbers and  $L/D$  ratios. However, due to the relatively low Grashof number of the present system (946.4), there is a total lack of a plateau in the center region of Figure 16 and the closely packed boundary layer flow near the walls is also missing. This boundary layer type flow is characteristic of much higher Grashof numbers such as those found in the comparative examples in [3], [7] and [15] where the  $Gr_L$  ranged from 5000 up to 18000.

Based solely on the thirty-five nodal point temperatures available from the solution, the isotherms were sketched as



shown in Figure 16. Lacking additional information, the shape of the contour lines between such nodes were linearly approximated, to a large extent, without speculating as to their exact curvature.

The actual height and width of the enclosure was normalized to  $y^*$  and  $x^*$ , respectively, for easier interpretation of the figure.





## V. EXPERIMENTAL PROCEDURE

### A. ARRANGEMENT OF TEST APPARATUS

The experimental apparatus was arranged so as to allow the study of an essentially two-dimensional fluid flow.

The major components of the apparatus consisted of the test platform, which housed the rectangular enclosure (Figure 6), a control system made up of two water circulators that maintained the vertical copper walls of the test platform at desired temperatures (Figure 7), and a large (250 mm DIA) plano-convex glass lens for reducing the object (8.5 x 1.875 inch vertical rectangular cavity) down to a smaller image size that could be completely captured on the 4x5 inch holographic plate (Figure 11).

The rectangular enclosure holding the fluid under investigation was 8.5 inches high, 7 inches long, and 1.875 inches wide. This test cavity was sandwiched between two plexiglas water reservoirs providing constant circulation by means of manifold connections on their tops and bottoms. Hot and cold water drains were located on top of the left and right reservoirs, respectively. Similarly, on the bottom were the hot and cold water inputs. The



inner walls of these two reservoirs were formed by quarter inch thick oxygen-free copper plates. Also, these same copper plates comprised the principle walls of the rectangular enclosure, with the "side walls" being made of plate glass, in order to allow visual observations. Six thermocouples were attached to each copper wall and then connected to a multichannel recorder for temperature monitoring purposes. The time needed for each copper plate to reach its respective equilibrium temperature once the water circulators had been turned on was approximately 39.8 seconds. For comparison purposes, it took the system just under one hour (58.1 minutes) for the 50-HB-3520 lubricating fluid to attain a steady equilibrium temperature under the same experimental conditions.

An important constraint imposed by interferometry is that the total distance traveled by the object beam must be nearly identical with the total path length of the reference beam, if the index of refraction throughout is uniform. Since the fluid in the rectangular enclosure possessed a refraction index of 1.461 and laser light along the object beam had to travel through 7 inches of this fluid, then the corrected path length through the test cavity was 10.23 inches, or a net increase of over 3 inches.



With this in mind, the equipment was arranged in a semi-elliptical pattern on a heavy table supported at six critical areas by inflated inner tubes. These were to act as stabilizing devices. Equipment could not be arranged on an exact ellipse due to the fact that a distance of four feet, five inches alone was needed from the object to the plano-convex lens out of a total table length of eight feet. Even so, the turning mirrors were located on the apexes of the "shortened" minor axis, the beam splitter at one end of the major axis, and the aqueous hologram holder at the opposite end (Figure 8). Spatial filters were employed to clean up each beam and expand it. A diverging lens was inserted just after the object beam spatial filter in order to expand this beam to proper size before it reached the rectangular test slit. Also, a collimating lens was placed between the spatial filter and hologram holder along the reference beam. Finally, a large diffusing screen made from a piece of developed film mounted on plexiglas and secured in a rigid metal frame, plus the test platform itself, were placed between the object mirror and the hologram holder (Figures 9 and 10).

Choosing the correct hologram holder is very important in live fringe holography. The reconstructed virtual image





must exactly match the original object. Unfortunately, after the processing of a hologram, the emulsion on its surface tends to dry, thus causing an unwanted displacement of the virtual image. One procedure that may be used to circumvent this problem is to choose a holder that maintains the hologram in aqueous surroundings, such as was utilized in this experiment. Also, in order to keep the hologram perfectly rigid during and after processing, the exposed glass plate was secured in a removable metal frame complete with handle. In this way, exact replacement in the hologram holder after processing posed no problem. Two micrometers built into the holder's top and left side were then used for fine adjustment of the hologram.

The test cavity or rectangular enclosure was filled with a very highly viscous fluid (actually a lubricant) produced by Union Carbide and known as "UCON" 50-HB-3520. This fluid was required to possess physical properties such that Rayleigh numbers, based on cavity width, of the order of  $10^4$ - $10^5$  could be obtained in the apparatus, at accurately measurable temperature differences. Since  $(T_H - T_C)$  was held constant throughout the experiment, only one Rayleigh number was calculated. Its value of  $1.0755 \times 10^4$  was well within the above tolerance zone. Worth mentioning



is the generally accepted prediction that above  $Ra \approx 2.0 \times 10^4$ , the phenomena known as "secondary flows" begin to occur. Obviously, such was not the case in this experiment.

A scribed grid pattern was attached to the back side of the test cavity to assist in alignment of the fringes. Two water circulators were connected by tubes to manifold nipples on the reservoir ends of the test platform. One circulator was set to deliver distilled water at 20°C (cold temp.) and the other at 25°C (hot temp). By using slide valves, the amount of water expended from the circulators could be regulated and controlled.

The experimental procedure was initiated only after the entire system had been carefully aligned. The rectangular enclosure was allowed to sit undisturbed for a period of at least several hours to ensure an equilibrium temperature state throughout the fluid. Then, a shutter was placed directly in front of the beam of a 3 milliwatt, helium-neon continuous wave laser serving as the coherent light source. After an Agfa-Gevaert Inc. 10E75 holographic recording plate was placed in the holder, the shutter was opened and the plate was exposed for one and one-half seconds (Figure 13). This plate was then removed, developed, and replaced in its exact position. Both circulators were



then turned on, and a continuous flow of water at 20°C and 25°C was allowed to cycle through the reservoirs on the test platform. Once fringe lines appear, their visibility can be strengthened by following the procedure outlined in the next subsection.

If one word could be used to describe the single most important factor determining the success or failure of this experiment, it would have to be rigidity. All relatively light-weight gear, such as; the laser, turning mirrors, spatial filters, and the plano-convex lens had to be weighted down to make them immovable. The test platform, in which the rectangular cavity was located, was sufficiently heavy on its own to preclude it from having to be additionally weighted down. The hologram holder already came with a very heavy base attached. All connecting devices, including the tubes transporting heated water to the plexiglas reservoirs and plastic sleeves housing the thermocouple leads were securely taped together to prevent vibration or motion. Any such random vibration would cause the fringe patterns to become lost.

The viewing of these fringes and the subsequent collecting of data can be accomplished by positioning the appropriate camera in a direct line with the rectangular





enclosure, plano-convex lens, and hologram holder (Figure 11). A television monitor (Figure 12) was employed for convenient viewing of the fringe patterns in an area adjacent to the experimental set-up.

## B. HOLOGRAPHIC INTERFEROMETRY APPLICATIONS

Holographic interferometry is an excellent technique for developing interference fringe patterns, which may in turn be evaluated to quantitatively provide an accurate temperature field throughout the domain of the system.

Heat convection in a rectangular cavity would be difficult to analyze empirically. However, by replacing the sensors that would ordinarily be used to record temperature changes and flow rates with holographic technology, one can analyze directly the variation of the density fields within the rectangular enclosure. This technique also eliminates the inherent change in temperature and flow pattern caused by the physical insertion of the sensors into the test fluid.

Real-time holographic interferometry allows a continuous flow of information to be recorded at the precise time any changes in the observed fluid occur. Single exposure holograms are utilized with real-time interferometry. A time





sequence can be derived for each different viewing position, with the use of a single developed hologram.

Such an exposure technique consists of recording phase and amplitude information from an object, in this case the rectangular fluid enclosure, onto a holographic plate. The recording is accomplished through the use of a reference and an object (scene) beam, originating from a single source (Figure 13). After processing, the hologram is accurately repositioned in its holder. Illuminating the plate with the original reference beam results in the primary (virtual) image being projected onto the same area as was the object (Figure 14). By focusing the object and virtual image beams onto a film or focal plane, and then adjusting the system so that the two interfering wavefronts (object wave and reconstructed wave) coincide, fringes can be produced.

The hologram now can be finely adjusted to orient the fringes in either a vertical or horizontal reference frame. Likewise at this time, the fringe patterns can be made to appear more visible by varying the beamsplitter to increase the intensity of the reference beam while decreasing that of the scene beam. If the original object is changed or altered in any fashion by the effect of temperature, motion, or pressure, an exact superposition will create a reinforcement



or cancellation of the intensities of the two waves with the result being the establishment of a fringe pattern. A dark fringe is produced whenever the difference between the object and reconstructed wavefront involves an odd factor of  $\pi/2$ . Bright fringes occur when this difference equals an integer value of  $2\pi$ .

By inserting a camera in-line with the scene (object) beam, but on the back side of the holographic plate, one can observe and record live fringe data. This technique provides a real time analysis of an unsteady system without the need for expensive and time consuming sensors and calibration.

After processing has been completed, problems arising from live fringe single exposure holography include; displacement of the virtual image due to drying emulsions on the holographic plate, and non-exact replacement of the hologram in its holder. If any relative motion whatsoever has transpired between pieces of the experimental equipment during or after replacement of the hologram, the fringe patterns may be destroyed.

It was this last problem that caused a particularly detrimental effect on the experimental results of this thesis. Somewhere within the system (apparatus arrangement)



there existed a source of motion or a piece of gear slightly off the horizontal reference plane that completely eliminated these fringe formations almost immediately after they evolved. The exact source(s) was never totally isolated, but the possible choices were reduced down to two, the water circulators and/or the support stand of the hologram holder.





## VI. CONCLUSIONS

The finite-element method was incorporated into steady state and time dependent computer programs for analyzing laminar convective heat transfer between parallel plates. Two sample cases were tested utilizing the general steady state program. In each case, values of derived field variables compared favorably to either an exact solution, in the case of the Couette flow problem (Figure 15); or to similar theoretical results, in the case of the heat transfer problem (Figure 16). The exact solution of the velocity profile for Couette flow was obtained by programming the analytical expression given by equation 5.5 in [11].

After successful steady state results were achieved, a second computer program was then designed to take into account the previously-neglected transient behavior of the system. A major portion of the steady state fluid mechanics problem was interfaced with a series of subroutines, wherein the time-dependent terms were to be calculated, to yield a total solution to the governing system of equations and associated boundary conditions. Time itself became a limiting factor in the completion of this second program.



A problem associated with the convergence of the field variables remains to be resolved.

In the experimental phase of this thesis, five attempts were made to produce live fringe formations through the use of holographic interferometry. In only one of these attempts was there observed a momentary interference pattern, corresponding to the temperature gradient, across the test section. This observation lasted approximately three (3) seconds after the water heaters/circulators were activated.

The main factor(s) influencing this inability to acquire such live fringes, on film, was the necessary exclusion of the hologram holder from the recording plane because of limited table length and/or the vibrations generated by the water circulators used in the experiment. Either of these detrimental conditions could have eliminated completely the formation of interference fringe patterns.

Due to considerable time delay in the acquisition of some of the experimental apparatus, no further documentation of real time holographic interferometry study could be made beyond the previously-mentioned five attempts.



APPENDIX A

FIGURES

Figure 1  
Rectangular Enclosure

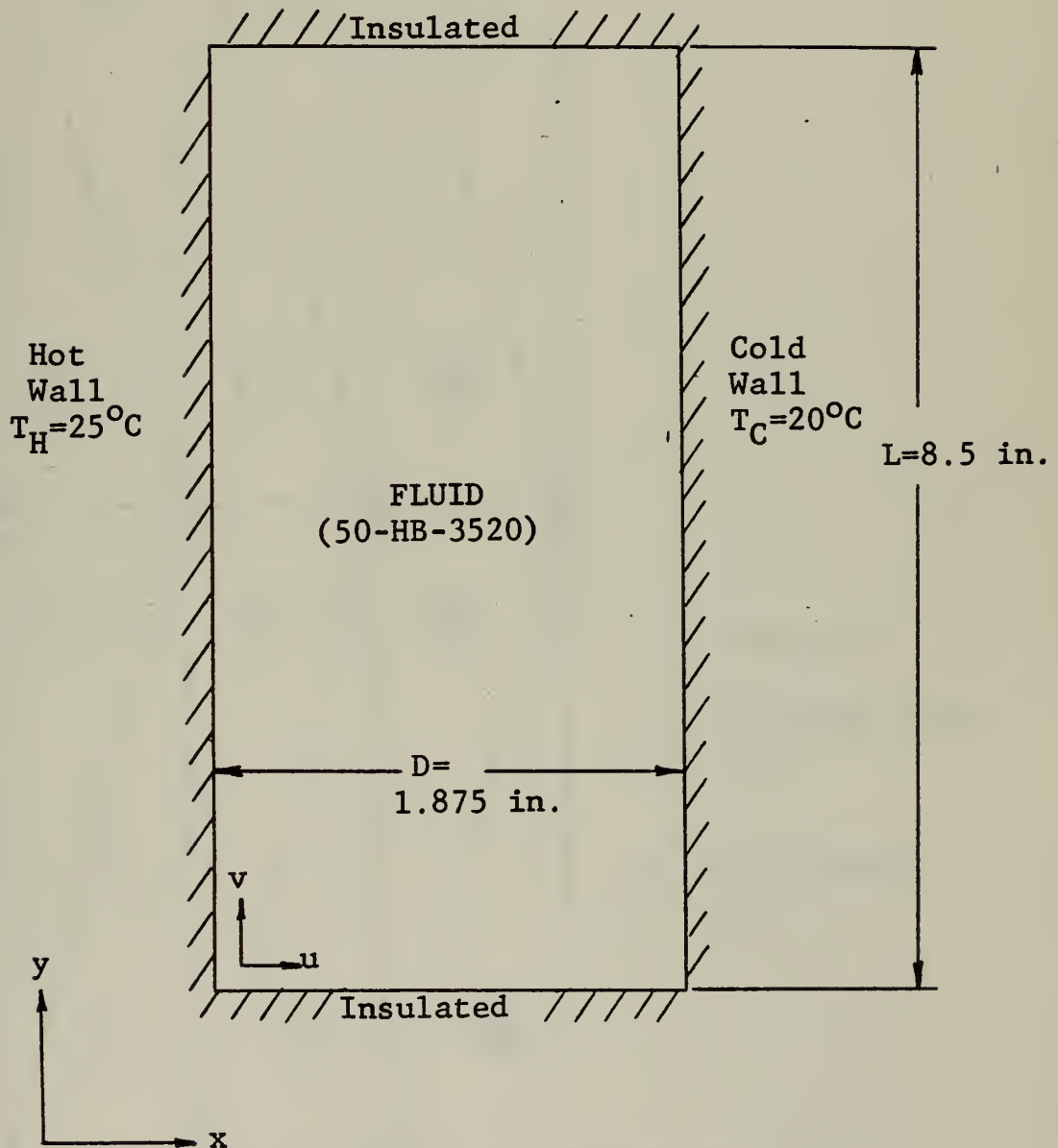




Figure 2  
Discretization of Solution Domain

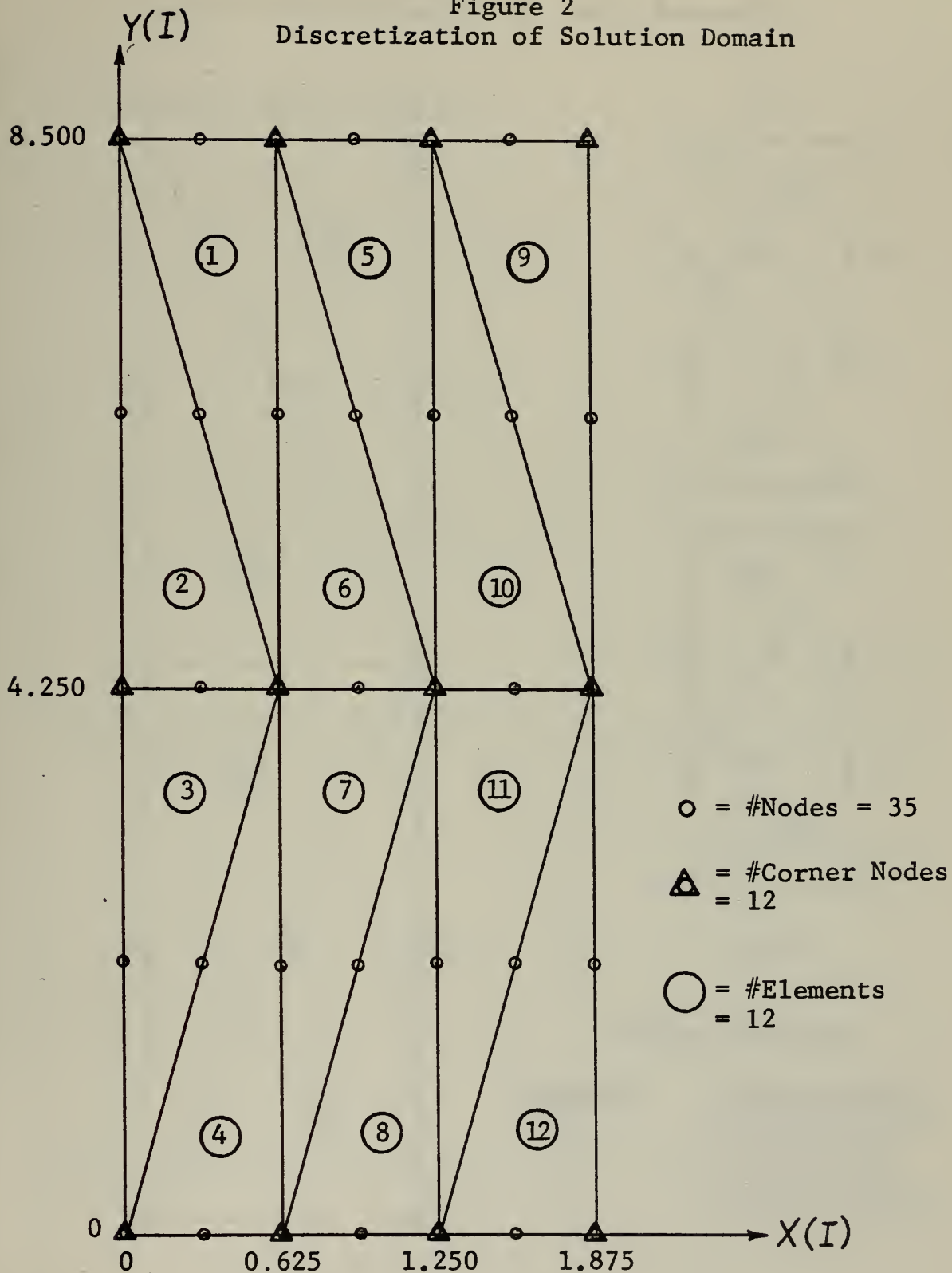
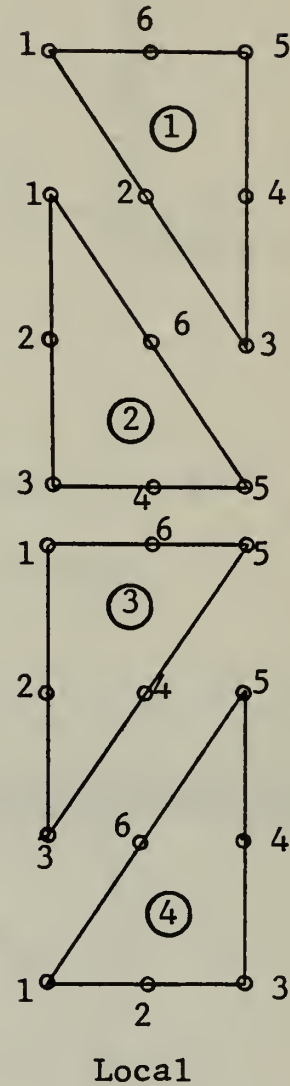
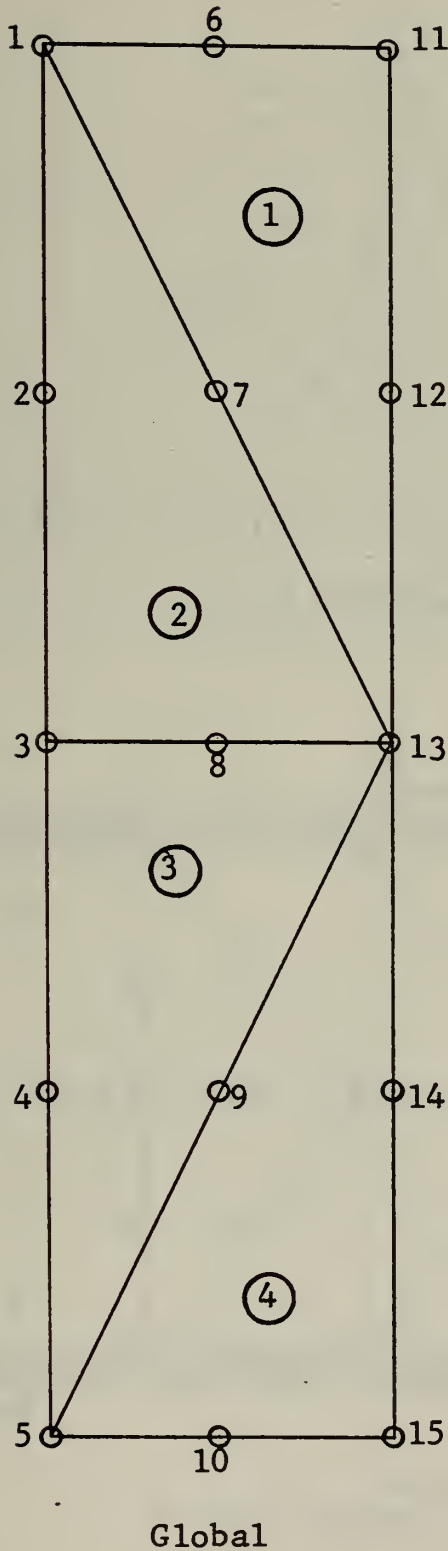






Figure 3  
Global Node-Numbering vs. Local Numbering



System Topology

<u>Element#</u>	<u>Global Node#'s</u>
1	1, 7, 13, 12, 11, 6
2	1, 2, 3, 8, 13, 7
3	3, 4, 5, 9, 13, 8
4	5, 10, 15, 14, 13, 9



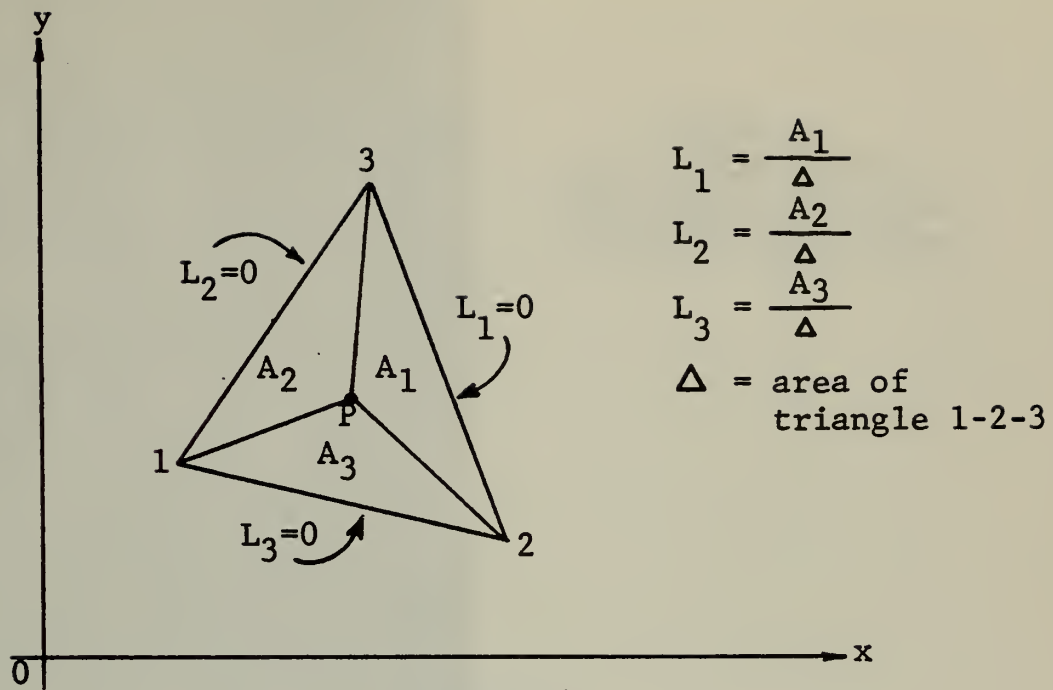


Figure 4  
Area Coordinates for a Triangle

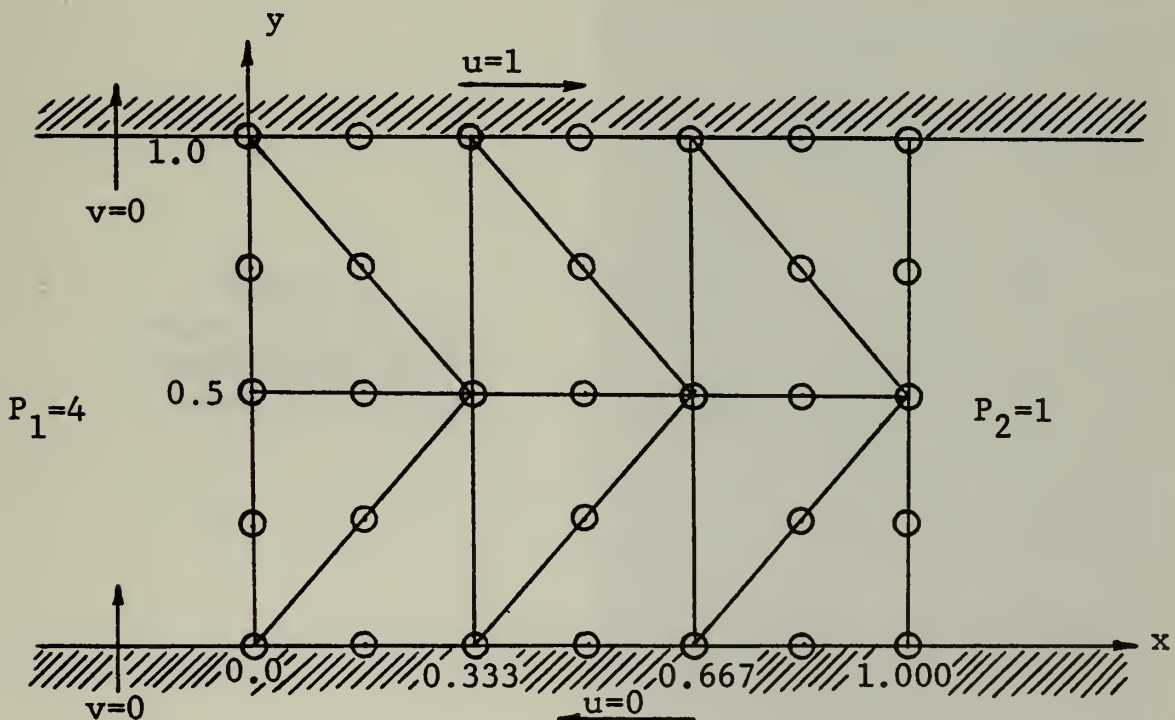


Figure 5  
F.E.M. Analysis of Couette Flow





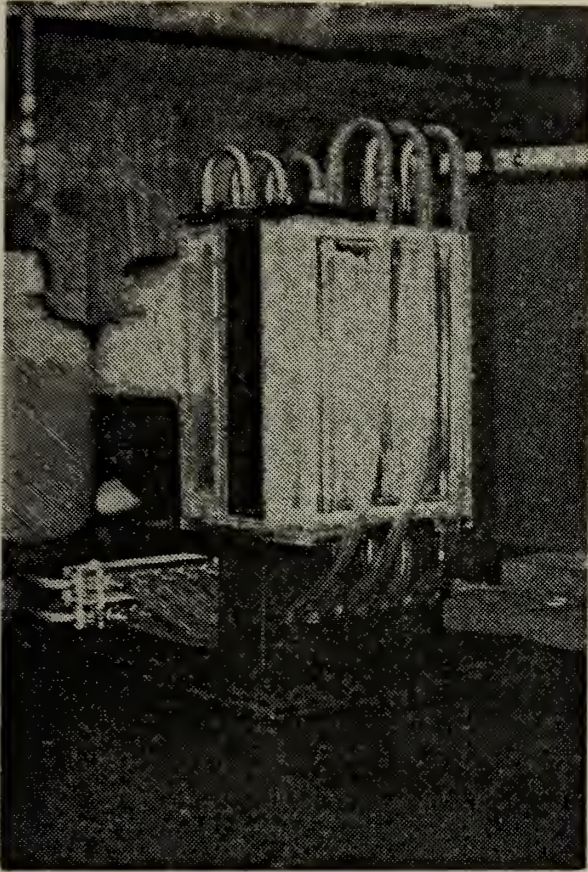


Figure 6  
Test Platform with  
Rectangular Enclosure  
and Water Reservoirs

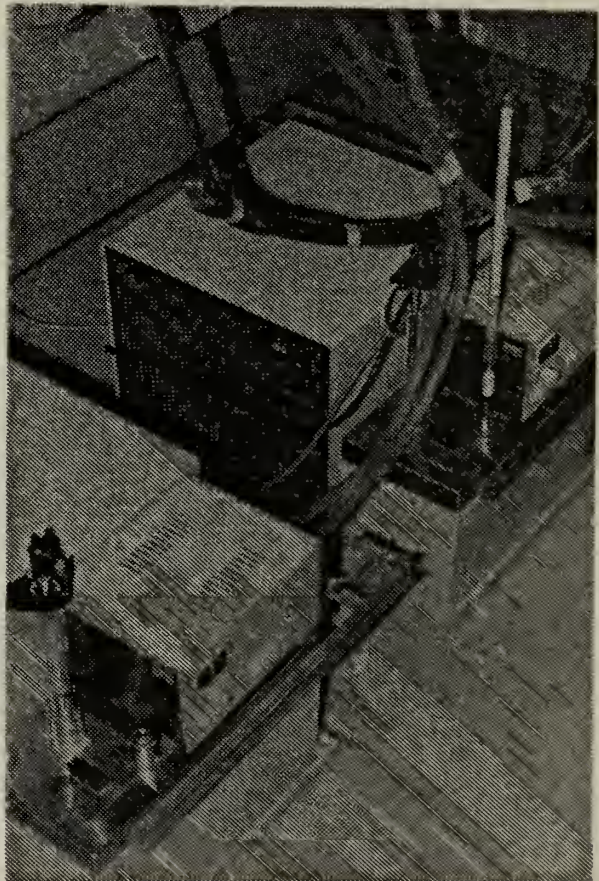
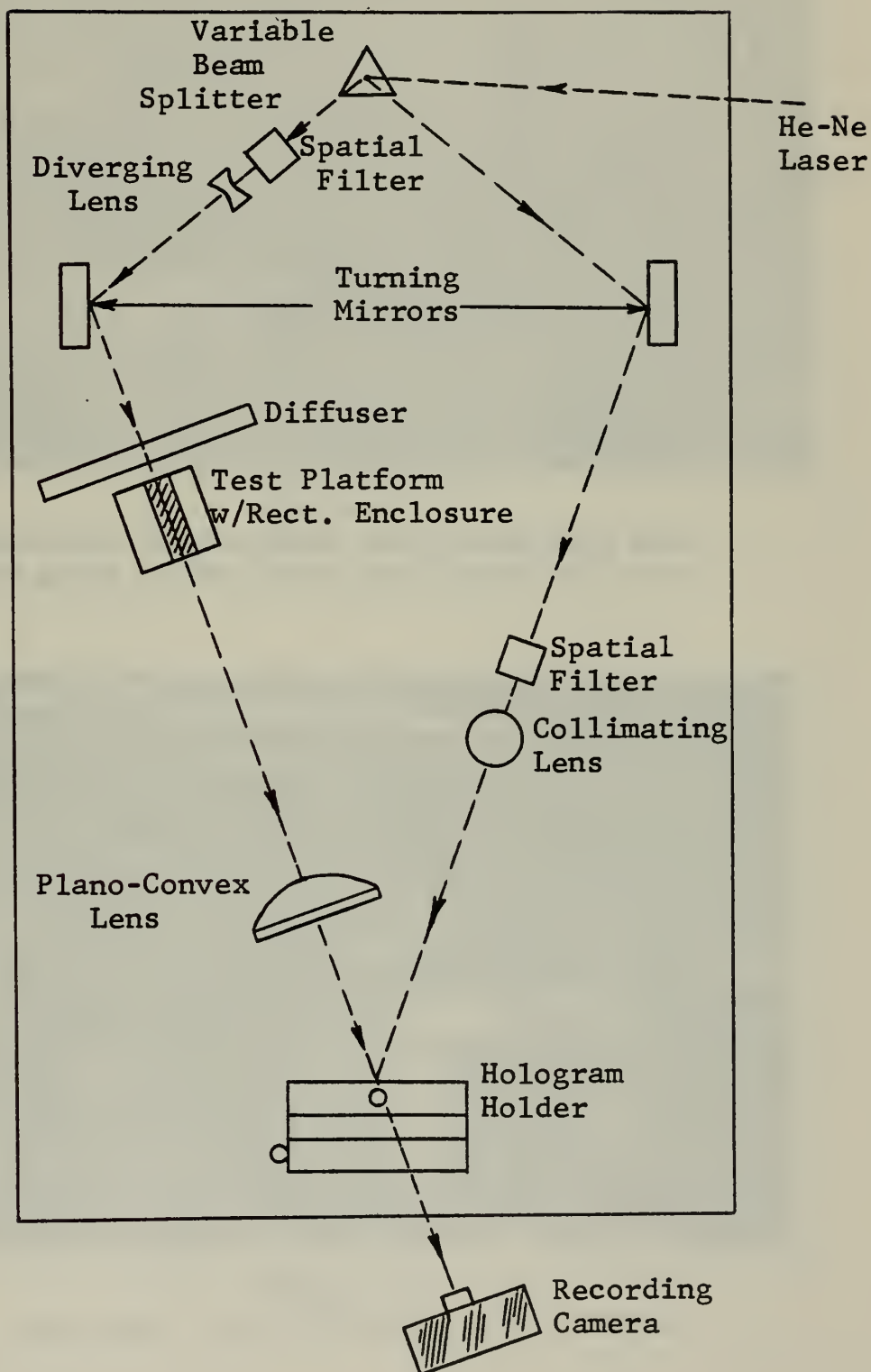


Figure 7  
Water Heaters and  
Circulators with  
Connecting Tubes





Figure 8  
Table Arrangement (top view)







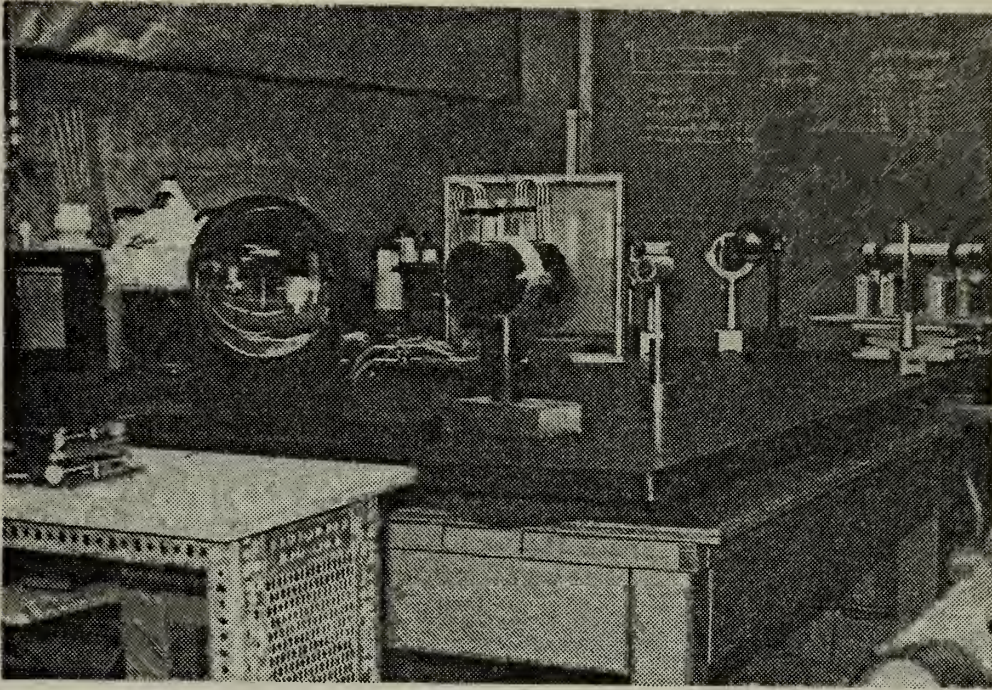


Figure 9  
Apparatus Arrangement with Reference Beam  
Oriented on the Right-hand-side of Table

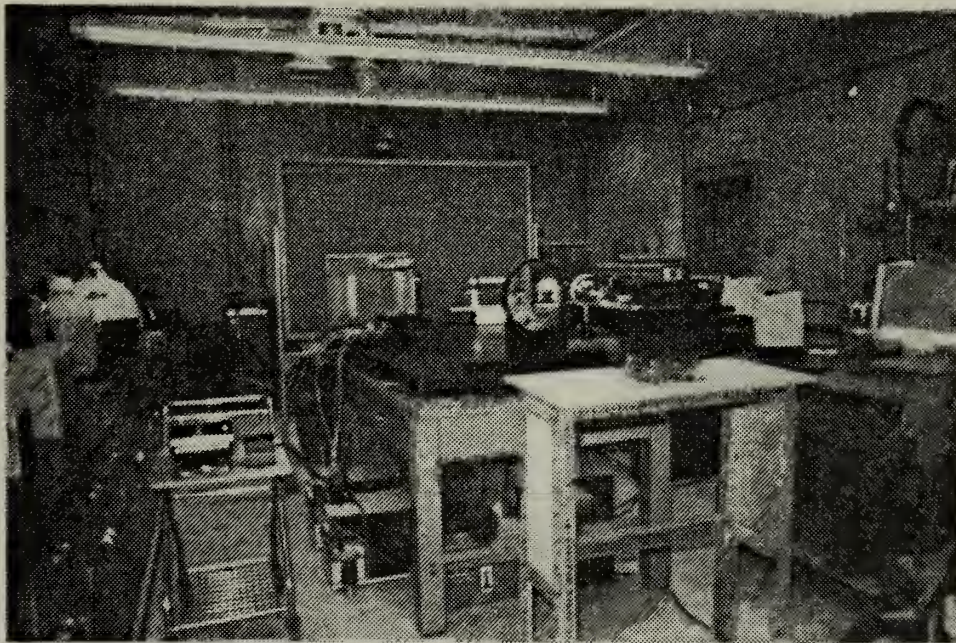


Figure 10  
Panoramic View of Experimental Layout





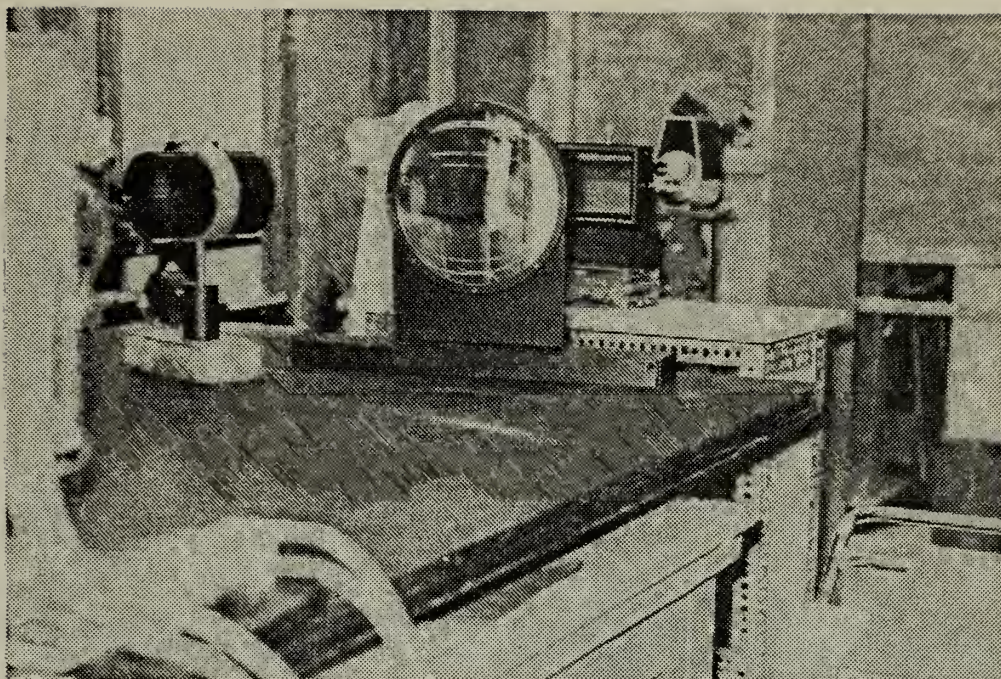


Figure 11  
Direct Line-up on Object Beam from  
Test Platform to Recording Camera

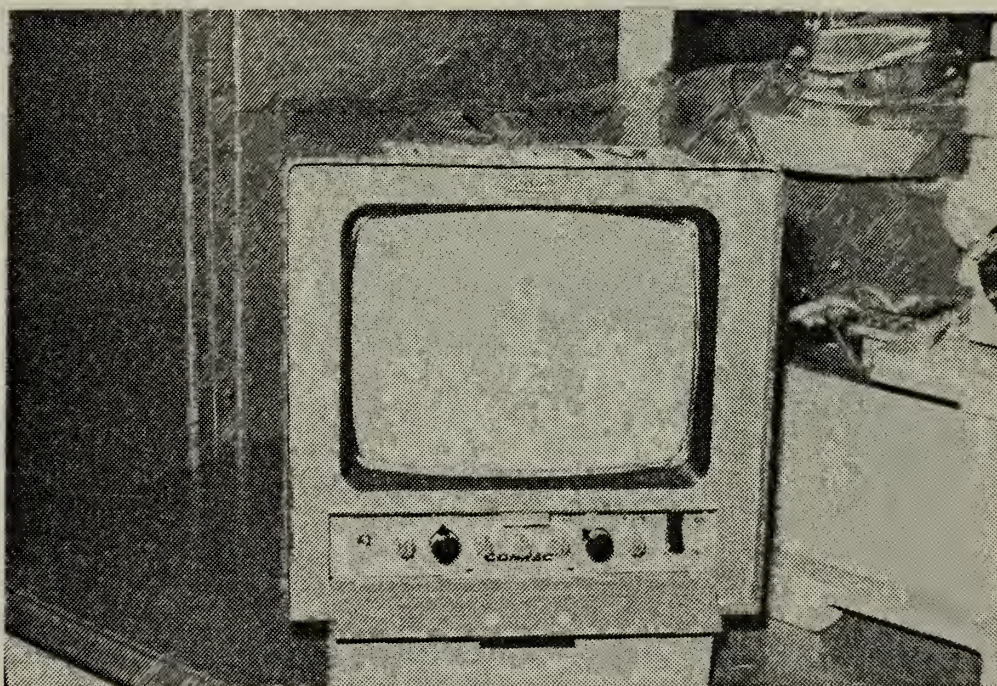


Figure 12  
Television Monitor Used for Convenient Viewing of Fringes





Figure 13  
Holographic Recording (top view)

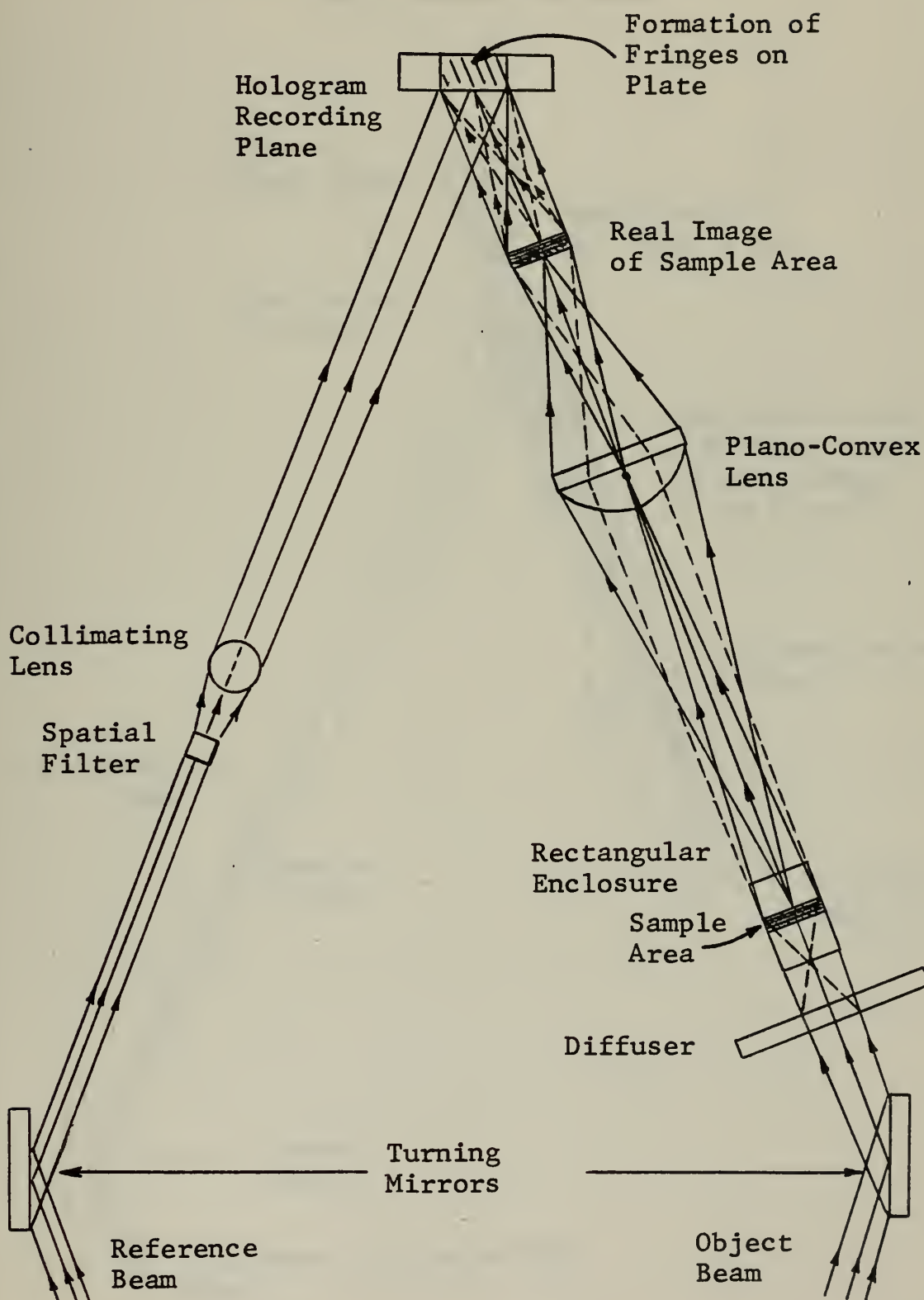
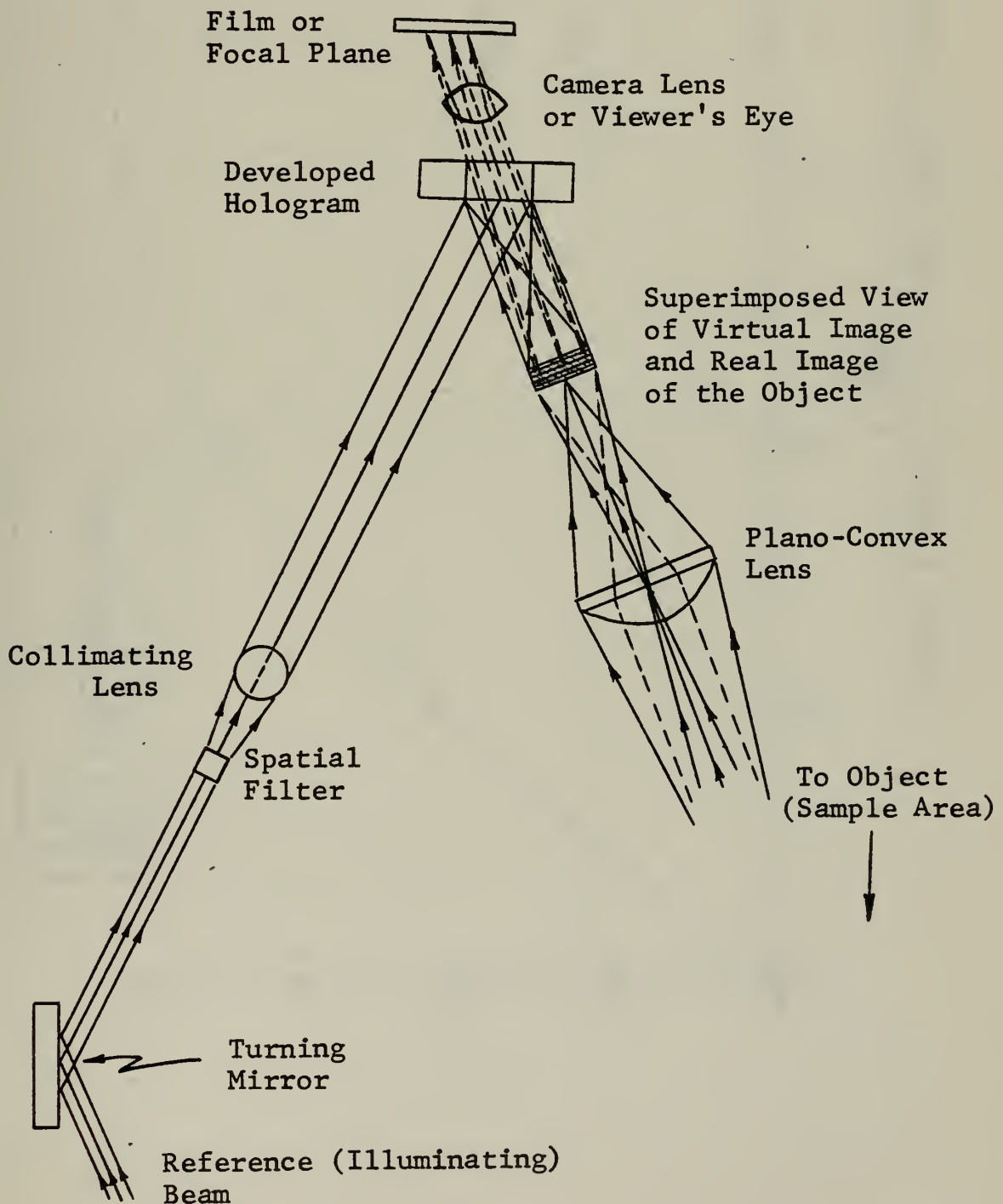






Figure 14  
Reconstruction and Recording of  
Interference Patterns





U VS. Y  
FEM & EXACT SOLUTIONS  
 $DP/DX = -3.0$

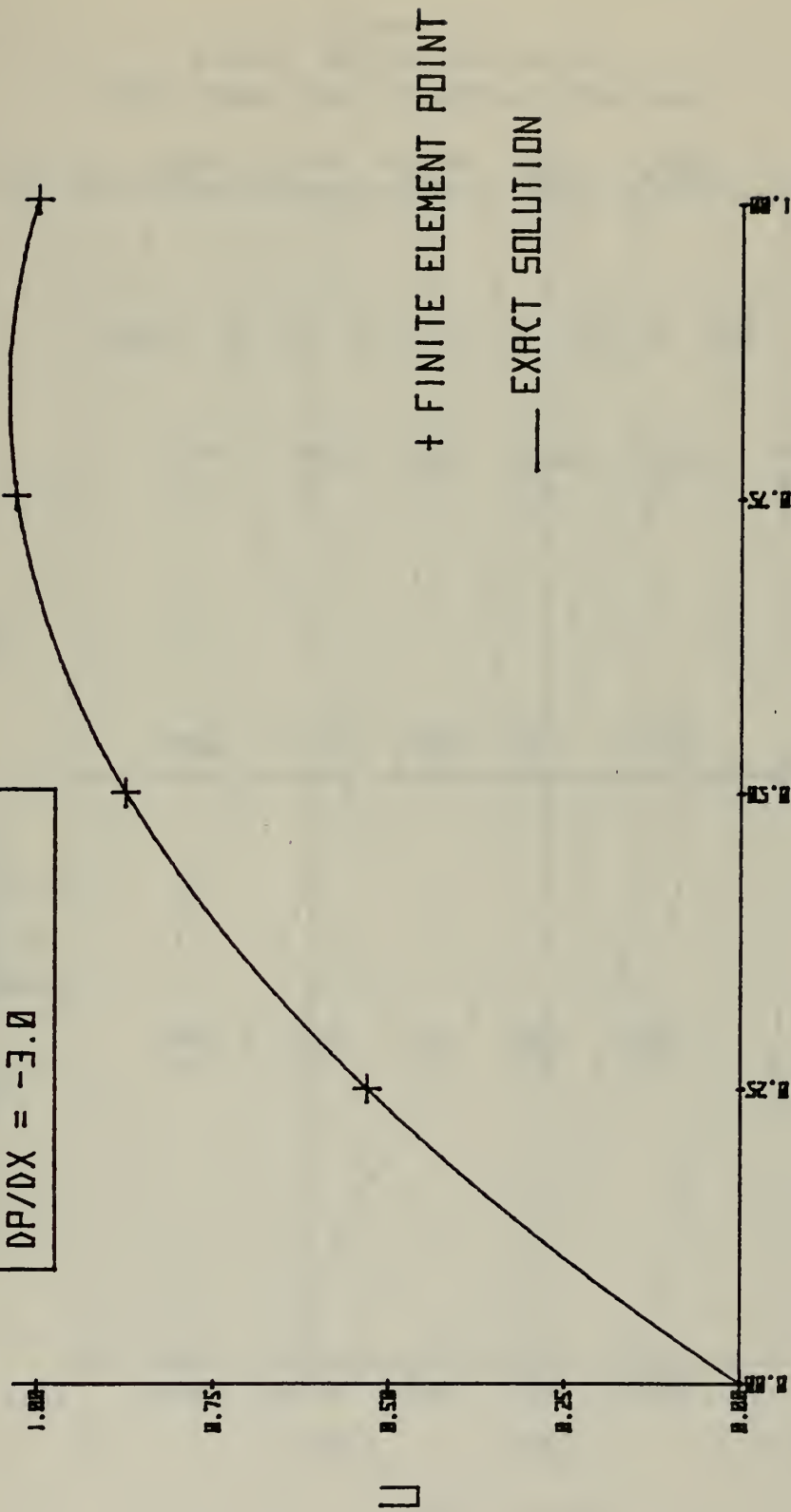
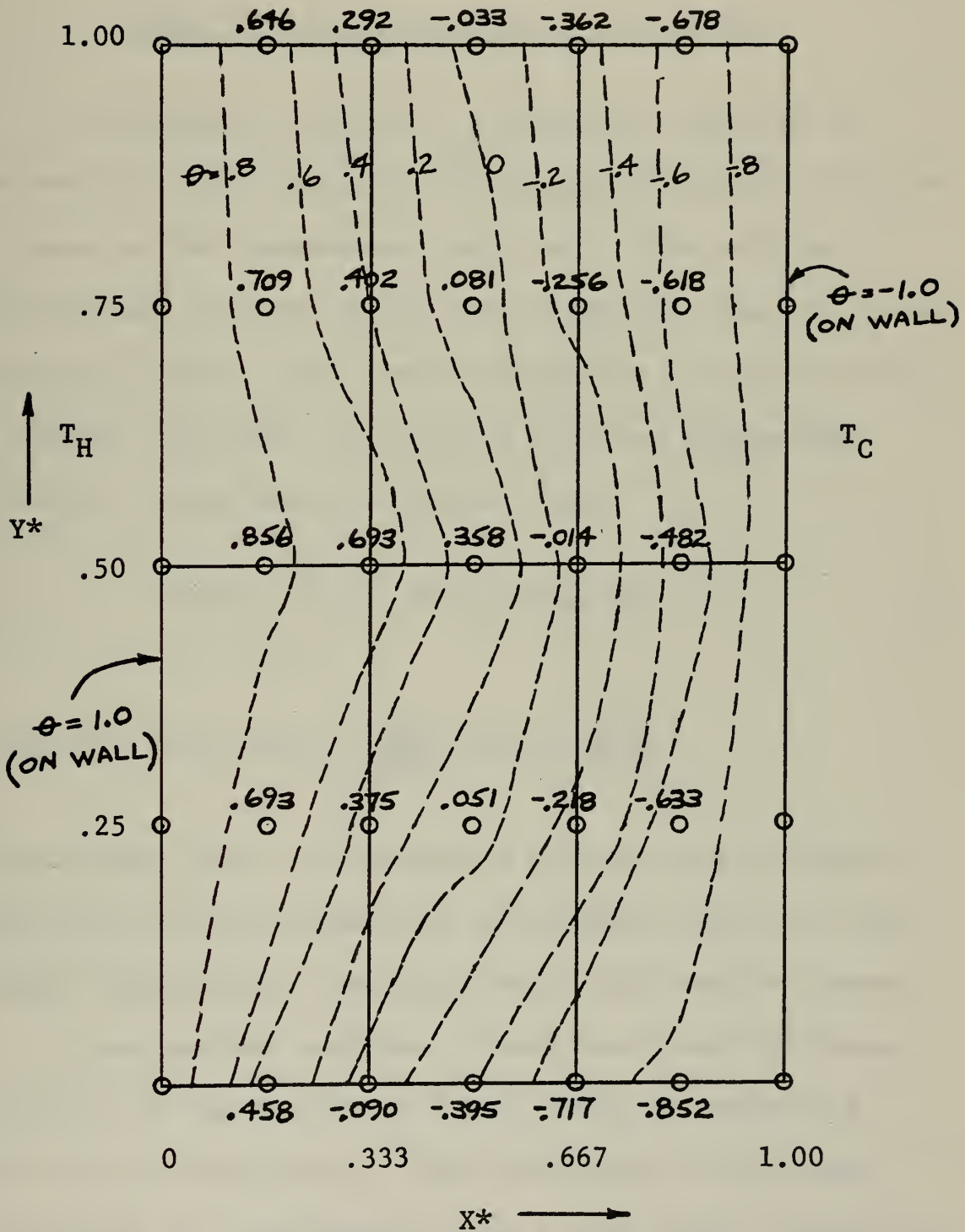


FIGURE 15  
VELOCITY PROFILE FOR LINEAR COUETTE FLOW



Figure 16  
Steady State Isotherms  
(Nonlinear Heat Transfer Problem)



$Gr_L = 9.464 \times 10^2,$

$Pr = 1.0755 \times 10^4,$

$L/D = 4.53$





## APPENDIX B

### BRIEF REVIEW ON CALCULUS OF VARIATIONS

A fundamental problem in differential calculus is extremizing (maximizing or minimizing) a function  $f(x)$  for a range of the independent variable  $x$ . The problem in variational calculus is also extremization; however, it is concerned chiefly with the extremization of a functional. A simple functional, in terms of only one independent variable, would have the typical form

$$I(\phi) = \int_{x_1}^{x_2} F(x, \phi, \phi_x, \phi_{xx}) dx$$

where  $\phi = \phi(x)$  and  $\phi_x = \frac{\partial \phi}{\partial x}$ ,  $\phi_{xx} = \frac{\partial^2 \phi}{\partial x^2}$ .

Summarizing, the two branches of calculus are related in that both are concerned with an extremum; one deals with number spaces while the other deals with function spaces.

In variational problems a functional which is characteristic of the problem is first formed in terms of a function (or functions). Then variations of this same functional are investigated with a view toward extremizing the functional. In some cases this approach results in a



closed form, exact solution. But more often, the problem must be solved by an approximate method. One such method is the Rayleigh-Ritz technique. This approach is preferable to the direct application of finite difference methodology to solve the differential equation with its associated boundary conditions, because the functional can often be used to assure convergence of the approximate solution.

A simple example of variational calculus is the problem of finding the plane curve joining two points  $(x_1, y_1)$  and  $(x_2, y_2)$  which has the shortest length. The solution sought here is the function  $y(x)$  describing the curve of shortest length; the corresponding functional is the length of the curve given by

$$I(y) = \int_{x_1}^{x_2} \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx$$

Using the method of variation of calculus implies that of all the curves

$$Y(x) = y(x) + \epsilon \delta(x)$$

which pass through the given end points, the shortest one  $y(x)$  must be selected. The problem thus reduces to finding the function  $y(x)$  that makes the integral  $I(y)$  a minimum.



Generally, in order to minimize the integral

$$I(y) = \int_{x_1}^{x_2} F(x, y, y') dx$$

where  $y' = \frac{dy}{dx}$ , the function  $y(x)$  must satisfy the boundary conditions and the Euler-Lagrange differential equation

$$\frac{\partial F}{\partial y} - \frac{d}{dx} \left( \frac{\partial F}{\partial y'} \right) = 0$$

The previous result could be extended to several dependent and independent variables. For example, in order to minimize the integral

$$I(\phi) = \iint_A F(x, y, \phi, \phi_x, \phi_y) dx dy$$

in which  $\phi_x$  and  $\phi_y$  are the partial derivatives of  $\phi$  with respect to  $x$  and  $y$ , respectively, the general function  $\phi$  must satisfy the Euler-Lagrange differential equation

$$\frac{\partial F}{\partial \phi} - \frac{\partial}{\partial x} \left( \frac{\partial F}{\partial \phi_x} \right) - \frac{\partial}{\partial y} \left( \frac{\partial F}{\partial \phi_y} \right) = 0$$

in addition to the specified boundary conditions.

In the past two decades, since the advent of high speed digital computers, the variational formulation has been quite extensively employed in the fields of structural



and continuum mechanics. Important variational principles such as least work, minimum strain energy, minimum potential energy, and Reissner's variational theorem of elasticity have been well developed and are documented in standard textbooks. However, similar variational principles applicable to fluid mechanic problems have not been as comprehensively developed. Calculus of variations has, until only recently, been utilized sparingly in the field of fluid mechanics.





# THIS IS A 2-D NONLINEAR COUETTE FLOW PROBLEM

IBAND= 26

NEQ = 82

NO. OF NODES= 35

NO. OF ELEMENTS= 12

NO. OF CORNER NODES= 12

NNVELS= 14

NNCXY= 21

NNPS= 12

## SUMMARY OF NODAL COORDINATES

I	X(I)	Y(I)
1	0.0	1.000
3	0.0	0.500
5	0.0	0.0
11	0.3333	1.000
13	0.3333	0.500
15	0.3333	0.0
21	0.667	1.000
23	0.667	0.500
25	0.667	0.0
31	1.000	1.000
33	1.000	0.500
35	1.000	0.0

## LISTING OF SYSTEM TOPOLOGY

ELEMENT NUMBER

NODE NUMBERS

1	1	7	13	12	11	6
2	1	2	3	8	13	7
3	3	4	5	9	13	8
4		10	15	14	13	9
5	11	17	23	22	21	16
6	11	12	13	18	23	17
7	13	14	15	19	23	18
8	15	20	25	24	23	19
9	21	27	33	32	31	26
10	21	22	23	28	33	27
11	23	24	25	29	33	28
12	25	30	35	34	33	29

## NODES WHERE VELOCITIES ARE SPECIFIED

I	NODE	U VELOCITY	V VELOCITY
1	1	1.000	0.0
2	5	0.0	0.0
3	6	1.000	0.0
4	10	0.0	0.0
5	11	1.000	0.0
6	15	0.0	0.0
7	16	1.000	0.0
8	20	0.0	0.0
9	21	1.000	0.0
10	25	0.0	0.0
11	26	1.000	0.0
12	30	0.0	0.0
13	31	1.000	0.0
14	35	0.0	0.0



NODES WHERE QX AND QY ARE SPECIFIED

I	NODE	QX	QY
1	7	0.0	0.0
2	8	0.0	0.0
3	9	0.0	0.0
4	12	0.0	0.0
5	13	0.0	0.0
6	14	0.0	0.0
7	17	0.0	0.0
8	18	0.0	0.0
9	19	0.0	0.0
10	22	0.0	0.0
11	23	0.0	0.0
12	24	0.0	0.0
13	27	0.0	0.0
14	28	0.0	0.0
15	29	0.0	0.0
16	22	1.333	0.0
17	3	0.667	0.0
18	4	1.333	0.0
19	32	-0.333	0.0
20	33	-0.167	0.0
21	34	-0.333	0.0

NODES WHERE PRESSURE IS SPECIFIED

I	NODE	PRESSURE
1	1	4.000
2	3	4.000
3	5	4.000
4	11	3.000
5	13	3.000
6	15	3.000
7	21	2.000
8	23	2.000
9	25	2.000
10	31	1.000
11	33	1.000
12	35	1.000

NODAL VARIABLE IS THE U-VELOCITY AT NODES 1 - 35;  
 THE V-VELOCITY AT NODES 36 - 70;  
 AND THE PRESSURES AT NODES 71 - 82.

THE FIRST SEQUENCE OF 82 NODAL VARIABLES  
 REPRESENTS A LINEAR, STEADY STATE SYSTEM;  
 WHILE THE SECOND SET OF THE 82 VALUES CORRESPONDS  
 TO A NONLINEAR ANALYSIS OF THE SOLUTION DOMAIN.

A PRESSURE GRADIENT IN THE HORIZONTAL SHEAR  
 DIRECTION OF -3 UNITS IN MAGNITUDE HAS BEEN ADDED  
 TO PRODUCE A CURVED VELOCITY PROFILE.



NCDE NC.	NODE VARIABLES
1	0.1000000000 01
2	0.1031249996 01
3	0.8750000067 00
4	0.5312499961 00
5	0.0
6	0.1000000000 01
7	0.1031249999 01
8	0.8750000022 00
9	0.5312499991 00
10	0.0
11	0.1000000000 01
12	0.1031249999 01
13	0.8749999980 00
14	0.5312499995 00
15	0.0
16	0.1000000000 01
17	0.1031250000 01
18	0.8749999980 00
19	0.5312499995 00
20	0.0
21	0.1000000000 01
22	0.1031250002 01
23	0.8750000014 00
24	0.5312500020 00
25	0.0
26	0.1000000000 01
27	0.1031250006 01
28	0.8749999986 00
29	0.5312500056 00
30	0.0
31	0.1000000000 01
32	0.1031250014 01
33	0.8749999887 00
34	0.5312500139 00
35	0.0
36	0.0
37	0.60994269640-19
38	-0.71251878180-42
39	-0.60994269640-19
40	0.0
41	0.0
42	0.89102833380-19





43	-0.81332488890-42
44	-0.88102833380-19
45	0.0
46	0.0
47	0.19352502520-18
48	-0.11149020410-41
49	-0.19352502520-18
50	0.0
51	0.0
52	0.47096946920-18
53	-0.15718322820-41
54	-0.47096946920-18
55	0.0
56	0.0
57	0.11670534580-17
58	-0.21868403170-41
59	-0.11670534580-17
60	0.0
61	0.0
62	0.29005182730-17
63	-0.26715796970-41
64	-0.29005182730-17
65	0.0
66	0.0
67	0.20080511240-17
68	-0.33565885460-41
69	-0.20080511240-17
70	0.0
71	0.40000000000 01
72	0.40000000000 01
73	0.40000000000 01
74	0.30000000000 01
75	0.30000000000 01
76	0.30000000000 01
77	0.20000000000 01
78	0.20000000000 01
79	0.20000000000 01
80	0.10000000000 01
81	0.10000000000 01
82	0.10000000000 01

NODE NO.

NODE VARIABLES

1

0.10000000000 01



2	0.10288814530 01
3	0.8689584459D 00
4	0.5288644394D 00
5	0.0
6	0.1000000000D 01
7	0.1029118812D 01
8	0.8708250523D 00
9	0.5291035809D 00
10	0.0
11	0.1000000000D 01
12	0.1029300447D 01
13	0.8707893553D 00
14	0.5292824849D 00
15	0.0
16	0.1000000000D 01
17	0.1029562570D 01
18	0.8718195572D 00
19	0.5295482546D 00
20	0.0
21	0.1000000000D 01
22	0.1029726484D 01
23	0.8714843669D 00
24	0.5297056677D 00
25	0.0
26	0.1000000000D 01
27	0.1030036890D 01
28	0.8728957661D 00
29	0.5300233390D 00
30	0.0
31	0.1000000000D 01
32	0.1030173492D 01
33	0.8734199036D 00
34	0.5301609830D 00
35	0.0
36	0.0
37	0.9409332543D-19
38	0.2406344091D-19
39	-0.5690421895D-19
40	0.0
41	0.0
42	0.1282493313D-18
43	0.2611072288D-19
44	-0.8653878326D-19
45	0.0



46	0.0
47	0.25383549350-18
48	0.32006190050-19
49	-0.19986923340-18
50	0.0
51	0.0
52	0.56134699840-18
53	0.36351135030-19
54	-0.49119243710-18
55	0.0
56	0.0
57	0.12819175680-17
58	0.37806257670-19
59	-0.12007464690-17
60	0.0
61	0.0
62	0.29622390920-17
63	0.32117633760-19
64	-0.29108457190-17
65	0.0
66	0.0
67	0.20082897640-17
68	0.15102713170-19
69	-0.19908894340-17
70	0.0
71	0.40000000000 01
72	0.40000000000 01
73	0.40000000000 01
74	0.30000000000 01
75	0.30000000000 01
76	0.30000000000 01
77	0.20000000000 01
78	0.20000000000 01
79	0.20000000000 01
80	0.10000000000 01
81	0.10000000000 01
82	0.10000000000 01



STEADY STATE FLUID MECHANICS PROBLEM  
(HEAT TRANSFER)  
THIS IS A 2-D NONLINEAR PROBLEM

IBAND= 26

NEC=117

NO. OF NODES= 35

NO. OF ELEMENTS= 12

NC. OF CORNER NODES= 12

NNVELS= 20

NNCXY= 15

NNPS= 6

NNTS= 10

NNCZC= 6

NNCZ= 25

SUMMARY OF NODAL COORDINATES

I	X(I)	Y(I)
1	0.0	8.500
3	0.0	4.250
5	0.0	0.0
11	0.625	8.500
13	0.625	4.250
15	0.625	0.0
21	1.250	8.500
23	1.250	4.250
25	1.250	0.0
31	1.875	8.500
33	1.875	4.250
35	1.875	0.0

LISTING OF SYSTEM TOPOLOGY

ELEMENT NUMBER

NODE NUMBERS

1	1	7	13	12	11	6
2	1	2	3	8	11	7
3	3	4	5	9	13	8
4	5	10	15	14	13	9
5	11	17	23	22	21	16
6	11	12	13	18	23	17
7	13	14	15	19	23	18
8	15	20	25	24	23	19
9	15	20	25	24	23	19
10	21	27	33	32	31	26
11	21	22	23	28	33	27
12	23	24	25	29	33	28
	25	30	35	34	33	29

NODES WHERE VELOCITIES ARE SPECIFIED

I	NODE	U VELOCITY	V VELOCITY
---	------	------------	------------





1	1	0.0	0.0
2	2	0.0	0.0
3	3	0.0	0.0
4	4	0.0	0.0
5	5	0.0	0.0
6	6	0.0	0.0
7	7	0.0	0.0
8	8	0.0	0.0
9	9	0.0	0.0
10	10	0.0	0.0
11	11	0.0	0.0
12	12	0.0	0.0
13	13	0.0	0.0
14	14	0.0	0.0
15	15	0.0	0.0
16	16	0.0	0.0
17	17	0.0	0.0
18	18	0.0	0.0
19	19	0.0	0.0
20	20	0.0	0.0

# NODES WHERE QX AND QY ARE SPECIFIED

I	NODE	QX	QY
1	7	0.0	0.0
2	8	0.0	0.0
3	9	0.0	0.0
4	10	0.0	0.0
5	11	0.0	0.0
6	12	0.0	0.0
7	13	0.0	0.0
8	14	0.0	0.0
9	15	0.0	0.0
10	16	0.0	0.0
11	17	0.0	0.0
12	18	0.0	0.0
13	19	0.0	0.0
14	20	0.0	0.0
15	21	0.0	0.0

# NODES WHERE PRESSURE IS SPECIFIED

I	NODE	PRESSURE
1	1	1014000.000
2	2	1014000.000
3	3	1014000.000
4	4	1014000.000
5	5	1014000.000
6	6	1014000.000

# NODES WHERE TEMPERATURE IS SPECIFIED

I	NODE	TEMPERATURE
1	1	25.000
2	2	25.000
3	3	25.000
4	4	25.000
5	5	25.000
6	6	20.000
7	7	20.000
8	8	20.000
9	9	20.000
10	10	20.000

# NODES WHERE QZC IS SPECIFIED

I	NODE	QZC
1	4	0.0
2	5	0.0
3	6	0.0
4	7	0.0
5	8	0.0
6	9	0.0

# NODES WHERE HEAT FLUX QZ IS SPECIFIED



I	NODE	HEAT FLUX
1	6	0.0
2	7	0.00
3	8	0.00
4	9	0.00
5	10	0.00
6	11	0.00
7	12	0.00
8	13	0.00
9	14	0.00
10	15	0.00
11	16	0.00
12	17	0.00
13	18	0.00
14	19	0.00
15	20	0.00
16	21	0.00
17	22	0.00
18	23	0.00
19	24	0.00
20	25	0.00
21	26	0.00
22	27	0.00
23	28	0.00
24	29	0.00
25	30	0.0

NODAL VARIABLE IS THE U-VELOCITY AT NODES 1 - 35;  
 THE V-VELOCITY AT NODES 36 - 70;  
 THE PRESSURE AT NODES 71 - 82;  
 AND THE TEMPERATURE AT NODES 83 - 117.

THE SPECIFIED WALL PRESSURES  
 ARE NORMALIZED TO ONE (1) ATMOSPHERE,  
 THAT IS, 1014000 DYNES/SQ.CM  
 (ALL PARAMETER VALUES ARE IN CGS UNITS).

THE FIRST SEQUENCE OF 117 NODAL VARIABLES  
 REPRESENTS A LINEAR, STEADY STATE SYSTEM;  
 WHILE THE SECOND SET OF THE 117 VALUES CORRESPONDS  
 TO A NONLINEAR ANALYSIS OF THE SOLUTION DOMAIN.

NODE NO.	NODE VARIABLE
1	0.0
2	0.0
3	0.0
4	0.0



5	0.0
6	0.0
7	-0.1594580172D 01
8	-0.7882672787D 01
9	-0.1594580172D 01
10	0.0
11	0.0
12	-0.3649507488D 00
13	-0.1203896378D 02
14	-0.3649507488D 00
15	0.0
16	0.0
17	-0.1216689153D 00
18	-0.9223308785D 01
19	-0.1216689153D 00
20	0.0
21	0.0
22	-0.1605186515D 01
23	-0.1070997583D 02
24	-0.1605186515D 01
25	0.0
26	0.0
27	-0.1686613562D 01
28	-0.6458410240D 01
29	-0.1686613562D 01
30	0.0
31	0.0
32	0.0
33	0.0
34	0.0
35	0.0
36	0.0
37	0.0
38	0.0
39	0.0
40	0.0
41	0.0
42	0.3287813820D 00
43	-0.8671505203D-23
44	-0.3287813820D 00
45	0.0
46	0.0
47	0.4841816743D 00
48	-0.1212888826D-22





49	-0.48418181430 00
50	0.0
51	0.0
52	0.28532060500 00
53	0.42109196680-23
54	-0.28532060500 00
55	0.0
56	0.0
57	0.15892437070-01
58	0.92415574840-23
59	-0.15892437070-01
60	0.0
61	0.0
62	-0.43460776970-01
63	0.11208359430-22
64	0.43460776970-01
65	0.0
66	0.0
67	0.0
68	0.0
69	0.0
70	0.0
71	0.10140000000 07
72	0.10140000000 07
73	0.10140000000 07
74	0.10140370640 07
75	0.10142256570 07
76	0.10140370640 07
77	0.10139803130 07
78	0.10138750670 07
79	0.10139803130 07
80	0.10140000000 07
81	0.10140000000 07
82	0.10140000000 07
83	0.25000000000 02
84	0.25000000000 02
85	0.25000000000 02
86	0.25000000000 02
87	0.25000000000 02
88	0.24166666670 02
89	0.24166666670 02
90	0.24166666670 02
91	0.24166666670 02
92	0.24166666670 02



93	0.233333333D 02
94	0.233333333D 02
95	0.233333333D 02
96	0.233333333D 02
97	0.233333333D 02
98	0.225000000D 02
99	0.225000000D 02
100	0.225000000D 02
101	0.225000000D 02
102	0.225000000D 02
103	0.216666666D 02
104	0.216666666D 02
105	0.216666666D 02
106	0.216666666D 02
107	0.216666666D 02
108	0.208333333D 02
109	0.208333333D 02
110	0.208333333D 02
111	0.208333333D 02
112	0.208333333D 02
113	0.200000000D 02
114	0.200000000D 02
115	0.200000000D 02
116	0.200000000D 02
117	0.200000000D 02

NODE NO.	NODE VARIABLE
1	0.0
2	0.0
3	0.0
4	0.0
5	0.0
6	0.0
7	-0.1671967327D 01
8	-0.8146388287D 01
9	-0.1667709024D 01
10	0.0
11	0.0
12	-0.3529066201D 00
13	-0.1267086943D 02
14	-0.3464796968D 00
15	0.0
16	0.0



17	-0.95541541550-01
18	-0.96623861040 01
19	-0.88746546530-01
20	0.0
21	0.0
22	-0.16437945500 01
23	-0.11188367610 02
24	-0.16393742270 01
25	0.0
26	0.0
27	-0.16663660230 01
28	-0.68680556020 01
29	-0.16597517820 01
30	0.0
31	0.0
32	0.0
33	0.0
34	0.0
35	0.0
36	0.0
37	0.0
38	0.0
39	0.0
40	0.0
41	0.0
42	0.39280003440 00
43	0.28717151270-03
44	-0.39167605910 00
45	0.0
46	0.0
47	0.59303609440 00
48	0.61262434900-03
49	-0.59071993840 00
50	0.0
51	0.0
52	0.37670797430 00
53	0.99603754060-03
54	-0.37314213780 00
55	0.0
56	0.0
57	0.63827156120-01
58	0.13760499030-02
59	-0.59875228560-01
60	0.0



61	0.0
62	-0.2429545074D-01
63	0.1210812751D-02
64	0.2677263620D-01
65	0.0
66	0.0
67	0.0
68	0.0
69	0.0
70	0.0
71	0.1014000000D 07
72	0.1014000000D 07
73	0.1014000000D 07
74	0.1014038420D 07
75	0.1014261630D 07
76	0.1014038452D 07
77	0.1013999776D 07
78	0.1013884020D 07
79	0.1014000754D 07
80	0.1014000000D 07
81	0.1014000000D 07
82	0.1014000000D 07
83	0.2500000000D 02
84	0.2500000000D 02
85	0.2500000000D 02
86	0.2500000000D 02
87	0.2500000000D 02
88	0.2411490471D 02
89	0.2427187487D 02
90	0.2463978352D 02
91	0.2423258019D 02
92	0.2364452284D 02
93	0.2323056484D 02
94	0.2350586481D 02
95	0.2423194982D 02
96	0.2343829368D 02
97	0.2227636650D 02
98	0.2241562105D 02
99	0.2270150631D 02
100	0.2339548232D 02
101	0.2262750919D 02
102	0.2151348362D 02
103	0.2159431099D 02
104	0.2185937901D 02





105	0.22485905170 02
106	0.21795465440 02
107	0.20708330220 02
108	0.20806085770 02
109	0.20955083380 02
110	0.21294435330 02
111	0.20918293520 02
112	0.20370358320 02
113	0.20000000000 02
114	0.20000000000 02
115	0.20000000000 02
116	0.20000000000 02
117	0.20000000000 02

THE KINEMATIC VISCOSITY OF FLUID 50-HB-3520 AT 20 DEGREES C. =  
10.90 SC.CM/SEC

THE DENSITY OF 50-HB-3520 AT 20 DEGREES C. = 1.0596 GM/CC

THE COEFF. OF THERMAL EXPANSION OF 50-HB-3520 AT 20 DEGREES C. =  
0.002278/DEGREE C.

THE THERMAL DIFFUSIVITY OF 50-HB-3520 AT 20 DEGREES C. =  
0.00103 SQ.CM/SEC

THE GRASHOF NUMBER (GR(L)) =  $(G \cdot B \cdot L^3 \cdot (T_H - T_C)) / V^2$  = 946.4

THE U VELOCITY FORCING FUNCTION,  $G \cdot B \cdot T(\text{INITIAL})$ , = 65.962 CM/SQ.S







```

YC(I) = 0.00
NVS(I) = 0
NVCN(I) = 0
NCP(I) = 0
NPS(I) = 0
NQS(I) = 0
C
DO 200 J=1,6
NODE(I,J) = 0
CONTINUE
200
C
DO 300 I=1,MM
T1(I) = 0.00
T(I) = 0.00
NVIS(I) = 0
Q(I) = 0.00
NQIS(I) = 0
RHS(I) = 0.00
C
DO 300 J=1,MM
TM(I,J) = 0.00
CONTINUE
300
C
DO 400 I=1,15
N(I) = 0
C
DO 400 J=1,15
TM$(I,J) = 0.00
CONTINUE
400
C
DO 500 I=1,6
RP$(I) = 0.00
ZP$(I) = 0.00
CONTINUE
500
C
DO 600 I=1,3
XC$(I) = 0.00
YC$(I) = 0.00
CONTINUE
600
C
READ NODE NUMBERS AND COORDINATES
C
C
C
DO 700 J=1,NN

```





```

C      READ (NREAD,3900) WORD,I,XC(I),YC(I)
C      IF (WORD.EQ.STOP) GO TO 800
C      NCN(J) = I
C      700 CONTINUE
C
C      800 NNCN = J-1
C
C      THE ARRAY NCP(J) GENERATES THE GLOBAL PRESSURE INDICES (P1,P2.)
C      THUS PRESSURE NODES ARE LABELED AS CORNER NODES ARE INPUTTED
C      WHEN ONE INPUTS A GLOBAL CORNER NODE FOR J
C
C      DO 900 J=1,NNCN
C      NCP(NCN(J)) = J+NN+NN
C      900 CONTINUE
C
C      SYSTEM TOPOLOGY( ELEMENT NO. AND NODE NUMBERS IN
C      COUNTER-CLOCKWISE FASHION STARTING AT ANY CORNER NODE
C      ALWAYS COUNT FROM UPPER LEFT HAND CORNER
C
C      DO 1000 I=1,NE
C      READ (NREAD,4000) J,NODE(J,1),NODE(J,2),NODE(J,3),NODE(J,4),NODE(J,5),NODE(J,6)
C      1000 CONTINUE
C
C      MAXDIF = 0
C
C      DO 1100 I=1,NE
C      DO 1100 J=1,6
C      DO 1100 K=1,6
C      LL = IABS(NODE(I,J)-NODE(I,K))
C      IF (LL.GT.MAXDIF) MAXDIF=LL
C      IBAND = 2*(MAXDIF+1)
C      NEQ = 2*NN+NNCN
C      1100 CONTINUE
C
C      WRITE (NWRITE,3700) IBAND,NEQ
C
C      READ NODES WHERE BOTH U AND V VELOCITY IS SPECIFIED
C
C      DO 1200 I=1,MM
C      READ (NREAD,3900) WORD,NVELS,VELU,VELV
C      IF (WORD.EQ.STOP) GO TO 1300
C
C      COUT0085
C      COUT0086
C      COUT0087
C      COUT0088
C      COUT0089
C      COUT0090
C      COUT0091
C      COUT0092
C      COUT0093
C      COUT0094
C      COUT0095
C      COUT0096
C      COUT0097
C      COUT0098
C      COUT0099
C      COUT0100
C      COUT0101
C      COUT0102
C      COUT0103
C      COUT0104
C      COUT0105
C      COUT0106
C      COUT0107
C      COUT0108
C      COUT0109
C      COUT0110
C      COUT0111
C      COUT0112
C      COUT0113
C      COUT0114
C      COUT0115
C      COUT0116
C      COUT0117
C      COUT0118
C      COUT0119
C      COUT0120
C      COUT0121
C      COUT0122
C      COUT0123
C      COUT0124
C      COUT0125
C      COUT0126
C      COUT0127
C      COUT0128
C      COUT0129
C      COUT0130
C      COUT0131
C      COUT0132

```



```

      NVS(I) = NVELS
      T(NVS(I)) = VELU
      T(NVS(I)+NN) = VELV
1200 CONTINUE
C
      COUNT NODES HAVING SPECIFIED VELOCITIES
C
1300 NVELS = I-1
C
      READ QX AND QY VALUES AT INTERNAL NODES
C
      DO 1400 I=1,NN
      READ (NREAD,3900) WORD,NQXY,QXNS,QYNS
      IF (WORD.EQ.STOP) GO TO 1500
      NQS(I) = NQXY
      Q(NQS(I)) = QXNS
      Q(NQS(I)+NN) = QYNS
1400 CCNTINUE
C
      COUNT NODES HAVING SPECIFIED QX AND QY
C
1500 NNQXY = I-1
C
      READ NODE NUMBER AND PRESSURE WHERE SPECIFIED
C
      DO 1600 I=1,NN
      READ (NREAD,4100) WORD,NP,PNP
      IF (WORD.EQ.STOP) GO TO 1700
      NPS(I) = NP
      T(NCP(NPS(I))) = PNP
1600 CONTINUE
C
      COUNT BOUNDARY NODES WHERE PRESSURE SPECIFIED
C
1700 NNPS = I-1
C
      NQIS IS A LIST OF THE INDICES OF KNOWN QX,QY
C
      DO 2000 I=1,NNQXY
      NQIS(I) = NQS(I)
      NQIS(I+NNQXY) = NQS(I)+NN
2000 CCNTINUE
C

```

```

COUT01133
COUT01134
COUT01135
COUT01136
COUT01137
COUT01138
COUT01139
COUT01140
COUT01141
COUT01142
COUT01143
COUT01144
COUT01145
COUT01146
COUT01147
COUT01148
COUT01149
COUT01150
COUT01151
COUT01152
COUT01153
COUT01154
COUT01155
COUT01156
COUT01157
COUT01158
COUT01159
COUT01160
COUT01161
COUT01162
COUT01163
COUT01164
COUT01165
COUT01166
COUT01167
COUT01168
COUT01169
COUT01170
COUT01171
COUT01172
COUT01173
COUT01174
COUT01175
COUT01176
COUT01177
COUT01178
COUT01179
COUT01180

```



```

C      NVIS IS A LIST OF KNOWN VELOCITY AND PRESSURE INDICES
C
C      DO 2200 I=1,NNVELS
C      NVIS(I) = NVS(I)
C      NVIS(I+NNVELS) = NVS(I)+NN
C      CONTINUE
C
C      DO 2300 J=1,NNPS
C      NVIS(2*NNVELS+J) = NCP(NPS(J))
C      CONTINUE
C
C      NNHC=NUMBER OF NODES WHERE HEAT TRANSFER COEFFICIENT IS SPECIFIED
C      NNHC = 0
C
C      NTOTQ=TOTAL NUMBER OF KNOWN QX,QY
C      NTOTQ = 2*NNQXY
C
C      NTOTVP=TOTAL NUMBER OF KNOWN VELOCITIES, AND PRESSURES
C      NTOTVP = 2*NNVELS+NNPS
C
C      PRINT ALL INPUT DATA
C
C      WRITE (NWRITE,4300) NN,NE,NNCN
C      WRITE (NWRITE,4400) NNVELS
C      WRITE (NWRITE,4500) NNQXY
C      WRITE (NWRITE,4600) NNPS
C      WRITE (NWRITE,4700)
C
C      DO 2400 I=1,NNCN
C      WRITE (NWRITE,4800) NCN(I),XC(NCN(I)),YC(NCN(I))
C      CONTINUE
C
C      WRITE (NWRITE,4900)
C
C      DO 2500 I=1,NE
C      WRITE (NWRITE,5000) I,NODE(I,1),NODE(I,2),NODE(I,3),NODE(I,4),NODE
C      1(I,5),NODE(I,6)
C      CONTINUE
C
C      WRITE (NWRITE,5100)
C
C      DO 2600 I=1,NNVELS
C      WRITE (NWRITE,5200) I,NVS(I),T(NVS(I)),T(NVS(I)+NN)

```





COUT0229  
COUT0230  
COUT0231  
COUT0232  
COUT0233  
COUT0234  
COUT0235  
COUT0236  
COUT0237  
COUT0238  
COUT0239  
COUT0240  
COUT0241  
COUT0242  
COUT0243  
COUT0244  
COUT0245  
COUT0246  
COUT0247  
COUT0248  
COUT0249  
COUT0250  
COUT0251  
COUT0252  
COUT0253  
COUT0254  
COUT0255  
COUT0256  
COUT0257  
COUT0258  
COUT0259  
COUT0260  
COUT0261  
COUT0262  
COUT0263  
COUT0264  
COUT0265  
COUT0266  
COUT0267  
COUT0268  
COUT0269  
COUT0270  
COUT0271  
COUT0272  
COUT0273  
COUT0274  
COUT0275  
COUT0276

```

2600 CONTINUE
C
WRITE (NWRITE,5300)
C
DO 2700 I=1,NNQXY
WRITE (NWRITE,5200) I,NQS(I),Q(NQS(I)),Q(NQS(I)+NN)
CONTINUE
2700
C
WRITE (NWRITE,5400)
C
DO 2800 I=1,NNPS
WRITE (NWRITE,5600) I,NPS(I),T(NCP(NPS(I)))
CONTINUE
2800
WRITE(NWRITE,1090)
WRITE(NWRITE,1092)
WRITE(NWRITE,1093)
2840
CONTINUE
DO 2850 I=1,MM
RHS(I) = 0.00
DO 2850 J=1,MM
TM(I,J) = 0.00
CONTINUE
2850
DO 2860 I=1,15
DO 2860 J=1,15
TM$(I,J) = 0.00
CONTINUE
2860
C
C
C
C
C
END OF INPUT AND VERIFICATION ROUTINE

DO 3200 K=1,NE
N1=NODE(K,1)
N2=NODE(K,2)
N3=NODE(K,3)
N4=NODE(K,4)
N5=NODE(K,5)
N6=NODE(K,6)
N7=NODE(K,1)+NN
N8=NODE(K,2)+NN
N9=NODE(K,3)+NN
N10=NODE(K,4)+NN
N11=NODE(K,5)+NN
N12=NODE(K,6)+NN
N13=NCP(NODE(K,1))
N14=NCP(NODE(K,3))
N15=NCP(NODE(K,5))

```





```

XC(NODE(K,1))
XC(NODE(K,3))
XC(NODE(K,5))
YC(NODE(K,1))
YC(NODE(K,3))
YC(NODE(K,5))
DO DO
AA = XC$(1)
AA1 = XC$(2)
AA2 = XC$(3)
AA3 = XC$(1)*XC$(2)-XC$(3)
BB1 = YC$(1)
BB2 = YC$(2)
BB3 = YC$(3)
CC1 = XC$(1)*YC$(2)-XC$(3)
CC2 = XC$(1)*YC$(3)-XC$(2)
CC3 = XC$(2)*YC$(3)-XC$(1)
DEL = DABS(0.5D0*(XC$(1)*YC$(2)-YC$(3)+XC$(2)*YC$(3)-YC$(1))+XC$(1)*YC$(2)-YC$(3))
1$CNST=(1.D0*A/(3.D0*DEL))*AA
C1=-B1/6.D0
C2=-B2/6.D0
C3=-B3/6.D0
EE12=-C1/6.D0
EE23=-C2/6.D0
EE31=-C3/6.D0
FF1=B1/2520.D0
FF2=B2/2520.D0
FF3=B3/2520.D0
GG1=C1/2520.D0
GG2=C2/2520.D0
GG3=C3/2520.D0
UU1=TI(N1)
UU2=TI(N2)
UU3=TI(N3)
UU4=TI(N4)
UU5=TI(N5)
UU6=TI(N6)
VV1=TI(N7)
VV2=TI(N8)
VV3=TI(N9)
VV4=TI(N10)
VV5=TI(N11)
VV6=TI(N12)
TM$(1,1)=
TM$(1,2)=
TM$(1,3)=
TM$(1,4)=
0.75D0*(B1*B1+C1*C1)*CONST
(B1*B2+C1*C2)*CONST
-TM$(1,2)*0.25D0
0.

```



```
TM$(1,6) = (B1*B3+C1*C3)*CONST
TM$(1,5) = -TM$(1,6)*.25D0
TM$(3,3) = .75D0*(B2*B2+C2*C2)*CONST
TM$(3,4) = (B2*B3+C2*C3)*CONST
TM$(3,5) = -TM$(3,4)*.25D0
TM$(2,1) = TM$(1,2)
TM$(2,3) = TM$(1,2)
TM$(2,2) = 8.D0/3.D0*(TM$(1,1)+TM$(3,3))+2.D0*TM$(1,2)
TM$(2,5) = 2.D0*TM$(1,6)+TM$(3,4)+TM$(1,2)+4.D0/3.D0*TM$(1,1)
TM$(2,6) = 0.D0
TM$(3,1) = TM$(1,3)
TM$(3,2) = TM$(2,3)
TM$(3,6) = 0.D0
TM$(5,5) = TM$(3,6)*(B3*B3+C3*C3)*CONST
TM$(4,1) = .TM$(1,4)
TM$(4,2) = TM$(2,4)
TM$(4,3) = TM$(3,4)
TM$(4,4) = 8.D0/3.D0*(TM$(3,3)+TM$(5,5))+2.D0*TM$(3,4)
TM$(4,5) = TM$(3,4)
TM$(4,6) = TM$(1,6)+TM$(3,4)+TM$(1,2)+4.D0/3.D0*TM$(5,5)
TM$(5,1) = TM$(2,5)
TM$(5,2) = TM$(3,5)
TM$(5,3) = TM$(4,5)
TM$(5,4) = TM$(4,5)
TM$(5,6) = TM$(1,6)
TM$(6,1) = TM$(2,6)
TM$(6,2) = TM$(2,6)
TM$(6,4) = TM$(4,6)
TM$(6,5) = TM$(5,6)
TM$(6,6) = 8.D0/3.D0*(TM$(5,5)+TM$(1,1))+2.D0*TM$(1,6)

      BEGIN INPUT OF NONLINEAR TERMS
1- TM$(1,1)=TM$(1,1)
2- (78.D0*U1+48.D0*U2-9.D0*U3+12.D0*U4-9.D0*U5+48.D0*U6)*F1
3*G1 - (78.D0*V1+48.D0*V2-9.D0*V3+12.D0*V4-9.D0*V5+48.D0*V6)
1- TM$(2,1)=TM$(2,1)
2- (48.D0*U1+160.D0*U2-32.D0*U3+16.D0*U4-20.D0*U5+80.D0*U6)*F1
3*G1 - (48.D0*V1+160.D0*V2-32.D0*V3+16.D0*V4-20.D0*V5+80.D0*V6)*G1
1- TM$(3,1)=TM$(3,1)
2- (-9.D0*U1-32.D0*U2-18.D0*U3-16.D0*U4+11.D0*U5-20.D0*U6)*F1
3*G1 - (-9.D0*V1-32.D0*V2-18.D0*V3-16.D0*V4+11.D0*V5-20.D0*V6)*G1
      TM$(4,1)=TM$(4,1)
```

```
COUT0325
COUT0326
COUT0327
COUT0328
COUT0329
COUT0330
COUT0331
COUT0332
COUT0333
COUT0334
COUT0335
COUT0336
COUT0337
COUT0338
COUT0339
COUT0340
COUT0341
COUT0342
COUT0343
COUT0344
COUT0345
COUT0346
COUT0347
COUT0348
COUT0349
COUT0350
COUT0351
COUT0352
COUT0353
COUT0354
COUT0355
COUT0356
COUT0357
COUT0358
COUT0359
COUT0360
COUT0361
COUT0362
COUT0363
COUT0364
COUT0365
COUT0366
COUT0367
COUT0368
COUT0369
COUT0370
COUT0371
COUT0372
```





```

1- (12. D0*U1+16. D0*U2-16. D0*U3-96. D0*U4-16. D0*U5+16. D0*U6)*F1      COUT0373
2- (12. D0*V1+16. D0*V2-16. D0*V3-96. D0*V4-16. D0*V5+16. D0*V6)*F1      COUT0374
36)*G1                                     COUT0375
TM$(5,1)=TM$(5,1)                         COUT0376
1- (-9. D0*U1-20. D0*U2+11. D0*U3-16. D0*U4-18. D0*U5-32. D0*U6)*F1      COUT0377
2- (-9. D0*V1-20. D0*V2+11. D0*V3-16. D0*V4-18. D0*V5-32. D0*V6)*F1      COUT0378
36)*G1                                     COUT0379
TM$(6,1)=TM$(6,1)                         COUT0380
1- (48. D0*U1+80. D0*U2-20. D0*U3+16. D0*U4-32. D0*U5+160. D0*U6)*F1    COUT0381
2- (48. D0*V1+80. D0*V2-20. D0*V3+16. D0*V4-32. D0*V5+160. D0*V6)*F1    COUT0382
3V6)*G1                                     COUT0383
TM$(1,2)=TM$(1,2)                         COUT0384
1- (24. D0*U1-32. D0*U2-16. D0*U3-48. D0*U4+4. D0*U5-16. D0*U6)*F1      COUT0385
2- (24. D0*V1-32. D0*V2-16. D0*V3-48. D0*V4+4. D0*V5-16. D0*V6)*F1      COUT0386
3)*G1                                     COUT0387
4D0*U5+48. D0*U6)*F2                      COUT0388
516. D0*V4-16. D0*V5+48. D0*V6)*G2      COUT0389
TM$(2,2)=TM$(2,2)                         COUT0390
1- (-32. D0*U1+384. D0*U2+48. D0*U3+192. D0*U4-48. D0*U5+128. D0*U6)*F1    COUT0391
2- (-32. D0*V1+384. D0*V2+48. D0*V3+192. D0*V4-48. D0*V5+128. D0*V6)*F1    COUT0392
3D0*V6)*G1                                 COUT0393
4D0*U5+192. D0*U6)*F2                      COUT0394
5128. D0*V4-48. D0*V5+192. D0*V6)*G2      COUT0395
TM$(3,2)=TM$(3,2)                         COUT0396
1- (-16. D0*U1+48. D0*U2+120. D0*U3+48. D0*U4-16. D0*U5-16. D0*U6)*F1    COUT0397
2- (-16. D0*V1+48. D0*V2+120. D0*V3+48. D0*V4-16. D0*V5-16. D0*V6)*F1    COUT0398
3V6)*G1                                     COUT0399
40*U5-48. D0*U6)*F2                      COUT0400
516. D0*V4+4. D0*V5-48. D0*V6)*G2      COUT0401
TM$(4,2)=TM$(4,2)                         COUT0402
1- (-48. D0*U1+192. D0*U2+48. D0*U3+384. D0*U4-32. D0*U5+128. D0*U6)*F1    COUT0403
2- (-48. D0*V1+192. D0*V2+48. D0*V3+384. D0*V4-32. D0*V5+128. D0*V6)*F1    COUT0404
3D0*V6)*G1                                 COUT0405
46. D0*U5+128. D0*U6)*F2                      COUT0406
5+128. D0*V4-16. D0*V5+128. D0*V6)*G2      COUT0407
TM$(5,2)=TM$(5,2)                         COUT0408
1- (4. D0*U1-48. D0*U2-16. D0*U3-32. D0*U4+24. D0*U5-16. D0*U6)*F1      COUT0409
2- (4. D0*V1-48. D0*V2-16. D0*V3-32. D0*V4+24. D0*V5-16. D0*V6)*F1      COUT0410
3)*G1                                     COUT0411
40*U5-32. D0*U6)*F2                      COUT0412
56. D0*V4+24. D0*V5-32. D0*V6)*G2      COUT0413
TM$(6,2)=TM$(6,2)                         COUT0414
1- (-16. D0*U1+128. D0*U2-16. D0*U3+128. D0*U4-16. D0*U5+128. D0*U6)*F1    COUT0415
2- (-16. D0*V1+128. D0*V2-16. D0*V3+128. D0*V4-16. D0*V5+128. D0*V6)*F1    COUT0416
3D0*V6)*G1                                 COUT0417
4D0*U5+384. D0*U6)*F2                      COUT0418
5128. D0*V4-32. D0*V5+384. D0*V6)*G2      COUT0419
TM$(1,3)=TM$(1,3)                         COUT0420

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1-(-18.D0*U1-32.D0*U2-9.D0*U3-20.D0*U4+11.D0*U5-16.D0*U6)*F2      COUT0421
2-(-18.D0*V1-32.D0*V2-9.D0*V3-20.D0*V4+11.D0*V5-16.D0*V6)*F2      COUT0422
3)*G2                             COUT0423
TM$(2,3)=TM$(2,3)                COUT0424
1-(-32.D0*U1+160.D0*U2+48.D0*U3+80.D0*U4-20.D0*U5+16.D0*U6)*F2      COUT0425
2-(-32.D0*V1+160.D0*V2+48.D0*V3+80.D0*V4-20.D0*V5+16.D0*V6)*F2      COUT0426
3)*G2                             COUT0427
TM$(3,3)=TM$(3,3)                COUT0428
1-(-9.D0*U1+48.D0*U2+78.D0*U3+48.D0*U4-9.D0*U5+12.D0*U6)*F2      COUT0429
2-(-9.D0*V1+48.D0*V2+78.D0*V3+48.D0*V4-9.D0*V5+12.D0*V6)*F2      COUT0430
3)*G2                             COUT0431
TM$(4,3)=TM$(4,3)                COUT0432
1-(-20.D0*U1+80.D0*U2+48.D0*U3+160.D0*U4-32.D0*U5+16.D0*U6)*F2      COUT0433
2-(-20.D0*V1+80.D0*V2+48.D0*V3+160.D0*V4-32.D0*V5+16.D0*V6)*F2      COUT0434
3)*G2                             COUT0435
TM$(5,3)=TM$(5,3)                COUT0436
1-(-11.D0*U1-20.D0*U2-9.D0*U3-32.D0*U4-18.D0*U5-16.D0*U6)*F2      COUT0437
2-(-11.D0*V1-20.D0*V2-9.D0*V3-32.D0*V4-18.D0*V5-16.D0*V6)*F2      COUT0438
3)*G2                             COUT0439
TM$(6,3)=TM$(6,3)                COUT0440
1-(-16.D0*U1+16.D0*U2+12.D0*U3+16.D0*U4-16.D0*U5-96.D0*U6)*F2      COUT0441
2-(-16.D0*V1+16.D0*V2+12.D0*V3+16.D0*V4-16.D0*V5-96.D0*V6)*F2      COUT0442
3)*G2                             COUT0443
TM$(1,4)=TM$(1,4)                COUT0444
1-(-24.D0*U1-16.D0*U2+4.D0*U3-48.D0*U4-16.D0*U5-32.D0*U6)*F2      COUT0445
2-(-24.D0*V1-16.D0*V2+4.D0*V3-48.D0*V4-16.D0*V5-32.D0*V6)*F2      COUT0446
3)*G2                             COUT0447
TM$(1,4)=TM$(1,4)                COUT0448
4)*G2                             COUT0449
58.D0*U5-16.D0*U6)*F3            COUT0450
58.D0*V4+4.D0*V5-16.D0*V6)*G3  COUT0451
TM$(2,4)=TM$(2,4)                COUT0452
1-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F2  COUT0453
2-(-16.D0*V1+128.D0*V2-16.D0*V3+128.D0*V4-16.D0*V5+128.D0*V6)*F2  COUT0454
3D0*U5+128.D0*U6)*F3            COUT0455
48.D0*U4-48.D0*U5+128.D0*U6)*F3 COUT0456
TM$(3,4)=TM$(3,4)                COUT0457
1-(-4.D0*U1-16.D0*U2+24.D0*U3-32.D0*U4+16.D0*U5-48.D0*U6)*F2      COUT0458
2-(-4.D0*V1-16.D0*V2+24.D0*V3-32.D0*V4+16.D0*V5-48.D0*V6)*F2      COUT0459
3)*G2                             COUT0460
4.D0*U5-16.D0*U6)*F3            COUT0461
5+48.D0*V4-16.D0*V5-16.D0*V6)*G3 COUT0462
TM$(4,4)=TM$(4,4)                COUT0463
1-(-48.D0*U1+128.D0*U2-32.D0*U3+384.D0*U4+48.D0*U5+192.D0*U6)*F2  COUT0464
2-(-48.D0*V1+128.D0*V2-32.D0*V3+384.D0*V4+48.D0*V5+192.D0*V6)*F2  COUT0465
3D0*U5+128.D0*U6)*F3            COUT0466
42.D0*U4-32.D0*U5+128.D0*U6)*F3 COUT0467
5+384.D0*V4-32.D0*V5+128.D0*V6)*G3 COUT0468
TM$(5,4)=TM$(5,4)

```



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1-(-16.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*F2      COUT0469
2-(-16.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*F3      COUT0470
3*V6)*G2      COUT0471
4*U5-16.D0*U6)*F3      COUT0472
5.TM$(6,4)=TM$(6,4)      COUT0473
1-(-32.D0*U1+128.D0*U2-48.D0*U3+192.D0*U4+48.D0*U5+384.D0*U6)*F2      COUT0474
2-(-32.D0*U1+128.D0*U2-48.D0*U3+192.D0*U4+48.D0*U5+384.D0*U6)*F3      COUT0475
3D0*V6)*G2      COUT0476
46.D0*U5+128.D0*U6)*F3      COUT0477
5+128.D0*U4-16.D0*U5+48.D0*U6)*F3      COUT0478
1-(-18.D0*U1-16.D0*U2+11.D0*U3-20.D0*U4-9.D0*U5-32.D0*U6)*F3      COUT0479
2-(-18.D0*U1-16.D0*U2+11.D0*U3-20.D0*U4-9.D0*U5-32.D0*U6)*F3      COUT0480
36)*G3      COUT0481
1-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3      COUT0482
2-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3      COUT0483
3V6)*G3      COUT0484
1-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3      COUT0485
2-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3      COUT0486
3V6)*G3      COUT0487
1-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3      COUT0488
2-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3      COUT0489
3V6)*G3      COUT0490
1-(-20.D0*U1+16.D0*U2-32.D0*U3+160.D0*U4+48.D0*U5+80.D0*U6)*F3      COUT0491
2-(-20.D0*U1+16.D0*U2-32.D0*U3+160.D0*U4+48.D0*U5+80.D0*U6)*F3      COUT0492
3V6)*G3      COUT0493
1-(-20.D0*U1+16.D0*U2-32.D0*U3+160.D0*U4+48.D0*U5+80.D0*U6)*F3      COUT0494
2-(-20.D0*U1+16.D0*U2-32.D0*U3+160.D0*U4+48.D0*U5+80.D0*U6)*F3      COUT0495
3V6)*G3      COUT0496
1-(-9.D0*U1+12.D0*U2-9.D0*U3+48.D0*U4+78.D0*U5+48.D0*U6)*F3      COUT0497
2-(-9.D0*U1+12.D0*U2-9.D0*U3+48.D0*U4+78.D0*U5+48.D0*U6)*F3      COUT0498
3V6)*G3      COUT0499
1-(-32.D0*U1+16.D0*U2-20.D0*U3+80.D0*U4+48.D0*U5+160.D0*U6)*F3      COUT0500
2-(-32.D0*U1+16.D0*U2-20.D0*U3+80.D0*U4+48.D0*U5+160.D0*U6)*F3      COUT0501
3V6)*G3      COUT0502
1-(-24.D0*U1-16.D0*U2+4.D0*U3-48.D0*U4-16.D0*U5-32.D0*U6)*F1      COUT0503
2-(-24.D0*U1-16.D0*U2+4.D0*U3-48.D0*U4-16.D0*U5-32.D0*U6)*F1      COUT0504
3V6)*G3      COUT0505
1-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0506
2-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0507
3V6)*G3      COUT0508
1-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0509
2-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0510
3V6)*G3      COUT0511
1-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0512
2-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0513
3V6)*G3      COUT0514
1-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0515
2-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1      COUT0516
3V6)*G3      COUT0517

```





CC

```

1- (4. D0*U1-16. D0*U2+24. D0*U3-32. D0*U4-16. D0*U5-48. D0*U6)*F1
2- (4. D0*U1-16. D0*U2+24. D0*U3-32. D0*U4-16. D0*U5-48. D0*U6)*F1
3)*G1
4)*G1
5)*G1
6)*G1
7)*G1
8)*G1
9)*G1
10)*G1
11)*G1
12)*G1
13)*G1
14)*G1
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77)*G1
78)*G1
79)*G1
80)*G1
81)*G1
82)*G1
83)*G1
84)*G1
85)*G1
86)*G1
87)*G1
88)*G1
89)*G1
90)*G1
91)*G1
92)*G1
93)*G1
94)*G1
95)*G1
96)*G1
97)*G1
98)*G1
99)*G1
100)*G1

```

THIS ENDS ADDITION OF NONLINEAR TERMS TO THE LOCAL ARRAY

```

TM$(7,7) = TM$(1,1)
TM$(7,8) = TM$(1,2)
TM$(7,9) = TM$(1,3)
TM$(7,10) = TM$(1,4)
TM$(7,11) = TM$(1,5)
TM$(7,12) = TM$(1,6)
TM$(8,7) = TM$(2,1)
TM$(8,8) = TM$(2,2)
TM$(8,9) = TM$(2,3)
TM$(8,10) = TM$(2,4)
TM$(8,11) = TM$(2,5)
TM$(8,12) = TM$(2,6)
TM$(9,7) = TM$(3,1)
TM$(9,8) = TM$(3,2)
TM$(9,9) = TM$(3,3)
TM$(9,10) = TM$(3,4)
TM$(9,11) = TM$(3,5)
TM$(9,12) = TM$(3,6)
TM$(10,7) = TM$(4,1)
TM$(10,8) = TM$(4,2)
TM$(10,9) = TM$(4,3)
TM$(10,10) = TM$(4,4)

```

```

COUT0517
COUT0518
COUT0519
COUT0520
COUT0521
COUT0522
COUT0523
COUT0524
COUT0525
COUT0526
COUT0527
COUT0528
COUT0529
COUT0530
COUT0531
COUT0532
COUT0533
COUT0534
COUT0535
COUT0536
COUT0537
COUT0538
COUT0539
COUT0540
COUT0541
COUT0542
COUT0543
COUT0544
COUT0545
COUT0546
COUT0547
COUT0548
COUT0549
COUT0550
COUT0551
COUT0552
COUT0553
COUT0554
COUT0555
COUT0556
COUT0557
COUT0558
COUT0559
COUT0560
COUT0561
COUT0562
COUT0563
COUT0564

```



```

TM$(10,10)
TM$(11,11)
TM$(12,12)
TM$(13,13)
TM$(14,14)
TM$(15,15)
TM$(16,16)
TM$(17,17)
TM$(18,18)
TM$(19,19)
TM$(20,20)
TM$(21,21)
TM$(22,22)
TM$(23,23)
TM$(24,24)
TM$(25,25)
TM$(26,26)
TM$(27,27)
TM$(28,28)
TM$(29,29)
TM$(30,30)
TM$(31,31)
TM$(32,32)
TM$(33,33)
TM$(34,34)
TM$(35,35)
TM$(36,36)
TM$(37,37)
TM$(38,38)
TM$(39,39)
TM$(40,40)
TM$(41,41)
TM$(42,42)
TM$(43,43)
TM$(44,44)
TM$(45,45)
TM$(46,46)
TM$(47,47)
TM$(48,48)
TM$(49,49)
TM$(50,50)
TM$(51,51)
TM$(52,52)
TM$(53,53)
TM$(54,54)
TM$(55,55)
TM$(56,56)
TM$(57,57)
TM$(58,58)
TM$(59,59)
TM$(60,60)
TM$(61,61)
TM$(62,62)
TM$(63,63)
TM$(64,64)
TM$(65,65)
TM$(66,66)
TM$(67,67)
TM$(68,68)
TM$(69,69)
TM$(70,70)
TM$(71,71)
TM$(72,72)
TM$(73,73)
TM$(74,74)
TM$(75,75)
TM$(76,76)
TM$(77,77)
TM$(78,78)
TM$(79,79)
TM$(80,80)
TM$(81,81)
TM$(82,82)
TM$(83,83)
TM$(84,84)
TM$(85,85)
TM$(86,86)
TM$(87,87)
TM$(88,88)
TM$(89,89)
TM$(90,90)
TM$(91,91)
TM$(92,92)
TM$(93,93)
TM$(94,94)
TM$(95,95)
TM$(96,96)
TM$(97,97)
TM$(98,98)
TM$(99,99)
TM$(100,100)

```

```

COUT0565
COUT0566
COUT0567
COUT0568
COUT0569
COUT0570
COUT0571
COUT0572
COUT0573
COUT0574
COUT0575
COUT0576
COUT0577
COUT0578
COUT0579
COUT0580
COUT0581
COUT0582
COUT0583
COUT0584
COUT0585
COUT0586
COUT0587
COUT0588
COUT0589
COUT0590
COUT0591
COUT0592
COUT0593
COUT0594
COUT0595
COUT0596
COUT0597
COUT0598
COUT0599
COUT0600
COUT0601
COUT0602
COUT0603
COUT0604
COUT0605
COUT0606
COUT0607
COUT0608
COUT0609
COUT0610
COUT0611
COUT0612

```









```

N(10)=N10
N(11)=N11
N(12)=N12
N(13)=N13
N(14)=N14
N(15)=N15
C
DO 3100 I$=1,15
I = N(I$)
C
DO 3100 J$=1,15
J = N(J$)
TM(I,J) = TM(I,J)+TM$(I$,J$)
3100 CONTINUE
C
3200 CONTINUE
C
DO 3300 I=1,NNQXY
RHS(NQS(I)) = RHS(NQS(I))+Q(NQS(I))
RHS(NQS(I)+NN) = RHS(NQS(I)+NN)+Q(NQS(I)+NN)
3300 CONTINUE
C
3400 CONTINUE
C
INSERT SYSTEM BOUNDARY CONDITIONS
C
DO 3500 I=1,MM
C
DO 3500 J=1,NTOTVP
JX = NV IS(J)
RHS(I) = RHS(I)-TM(I,JX)*T(JX)
TM(I,JX) = 0.00
TM(JX,I) = 0.00
TM(JX,JX) = 1.00
RHS(JX) = T(JX)
3500 CONTINUE
C
M = 1
ND = 82
IDGT = 82
CALL LEQ2F (TM,M,ND,IA,RHS,IDGT,WKAREA,IER)
WRITE (NWRITE,5700)
DO 322 J=1,MM
TDIFF=DABS(T1(J)-T(J))
EPSLN=1.D-06
IF(TDIFF-EPSLN) 322,324,324

```



```

322 CONTINUE
324 CONTINUE
C
DO 3600 I=1,MM
  T(I)=RHS(I)
  T1(I)=T(I)
  WRITE(NWRITE,5800) I,T(I)
CONTINUE
3600 IF(J.NE.MM) GO TO 2840
      WRITE(NWRITE,1091)
C
1090 FORMAT(///,5X,'NODAL VARIABLE IS THE U-VELOCITY AT NODES 1 - 35;',
1//,5X,'THE V-VELOCITY AT NODES 36 - 70;',
2//,5X,'AND THE PRESSURES AT NODES 71 - 82.',///)
1091 FORMAT(1H1)
1092 FORMAT(///,5X,'THE FIRST SEQUENCE OF 82 NODAL VARIABLES',
1//,5X,'REPRESENTS A LINEAR, STEADY STATE SYSTEM;',
2//,5X,'WHILE THE SECOND ANALYSIS OF THE 82 VALUES CORRESPONDS',
3//,5X,'TO A NONLINEAR ANALYSIS OF THE SOLUTION DOMAIN.',///)
1093 FORMAT(///,5X,'A PRESSURE GRADIENT IN THE HORIZONTAL SHEAR',
1//,5X,'DIRECTION OF -3 UNITS IN MAGNITUDE HAS BEEN ADDED',
2//,5X,'TO PRODUCE A CURVED VELOCITY PROFILE.',///)
1094 FORMAT(1H1,15X,'THIS IS A 2-D NONLINEAR COUETTE FLOW PROBLEM',//)
3700 FORMAT(5X,'IBAND=',I3//,5X,'NEQ =',I3,//)
3800 FORMAT(6X,A4,I10,2F10.0)
3900 FORMAT(6X,A4,I10,F10.0)
4000 FORMAT(6X,A4,I10,F10.0)
4100 FORMAT(5X,'NO. OF CORNER NODES=',I3,///)
4300 FORMAT(5X,'NO. OF CORNERS=',I3,///)
4400 FORMAT(5X,'NNVELS=',I3,///)
4500 FORMAT(5X,'NNQXY=',I3,///)
4600 FORMAT(5X,'NNPS=',I3,///)
4700 FORMAT(5X,'SUMMARY OF NODAL COORDINATES',//,
1 7X,I1,I12X,X(I),I13X,Y(I),//)
4800 FORMAT(5X,I3,2(7X,F10.3))
4900 FORMAT(///,5X,'LISTING OF SYSTEM TOPOLOGY',//,5X,
1 'ELEMENT NUMBER',20X,'NODE NUMBERS',//)
5000 FORMAT(5X,I3,10X,6(5X,I3))
5100 FORMAT(///,7X,'NODES WHERE U VELOCITY IS SPECIFIED',//)
5200 FORMAT(1,5X,'NODE',5X,F12.3,3X,F12.3)
5300 FORMAT(2X,2(4X,I3),3X,F12.3)
5400 FORMAT(///,5X,'NODES WHERE QX AND QY ARE SPECIFIED',
1//,2X,'NODE',1X,
2//,5X,'NODES WHERE PRESSURE IS SPECIFIED',
3//,5X,'I',5X,'NODE',5X,F12.3)
5600 FORMAT(7X,I3,3X,I3,10X,F12.3)
5700 FORMAT(///,5X,'NODE NO.',6X,'NODE VARIABLES',//)
      QY',//)

```





5800 FORMAT (9X,I3,5X,D17.10,/)
STOP
END

COUT0757
COUT0758
COUT0759



# STEADY STATE FLUID MECHANICS PROBLEM

```

IMPLICIT REAL*8(A-H,O-Z,$)
DATA NREAD/5/

THE U VELOCITY IS IN THE FIRST NN POSITIONS OF X(I)
THE V VELOCITY IS IN THE SECOND NN POSITIONS OF X I.E. X(I+NN)
THE P PRESSURE IS IN THE NN+NN+I POSITIONS OF X I.E. X(I+NN+NN)
THE T TEMPERATURE IS IN THE NN+NN+NNCN+I POSITIONS OF X I.E.
X(I+NN+NN+NNCN)
THERE ARE NNCN PRESSURE NODES (NNCN=NUMBER OF CORNER NODES)

TM MUST BE DIMENSIONED 3*NN+NNCN X 3*NN+NNCN

DATA NWRITE/6/
DATA STOP/.STOP./
DIMENSION XC(125),YC(125),NODE(125,6),NVS(125),NCN(125)
DIMENSION X(117),NVIS(117),NCP(125),NPS(125),Q(117)
DIMENSION NQS(125),T1(117)
DIMENSION TM$(21,21),N(21),NQIS(117)
DIMENSION RP$(6),ZP$(6),WKAREA(15000)
DIMENSION XC$(3),YC$(3),RHS(117),TM(117,117)

SPECIFY WHETHER TWO DIMENSIONAL, INCLUDING NON-LINEAR TERMS,
(NCASE=1) OR AXISYMMETRIC (NCASE=2)

READ(NREAD,500)NCASE
WRITE(NWRITE,600)
IF(NCASE.EQ.1)GO TO 5
WRITE(NWRITE,2015)
GC TO 6
5 WRITE(NWRITE,2020)
6 CONTINUE

THE FIRST PART OF THE PROGRAM CAN BE CONSIDERED AS AN INPUT ROUTINE
IN WHICH LINES 540 TO 2970 ARE INPUT VERIFICATION OF ALL DATA.
SUCH A SECTION WOULD BE PART OF ANY FINITE ELEMENT PROGRAM.

```

FLSS01110  
 FLSS01120  
 FLSS01130  
 FLSS01140  
 FLSS01150  
 FLSS01160  
 FLSS01170  
 FLSS01180  
 FLSS01190  
 FLSS01200  
 FLSS02210  
 FLSS02220  
 FLSS02230  
 FLSS02240  
 FLSS02250  
 FLSS02260  
 FLSS02270  
 FLSS02280  
 FLSS02290  
 FLSS02300  
 FLSS02310  
 FLSS02320  
 FLSS02330  
 FLSS02340  
 FLSS02350  
 FLSS02360  
 FLSS03370  
 FLSS03380  
 FLSS03390  
 FLSS03400  
 FLSS03410  
 FLSS03420  
 FLSS03430  
 FLSS03440  
 FLSS03460



READ IN NUMBER OF NODES AND ELEMENTS AND NO. OF CCRNER NODES

READ(NREAD,1005)NN,NE,NNCN

INITIALIZE ALL PARAMETERS

MM=2\*NN+NNCN  
MMM=3\*NN+NNCN  
DO 50 I=1,MMM

XC(I)=0.00  
YC(I)=0.00

NVS(I)=0

NCN(I)=0

NCP(I)=0

NPS(I)=0

NGS(I)=0

DO 50 J=1,6

NODE(I,J)=0

CCNTINUE

DO 51 I=1,MMM

TI(I)=0.00

X(I)=0.00

NVIS(I)=0

QC(I)=0.00

NGIS(I)=0

RHS(I)=0.00

DO 51 J=1,MMM

TM(I,J)=0.00

CCNTINUE

DO 52 I=1,21

N(I)=0

DO 52 J=1,21

TM\$(I,J)=0.00

CCNTINUE

DO 53 I=1,6

RP\$(I)=0.00

ZP\$(I)=0.00

CCNTINUE

DO 54 I=1,3

XC\$(I)=0.00

YC\$(I)=0.00

CCNTINUE

54

READ NODE NUMBER AND COORDINATES

DO 100 J=1,NN

FLSS0470  
FLSS0480  
FLSS0490  
FLSS0500  
FLSS0510  
FLSS0520  
FLSS0530  
FLSS0540  
FLSS0550  
FLSS0560  
FLSS0570  
FLSS0580  
FLSS0590  
FLSS0600  
FLSS0610  
FLSS0620  
FLSS0630  
FLSS0640  
FLSS0650  
FLSS0660  
FLSS0670  
FLSS0680  
FLSS0690  
FLSS0700  
FLSS0710  
FLSS0720  
FLSS0730  
FLSS0740  
FLSS0750  
FLSS0760  
FLSS0770  
FLSS0780  
FLSS0790  
FLSS0800  
FLSS0810  
FLSS0820  
FLSS0830  
FLSS0840  
FLSS0850  
FLSS0860  
FLSS0870  
FLSS0880  
FLSS0890  
FLSS0900  
FLSS0910  
FLSS0920  
FLSS0930



```

      READ(NREAD,1006)WORD,I,XC(I),YC(I)
      IF(WORD.EQ.STOP) GO TO 101
      NCN(J)=I
      100 CONTINUE
      101 NNCN=J-1

      THE ARRAY NCP(J) GENERATES THE GLOBAL PRESSURE INDICES (P1,P2,ETC.)
      THUS PRESSURE NODES ARE LABELED AS CORNER NODES AND ARE INPUTED
      WHEN ONE INPUTS A GLOBAL CORNER NODE FOR J

      DC 107 J=1,NNCN
      NCP(NCN(J))=J+NN+NN
      107 CCNTINUE

      READ SYSTEM TOPOLOGY (ELEMENT NO. AND NODE NUMBERS IN
      COUNTERCLOCKWISE FASHION STARTING AT THE UPPER LEFT
      HAND CORNER NODE).

      DO 105 I=1,NE
      READ(NREAD,1010)J,NODE(J,1),NODE(J,2),NODE(J,3),
      1 NODE(J,4),NODE(J,5),NODE(J,6)
      105 CCNTINUE
      MAXDIF=0
      DO 108 I=1,NE
      DC 108 J=1,6
      DO 108 K=1,6
      LL=IABS(NODE(I,J)-NODE(I,K))
      IF(LL.GT.MAXDIF) MAXDIF=LL
      IBAND=2*(MAXDIF+1)
      NEQ=3*NN+NNCN
      108 CONTINUE
      WRITE(NWRITE,1017)IBAND,NEQ
      1017 FORMAT(5X,'IBAND=',I3,'/',5X,'NEQ=',I3,'/')

      READ NODES WHERE BOTH U AND V VELOCITY IS SPECIFIED

      DO 110 I=1,MM
      READ(NREAD,1006)WORD,NVELS,VELU,VELV
      IF(WORD.EQ.STOP) GO TO 111
      NVS(I)=NVELS
      X(NVS(I))=VELU
      X(NVS(I)+NN)=VELV
      110 CONTINUE

      COUNT NODES HAVING SPECIFIED VELOCITIES

      111 NNVELS=I-1

```

```

FLSS0940
FLSS0950
FLSS0960
FLSS0970
FLSS0980
FLSS0990
FLSS1000
FLSS1010
FLSS1020
FLSS1030
FLSS1040
FLSS1050
FLSS1060
FLSS1070
FLSS1080
FLSS1090
FLSS1100
FLSS1110
FLSS1120
FLSS1130
FLSS1140
FLSS1150
FLSS1160
FLSS1170
FLSS1180
FLSS1190
FLSS1200
FLSS1210
FLSS1220
FLSS1230
FLSS1240
FLSS1250
FLSS1270
FLSS1280
FLSS1290
FLSS1300
FLSS1310
FLSS1320
FLSS1330
FLSS1340
FLSS1350
FLSS1360
FLSS1370
FLSS1380
FLSS1390
FLSS1400
FLSS1410

```





```

READ QX AND QY VALUES AT INTERNAL NODES
DO 125 I=1,NN
  READ(NREAD,1006)WORD,NQXY,QXNS,QYNS
  IF(WORD.EQ.STOP) GO TO 126
  NQS(I)=NQXY
  Q(NQS(I))=QXNS
  Q(NQS(I)+NN)=QYNS
125 CONTINUE
COUNT NODES HAVING SPECIFIED QX AND QY
126 NNQXY=I-1
READ NODE NUMBER AND PRESSURE WHERE SPECIFIED
DO 130 I=1,NN
  READ(NREAD,1025)WORD,NP,PNP
  IF(WORD.EQ.STOP) GO TO 135
  NPS(I)=NP
  X(NCP(NPS(I)))=PNP
130 CONTINUE
COUNT BOUNDARY NODES WHERE PRESSURE IS SPECIFIED
135 NNPS=I-1
READ NODE NUMBER AND TEMPERATURE WHERE SPECIFIED
DO 140 I=1,MM
  READ(NREAD,1025)WORD,NTEMP,TNT
  IF(WORD.EQ.STOP) GO TO 145
  NVS(I+NNVELS)=NTEMP
  X(NVS(I+NNVELS)+MM)=TNT
140 CONTINUE
COUNT NODES HAVING SPECIFIED TEMPERATURES
145 NNTS=I-1
READ NODE NUMBERS AND QZC WHERE SPECIFIED
DO 141 I=1,MM
  READ(NREAD,1025)WORD,NQZC,QZCNS
  IF(WORD.EQ.STOP) GO TO 146
  NQS(NNQXY+I)=NQZC
  Q(NQS(NNQXY+I)+2*NN)=QZCNS
141 CONTINUE

```

```

FLSSS1420
FLSSS1430
FLSSS1440
FLSSS1450
FLSSS1460
FLSSS1470
FLSSS1480
FLSSS1490
FLSSS1500
FLSSS1510
FLSSS1520
FLSSS1530
FLSSS1540
FLSSS1550
FLSSS1560
FLSSS1570
FLSSS1580
FLSSS1590
FLSSS1600
FLSSS1610
FLSSS1620
FLSSS1630
FLSSS1640
FLSSS1650
FLSSS1660
FLSSS1670
FLSSS1680
FLSSS1690
FLSSS1700
FLSSS1710
FLSSS1720
FLSSS1730
FLSSS1740
FLSSS1750
FLSSS1760
FLSSS1770
FLSSS1780
FLSSS1790
FLSSS1800
FLSSS1810
FLSSS1820
FLSSS1830
FLSSS1840
FLSSS1850
FLSSS1860
FLSSS1870
FLSSS1880
FLSSS1890

```



FLSS11900  
FLSS11910  
FLSS11920  
FLSS11930  
FLSS11940  
FLSS11950  
FLSS11960  
FLSS11970  
FLSS11980  
FLSS11990  
FLSS12000  
FLSS12010  
FLSS12020  
FLSS12030  
FLSS12040  
FLSS12050  
FLSS12060  
FLSS12070  
FLSS12080  
FLSS12090  
FLSS12100  
FLSS12110  
FLSS12120  
FLSS12130  
FLSS12140  
FLSS12150  
FLSS12160  
FLSS12170  
FLSS12180  
FLSS12190  
FLSS12200  
FLSS12210  
FLSS12220  
FLSS12230  
FLSS12240  
FLSS12250  
FLSS12260  
FLSS12270  
FLSS12280  
FLSS12290  
FLSS12300  
FLSS12310  
FLSS12320  
FLSS12330  
FLSS12340  
FLSS12350  
FLSS12360  
FLSS12370

COUNT NODES WHERE QZC IS SPECIFIED

146 NNQZC=I-1

READ NODE NUMBERS AND HEAT FLUX QZ WHERE SPECIFIED

DO 142 I=1,MM  
READ(NREAD,1025) WORD,NQZ,QZNS  
IF(WORD.EQ.STOP) GO TO 147  
NQS(NNQXY+NNQZC+I)=NQZ  
Q(NQS(NNQXY+NNQZC+I)+MM)=QZNS  
142 CONTINUE

COUNT NODES WHERE HEAT FLUX QZ IS SPECIFIED

147 NNQZ=I-1

NQIS IS A LIST OF INDICES OF KNOWN QX,QY,QZC AND QZ

DO 1140 I=1,NNQXY  
NQIS(I)=NQS(I)  
NCIS(I+NNQXY)=NQS(I)+NN  
1140 CONTINUE

DO 1141 I=1,NNQZC  
NQIS(2\*NNQXY+I)=NQS(NNQXY+I)+2\*NN  
1141 CONTINUE

DO 1145 I=1,NNQZ  
NQIS(2\*NNQXY+NNQZC+I)=NQS(NNQXY+NNQZC+I)+MM  
1145 CCNTINUE

NVIS IS A LIST OF KNOWN VELOCITY, PRESSURE, AND TEMPERATURE INDICES

DO 1150 I=1,NNVELS  
NVIS(I)=NVS(I)  
NVIS(I+NNVELS)=NVS(I)+NN  
1150 CONTINUE

DO 1155 J=1,NNPS  
NVIS(2\*NNVELS+J)=NCP(NPS(J))  
1155 CONTINUE

DO 1160 K=1,NNTS  
NVIS(2\*NNVELS+NNPS+K)=NVS(K+NNVELS)+MM  
1160 CONTINUE

NNHC=NUMBER OF NODES WHERE HEAT TRANSFER COEFFICIENT IS SPECIFIED

NNHC=0



```

NTOTC=TOTAL NUMBER OF KNOWN QX,QY,QZC, AND QZ
      NTOTQ=2*NNQXY+NNQZC+NNQZ
NTOTVP=TOTAL NUMBER OF KNOWN VELOCITIES, PRESSURES, AND TEMPERATURES
      NTOTVP=2*NNVELS+NNPS+NNNTS

PRINT ALL INPUT DATA
      WRITE(NWRITE,1035)NN,NE,NNCN
      WRITE(NWRITE,1036)NNVELS
      WRITE(NWRITE,1037)NNQXY
      WRITE(NWRITE,1038)NNPS
      WRITE(NWRITE,1039)NNNTS
      WRITE(NWRITE,1034)NNQZC
      WRITE(NWRITE,1040)NNQZ
      WRITE(NWRITE,1041)
      DC 150 I=1,NNCN
      WRITE(NWRITE,1045)NCN(I),XC(NCN(I)),YC(NCN(I))
      CONTINUE
      WRITE(NWRITE,1050)
      DO 155 I=1,NE
      WRITE(NWRITE,1055)I,NODE(I,1),NODE(I,2),NODE(I,3),
155 CONTINUE
      WRITE(NWRITE,1060)
      DO 160 I=1,NNVELS
      WRITE(NWRITE,1065)I,NVS(I),X(NVS(I)),X(NVS(I)+NN)
      CONTINUE
      WRITE(NWRITE,1070)
      DO 165 I=1,NNQXY
      WRITE(NWRITE,1065)I,NQS(I),Q(NQS(I)),Q(NQS(I)+NN)
      CONTINUE
      WRITE(NWRITE,1080)
      DO 170 I=1,NNPS
      WRITE(NWRITE,1085)I,NPS(I),X(NCP(NPS(I)))
      CONTINUE
      WRITE(NWRITE,1081)
      DO 171 I=1,NNNTS
      WRITE(NWRITE,1085)I,NVS(I+NNVELS),X(NVS(I+NNVELS)+MM)
      CONTINUE
      WRITE(NWRITE,1083)
      DO 173 I=1,NNQZC
      WRITE(NWRITE,1085)I,NQS(I+NNQXY),Q(NQS(I+NNQXY)+2*NN)
      CONTINUE
      WRITE(NWRITE,1082)
      DC 172 I=1,NNQZ

```

```

SS2380
FLSS2390
FLSS2400
FLSS2410
FLSS2420
FLSS2430
FLSS2440
FLSS2450
FLSS2460
FLSS2470
FLSS2480
FLSS2490
FLSS2500
FLSS2510
FLSS2520
FLSS2530
FLSS2540
FLSS2550
FLSS2560
FLSS2570
FLSS2580
FLSS2590
FLSS2600
FLSS2610
FLSS2620
FLSS2630
FLSS2640
FLSS2650
FLSS2660
FLSS2670
FLSS2680
FLSS2690
FLSS2700
FLSS2710
FLSS2720
FLSS2730
FLSS2740
FLSS2750
FLSS2760
FLSS2770
FLSS2780
FLSS2790
FLSS2800
FLSS2810
FLSS2820
FLSS2830
FLSS2840
FLSS2850

```





```

172 WRITE(NWRITE,1085)I,NQS(I+NNQXY+NNQZC),Q(NQS(I+NNQXY+NNQZC)+MM)
    CONTINUE
    WRITE(NWRITE,1090)
    WRITE(NWRITE,2055)
    WRITE(NWRITE,1091)
177 CONTINUE
    DO 178 I=1,MMM
    RHS(I)=0.D0
    DO 178 J=1,MMM
    TM(I,J)=0.D0
178 CONTINUE
    DO 181 I=1,21
    DO 181 J=1,21
    TM$(I,J)=0.D0
181 CONTINUE

END OF INPUT AND VERIFICATION ROUTINE

DO 300 K=1,NE
N1=NODE(K,1)
N2=NODE(K,2)
N3=NODE(K,3)
N4=NODE(K,4)
N5=NODE(K,5)
N6=NODE(K,6)
N7=NODE(K,1)+NN
N8=NODE(K,2)+NN
N9=NODE(K,3)+NN
N10=NODE(K,4)+NN
N11=NODE(K,5)+NN
N12=NODE(K,6)+NN
N13=NCP(NODE(K,1))
N14=NCP(NODE(K,3))
N15=NCP(NODE(K,5))
N16=NODE(K,1)+MM
N17=NODE(K,2)+MM
N18=NODE(K,3)+MM
N19=NODE(K,4)+MM
N20=NODE(K,5)+MM
N21=NODE(K,6)+MM
XC$(1)=XC(NODE(K,1))
XC$(2)=XC(NODE(K,3))
XC$(3)=XC(NODE(K,5))
YC$(1)=YC(NODE(K,1))
YC$(2)=YC(NODE(K,3))
YC$(3)=YC(NODE(K,5))
A=1.D0
ZBAR=(YC$(1)+YC$(2)+YC$(3))/3.D0

```

```

FLSS2860
FLSS2870
FLSS8320
FLSS2880
FLSS2890
FLSS2900
FLSS2910
FLSS2920
FLSS2930
FLSS2940
FLSS2950
FLSS2960
FLSS2970
FLSS2980
FLSS2990
FLSS3000
FLSS3010
FLSS3020
FLSS3030
FLSS3040
FLSS3050
FLSS3060
FLSS3070
FLSS3080
FLSS3090
FLSS3100
FLSS3110
FLSS3120
FLSS3130
FLSS3140
FLSS3150
FLSS3160
FLSS3170
FLSS3180
FLSS3190
FLSS3200
FLSS3210
FLSS3220
FLSS3230
FLSS3240
FLSS3250
FLSS3260
FLSS3270
FLSS3280
FLSS3290
FLSS3300

```



```

RBAR=(XC$(1)+XC$(2)+XC$(3))/3.D0
IF(NC=SE.EQ.2)A=RBAR
AA=1.D0
IF(NC=SE.EQ.2)AA=2.D0*3.14159D0*(RBAR)
A1=XC$(2)*YC$(3)-XC$(3)*YC$(2)
A2=XC$(1)*YC$(3)-XC$(3)*YC$(1)
A3=XC$(1)*YC$(2)-XC$(2)*YC$(1)
B1=XC$(2)-YC$(3)
B2=XC$(1)-YC$(2)
B3=XC$(3)-XC$(1)
C1=XC$(1)-XC$(2)
C2=XC$(2)-XC$(3)
C3=XC$(3)-XC$(1)
DEL=DABS(0.5D0*(XC$(1)*YC$(2)-YC$(3)+XC$(2)*YC$(3)-YC$(1))
1+XC$(3)*YC$(1)-YC$(2))
CONST=10.90D0*(1.D0*A/(3.D0*DEL))*AA
D1=-B1/6.D0
D2=-B2/6.D0
D3=-B3/6.D0
E1=-C1/6.D0
E2=-C2/6.D0
E3=-C3/6.D0
F1=B1/2520.D0
F2=B2/2520.D0
F3=B3/2520.D0
G1=C1/2520.D0
G2=C2/2520.D0
G3=C3/2520.D0
U1=TI(N1)
U2=TI(N2)
U3=TI(N3)
U4=TI(N4)
U5=TI(N5)
U6=TI(N6)
V1=TI(N7)
V2=TI(N8)
V3=TI(N9)
V4=TI(N10)
V5=TI(N11)
V6=TI(N12)
TM$(1,1)=0.75D0*(B1*B1+C1*C1)*CONST
TM$(1,2)=(B1*B2+C1*C2)*CONST
TM$(1,3)=-TM$(1,2)*0.25D0
TM$(1,4)=0
TM$(1,6)=(B1*B3+C1*C3)*CONST
TM$(1,5)=-TM$(1,6)*0.25D0
TM$(1,3)=0.75D0*(B2*B2+C2*C2)*CONST
TM$(3,4)=(B2*B3+C2*C3)*CONST

```



```

TM$(3,5)=-TM$(3,4)*0.25D0
TM$(2,1)=TM$(1,2)
TM$(2,3)=TM$(1,2)
TM$(2,2)=8.D0/3.D0*(TM$(1,1)+TM$(3,3))+2.D0*TM$(1,2)
TM$(2,4)=2.D0*TM$(1,6)+TM$(3,4)+TM$(1,2)+4.D0/3.D0*TM$(3,3)
TM$(2,5)=0.
TM$(2,6)=TM$(1,6)+2.D0*TM$(3,4)+TM$(1,2)+4.D0/3.D0*TM$(1,1)
TM$(3,1)=TM$(1,3)
TM$(3,2)=TM$(2,3)
TM$(3,3)=0.
TM$(3,5)=75D0*(B3*B3+C3*C3)*CONST
TM$(4,1)=TM$(1,4)
TM$(4,2)=TM$(2,4)
TM$(4,3)=TM$(3,4)
TM$(4,4)=8.D0/3.D0*(TM$(3,3)+TM$(5,5))+2.D0*TM$(3,4)
TM$(4,5)=TM$(1,6)+TM$(3,4)+TM$(3,4)+TM$(3,4)+TM$(5,5)
TM$(4,6)=TM$(3,6)
TM$(5,1)=TM$(1,5)
TM$(5,2)=TM$(2,5)
TM$(5,3)=TM$(3,5)
TM$(5,4)=TM$(4,5)
TM$(5,5)=TM$(1,6)
TM$(5,6)=TM$(2,6)
TM$(6,1)=TM$(4,6)
TM$(6,2)=TM$(5,6)
TM$(6,3)=8.D0/3.D0*(TM$(5,5)+TM$(1,1))+2.D0*TM$(1,6)
TM$(6,6)=8.D0/3.D0*(TM$(5,5)+TM$(1,1))+2.D0*TM$(1,6)
IF(NC.ASE.NE.1) GO TO 3000

      BEGIN INPUT OF NON-LINEAR TERMS
      TM$(1,1)=TM$(1,1)
1- (78.D0*U1+48.D0*U2-9.D0*U3+12.D0*U4-9.D0*U5+48.D0*U6)*F1
2- (78.D0*V1+48.D0*V2-9.D0*V3+12.D0*V4-9.D0*V5+48.D0*V6)
3*G1
      TM$(2,1)=TM$(2,1)
1- (48.D0*U1+160.D0*U2-32.D0*U3+16.D0*U4-20.D0*U5+80.D0*U6)*F1
2- (48.D0*V1+160.D0*V2-32.D0*V3+16.D0*V4-20.D0*V5+80.D0*
3*V6)*G1
      TM$(3,1)=TM$(3,1)
1- (-9.D0*U1-32.D0*U2-18.D0*U3-16.D0*U4+11.D0*U5-20.D0*U6)*F1
2- (-9.D0*V1-32.D0*V2-18.D0*V3-16.D0*V4+11.D0*V5-20.D0*V6)*G1
      TM$(4,1)=TM$(4,1)
1- (12.D0*U1+16.D0*U2-16.D0*U3-96.D0*U4-16.D0*U5+16.D0*U6)*F1
2- (12.D0*V1+16.D0*V2-16.D0*V3-96.D0*V4-16.D0*V5+16.D0*V6)*G1

```

```

FLSSSS3790
FLSSSS3800
FLSSSS3810
FLSSSS3820
FLSSSS3830
FLSSSS3840
FLSSSS3850
FLSSSS3860
FLSSSS3870
FLSSSS3880
FLSSSS3890
FLSSSS3900
FLSSSS3910
FLSSSS3920
FLSSSS3930
FLSSSS3940
FLSSSS3950
FLSSSS3960
FLSSSS3970
FLSSSS3980
FLSSSS3990
FLSSSS4000
FLSSSS4010
FLSSSS4020
FLSSSS4030
FLSSSS4040
FLSSSS4050
FLSSSS4060
FLSSSS4070
FLSSSS4080
FLSSSS4090
FLSSSS4100
FLSSSS4110
FLSSSS4120
FLSSSS4130
FLSSSS4140
FLSSSS4150
FLSSSS4160
FLSSSS4170
FLSSSS4180
FLSSSS4190
FLSSSS4200
FLSSSS4210
FLSSSS4220
FLSSSS4230
FLSSSS4240
FLSSSS4250
FLSSSS4260

```





TM\$(5,1)=TM\$(5,1)  
 1-(-9.D0\*U1-20.D0\*U2+11.D0\*U3-16.D0\*U4-18.D0\*U5-32.D0\*U6)\*F1  
 2-(-9.D0\*V1-20.D0\*V2+11.D0\*V3-16.D0\*V4-18.D0\*V5-32.D0\*V6)\*G1  
 36)\*G1  
 TM\$(6,1)=TM\$(6,1)  
 1-(-48.D0\*U1+80.D0\*U2-20.D0\*U3+16.D0\*U4-32.D0\*U5+160.D0\*U6)\*F1  
 2-(-48.D0\*V1+80.D0\*V2-20.D0\*V3+16.D0\*V4-32.D0\*V5+160.D0\*V6)\*G1  
 36)\*G1  
 TM\$(1,2)=TM\$(1,2)  
 1-(-24.D0\*U1-32.D0\*U2-16.D0\*U3-48.D0\*U4+4.D0\*U5-16.D0\*U6)\*F1  
 2-(-24.D0\*V1-32.D0\*V2-16.D0\*V3-48.D0\*V4+4.D0\*V5-16.D0\*V6)\*G1  
 36)\*G1  
 TM\$(2,2)=TM\$(2,2)  
 1-(-32.D0\*U1+384.D0\*U2+48.D0\*U3+192.D0\*U4-48.D0\*U5+128.D0\*U6)\*F1  
 2-(-32.D0\*V1+384.D0\*V2+48.D0\*V3+192.D0\*V4-48.D0\*V5+128.D0\*V6)\*G1  
 36)\*G1  
 TM\$(3,2)=TM\$(3,2)  
 1-(-16.D0\*U1+48.D0\*U2+120.D0\*U3+48.D0\*U4-16.D0\*U5-16.D0\*U6)\*F1  
 2-(-16.D0\*V1+48.D0\*V2+120.D0\*V3+48.D0\*V4-16.D0\*V5-16.D0\*V6)\*G1  
 36)\*G1  
 TM\$(4,2)=TM\$(4,2)  
 1-(-48.D0\*U1+192.D0\*U2+48.D0\*U3+384.D0\*U4-32.D0\*U5+128.D0\*U6)\*F1  
 2-(-48.D0\*V1+192.D0\*V2+48.D0\*V3+384.D0\*V4-32.D0\*V5+128.D0\*V6)\*G1  
 36)\*G1  
 TM\$(5,2)=TM\$(5,2)  
 1-(-4.D0\*U1-48.D0\*U2-16.D0\*U3-32.D0\*U4+24.D0\*U5-16.D0\*U6)\*F1  
 2-(-4.D0\*V1-48.D0\*V2-16.D0\*V3-32.D0\*V4+24.D0\*V5-16.D0\*V6)\*G1  
 36)\*G1  
 TM\$(6,2)=TM\$(6,2)  
 1-(-16.D0\*U1+128.D0\*U2-16.D0\*U3+128.D0\*U4-16.D0\*U5+128.D0\*U6)\*F1  
 2-(-16.D0\*V1+128.D0\*V2-16.D0\*V3+128.D0\*V4-16.D0\*V5+128.D0\*V6)\*G1  
 36)\*G1  
 TM\$(1,3)=TM\$(1,3)  
 1-(-18.D0\*U1-32.D0\*U2-9.D0\*U3-20.D0\*U4+11.D0\*U5-16.D0\*U6)\*F2  
 2-(-18.D0\*V1-32.D0\*V2-9.D0\*V3-20.D0\*V4+11.D0\*V5-16.D0\*V6)\*G2  
 36)\*G2





TM\$(2,3)=TM\$(2,3)  
 1-(-32.D0\*U1+160.D0\*U2+48.D0\*U3+80.D0\*U4-20.D0\*U5+16.D0\*U6)\*F2  
 2-(-32.D0\*V1+160.D0\*V2+48.D0\*V3+80.D0\*V4-20.D0\*V5+16.D0\*V6)\*G2  
 3\*V6)\*G2  
 TM\$(3,3)=TM\$(3,3)  
 1-(-9.D0\*U1+48.D0\*U2+78.D0\*U3+48.D0\*U4-9.D0\*U5+12.D0\*U6)\*F2  
 2-(-9.D0\*V1+48.D0\*V2+78.D0\*V3+48.D0\*V4-9.D0\*V5+12.D0\*V6)\*G2  
 3\*G2  
 TM\$(4,3)=TM\$(4,3)  
 1-(-20.D0\*U1+80.D0\*U2+48.D0\*U3+160.D0\*U4-32.D0\*U5+16.D0\*U6)\*F2  
 2-(-20.D0\*V1+80.D0\*V2+48.D0\*V3+160.D0\*V4-32.D0\*V5+16.D0\*V6)\*G2  
 3\*V6)\*G2  
 TM\$(5,3)=TM\$(5,3)  
 1-(-11.D0\*U1-20.D0\*U2-9.D0\*U3-32.D0\*U4-18.D0\*U5-16.D0\*U6)\*F2  
 2-(-11.D0\*V1-20.D0\*V2-9.D0\*V3-32.D0\*V4-18.D0\*V5-16.D0\*V6)\*G2  
 3\*G2  
 TM\$(6,3)=TM\$(6,3)  
 1-(-16.D0\*U1+16.D0\*U2+12.D0\*U3+16.D0\*U4-16.D0\*U5-96.D0\*U6)\*F2  
 2-(-16.D0\*V1+16.D0\*V2+12.D0\*V3+16.D0\*V4-16.D0\*V5-96.D0\*V6)\*G2  
 3\*V6)\*G2  
 TM\$(1,4)=TM\$(1,4)  
 1-(-24.D0\*U1-16.D0\*U2+4.D0\*U3-48.D0\*U4-16.D0\*U5-32.D0\*U6)\*F2  
 2-(-24.D0\*V1-16.D0\*V2+4.D0\*V3-48.D0\*V4-16.D0\*V5-32.D0\*V6)\*G2  
 3\*G2  
 TM\$(2,4)=TM\$(2,4)  
 1-(-16.D0\*U1+128.D0\*U2-16.D0\*U3+128.D0\*U4-16.D0\*U5-32.D0\*U6)\*F2  
 2-(-16.D0\*V1+128.D0\*V2-16.D0\*V3+128.D0\*V4-16.D0\*V5-32.D0\*V6)\*G2  
 3\*G2  
 TM\$(3,4)=TM\$(3,4)  
 1-(-4.D0\*U1-16.D0\*U2+24.D0\*U3-32.D0\*U4+16.D0\*U5-48.D0\*U6)\*F2  
 2-(-4.D0\*V1-16.D0\*V2+24.D0\*V3-32.D0\*V4+16.D0\*V5-48.D0\*V6)\*G2  
 3\*G2  
 TM\$(4,4)=TM\$(4,4)  
 1-(-48.D0\*U1+128.D0\*U2-32.D0\*U3+384.D0\*U4+48.D0\*U5+192.D0\*U6)\*F2  
 2-(-48.D0\*V1+128.D0\*V2-32.D0\*V3+384.D0\*V4+48.D0\*V5+192.D0\*V6)\*G2  
 3\*G2  
 TM\$(5,4)=TM\$(5,4)  
 1-(-16.D0\*U1-16.D0\*U2-16.D0\*U3+48.D0\*U4+120.D0\*U5+48.D0\*U6)\*F2  
 2-(-16.D0\*V1-16.D0\*V2-16.D0\*V3+48.D0\*V4+120.D0\*V5+48.D0\*V6)\*G2  
 3\*V6)\*G2

FLSS4750  
 FLSS4760  
 FLSS4770  
 FLSS4780  
 FLSS4790  
 FLSS4800  
 FLSS4810  
 FLSS4820  
 FLSS4830  
 FLSS4840  
 FLSS4850  
 FLSS4860  
 FLSS4870  
 FLSS4880  
 FLSS4890  
 FLSS4900  
 FLSS4910  
 FLSS4920  
 FLSS4930  
 FLSS4940  
 FLSS4950  
 FLSS4960  
 FLSS4970  
 FLSS4980  
 FLSS4990  
 FLSS5000  
 FLSS5010  
 FLSS5020  
 FLSS5030  
 FLSS5040  
 FLSS5050  
 FLSS5060  
 FLSS5070  
 FLSS5080  
 FLSS5090  
 FLSS5100  
 FLSS5110  
 FLSS5120  
 FLSS5130  
 FLSS5140  
 FLSS5150  
 FLSS5160  
 FLSS5170  
 FLSS5180  
 FLSS5190  
 FLSS5200  
 FLSS5210  
 FLSS5220



4\*U5-16.00\*U6)\*F3  
5.00\*V4+24.00\*V5-16.00\*V6)\*G3  
TM\$(6,4)=TM\$(6,4)  
1-(-32.00\*U1+128.00\*U2-48.00\*U3+192.00\*U4+48.00\*U5+384.00\*U6)\*F2  
2-(-32.00\*U1+128.00\*U2-48.00\*U3+192.00\*U4+48.00\*U5+384.00\*U6)\*F2  
300\*V6)\*G2  
46.00\*U5+128.00\*U6)\*F3  
5+128.00\*V4-16.00\*V5+128.00\*V6)\*G3  
TM\$(1,5)=TM\$(1,5)  
1-(-18.00\*U1-16.00\*U2+11.00\*U3-20.00\*U4-9.00\*U5-32.00\*U6)\*F3  
2-(-18.00\*U1-16.00\*U2+11.00\*U3-20.00\*U4-9.00\*U5-32.00\*U6)\*F3  
36)\*G3  
TM\$(2,5)=TM\$(2,5)  
1-(-16.00\*U1-96.00\*U2-16.00\*U3+16.00\*U4+12.00\*U5+16.00\*U6)\*F3  
2-(-16.00\*U1-96.00\*U2-16.00\*U3+16.00\*U4+12.00\*U5+16.00\*U6)\*F3  
3V6)\*G3  
TM\$(3,5)=TM\$(3,5)  
1-(-11.00\*U1-16.00\*U2-18.00\*U3-32.00\*U4-9.00\*U5-20.00\*U6)\*F3  
2-(-11.00\*U1-16.00\*U2-18.00\*U3-32.00\*U4-9.00\*U5-20.00\*U6)\*F3  
3)\*G3  
TM\$(4,5)=TM\$(4,5)  
1-(-20.00\*U1+16.00\*U2-32.00\*U3+160.00\*U4+48.00\*U5+80.00\*U6)\*F3  
2-(-20.00\*U1+16.00\*U2-32.00\*U3+160.00\*U4+48.00\*U5+80.00\*U6)\*F3  
3\*V6)\*G3  
TM\$(5,5)=TM\$(5,5)  
1-(-9.00\*U1+12.00\*U2-9.00\*U3+48.00\*U4+78.00\*U5+48.00\*U6)\*F3  
2-(-9.00\*U1+12.00\*U2-9.00\*U3+48.00\*U4+78.00\*U5+48.00\*U6)\*F3  
3)\*G3  
TM\$(6,5)=TM\$(6,5)  
1-(-32.00\*U1+16.00\*U2-20.00\*U3+80.00\*U4+48.00\*U5+160.00\*U6)\*F3  
2-(-32.00\*U1+16.00\*U2-20.00\*U3+80.00\*U4+48.00\*U5+160.00\*U6)\*F3  
3\*V6)\*G3  
TM\$(1,6)=TM\$(1,6)  
1-(-24.00\*U1-16.00\*U2+4.00\*U3-48.00\*U4-16.00\*U5-32.00\*U6)\*F1  
2-(-24.00\*U1-16.00\*U2+4.00\*U3-48.00\*U4-16.00\*U5-32.00\*U6)\*F1  
3)\*G1  
40\*U5+48.00\*U6)\*F3  
516.00\*V4-16.00\*V5+48.00\*V6)\*G3  
TM\$(2,6)=TM\$(2,6)  
1-(-16.00\*U1+128.00\*U2-16.00\*U3+128.00\*U4-16.00\*U5+128.00\*U6)\*F1  
2-(-16.00\*U1+128.00\*U2-16.00\*U3+128.00\*U4-16.00\*U5+128.00\*U6)\*F1  
300\*V6)\*G1  
4.00\*U5+192.00\*U6)\*F3  
5128.00\*V4-48.00\*V5+192.00\*V6)\*G3  
TM\$(3,6)=TM\$(3,6)  
1-(-4.00\*U1-16.00\*U2+24.00\*U3-32.00\*U4-16.00\*U5-48.00\*U6)\*F1  
2-(-4.00\*U1-16.00\*U2+24.00\*U3-32.00\*U4-16.00\*U5-48.00\*U6)\*F1  
3)\*G1





```

40*U5-48.D0*U6)*F3
516.D0*V4+4.D0*V5-48.D0*V6)*G3
TM$(4,6)=TM$(4,6)
1-(-48.D0*U1+128.D0*U2-32.D0*U3+384.D0*U4+48.D0*U5+192.D0*U6)*F1
2-(-48.D0*U1+128.D0*V1+128.D0*V2-32.D0*V3+384.D0*V4+48.D0*V5+192.D0*V6)*G1
3D0*V6)*G1
46.D0*U5+128.D0*U6)*F3
5+128.D0*V4-16.D0*V5+128.D0*V6)*G3
TM$(5,6)=TM$(5,6)
1-(-16.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*F1
2-(-16.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*G1
3*V6)*G1
40*U5-32.D0*U6)*F3
56.D0*V4+24.D0*V5-32.D0*V6)*G3
TM$(6,6)=TM$(6,6)
1-(-32.D0*U1+128.D0*U2-48.D0*U3+192.D0*U4+48.D0*U5+384.D0*U6)*F1
2-(-32.D0*U1+128.D0*V1+128.D0*V2-48.D0*V3+384.D0*V4+48.D0*V5+384.D0*V6)*G1
3D0*V6)*G1
4.D0*U5+384.D0*U6)*F3
5128.D0*V4-32.D0*V5+384.D0*V6)*G3
-(-16.D0*V1-32.D0*V2+24.D0*V3-
FLSS55710
FLSS55720
FLSS55730
FLSS55740
FLSS55750
FLSS55760
FLSS55770
FLSS55780
FLSS55790
FLSS55800
FLSS55810
FLSS55820
FLSS55830
FLSS55840
FLSS55850
FLSS55860
FLSS55870
FLSS55880
FLSS55890
FLSS55900
FLSS55910
FLSS55920
FLSS55930
FLSS55940
FLSS55950
FLSS55960
FLSS55970
FLSS55980
FLSS55990
FLSS6000
FLSS6010
FLSS6020
FLSS6030
FLSS6040
FLSS6050
FLSS6060
FLSS6070
FLSS6080
FLSS6090
FLSS6100
FLSS6110
FLSS6120
FLSS6130
FLSS6140
FLSS6150
FLSS6160
FLSS6170
FLSS6180

```

THIS ENCS ADDITION OF NON-LINEAR TERMS TO THE LOCAL ARRAY

3000

```

CONTINUE
TM$(7,7)=TM$(1,1)
TM$(7,8)=TM$(1,2)
TM$(7,9)=TM$(1,3)
TM$(7,10)=TM$(1,4)
TM$(7,11)=TM$(1,5)
TM$(7,12)=TM$(1,6)
TM$(8,7)=TM$(2,1)
TM$(8,8)=TM$(2,2)
TM$(8,9)=TM$(2,3)
TM$(8,10)=TM$(2,4)
TM$(8,11)=TM$(2,5)
TM$(8,12)=TM$(2,6)
TM$(9,7)=TM$(3,1)
TM$(9,8)=TM$(3,2)
TM$(9,9)=TM$(3,3)
TM$(9,10)=TM$(3,4)
TM$(9,11)=TM$(3,5)
TM$(9,12)=TM$(3,6)
TM$(10,7)=TM$(4,1)
TM$(10,8)=TM$(4,2)
TM$(10,9)=TM$(4,3)
TM$(10,10)=TM$(4,4)
TM$(10,11)=TM$(4,5)
TM$(10,12)=TM$(4,6)

```





TMM\$(1,1)	TMM\$(5,1)	7)=TMM\$(5,1)	
TMM\$(1,1)	TMM\$(5,2)	7)=TMM\$(5,2)	
TMM\$(1,1)	TMM\$(5,3)	7)=TMM\$(5,3)	
TMM\$(1,1)	TMM\$(5,4)	7)=TMM\$(5,4)	
TMM\$(1,1)	TMM\$(5,5)	7)=TMM\$(5,5)	
TMM\$(1,2)	TMM\$(6,1)	7)=TMM\$(6,1)	
TMM\$(1,2)	TMM\$(6,2)	7)=TMM\$(6,2)	
TMM\$(1,2)	TMM\$(6,3)	7)=TMM\$(6,3)	
TMM\$(1,2)	TMM\$(6,4)	7)=TMM\$(6,4)	
TMM\$(1,2)	TMM\$(6,5)	7)=TMM\$(6,5)	
TMM\$(1,3)	TMM\$(6,6)	7)=TMM\$(6,6)	
TMM\$(1,4)	TMM\$(7,1)	7)=TMM\$(7,1)	
TMM\$(1,5)	TMM\$(7,2)	7)=TMM\$(7,2)	
TMM\$(1,5)	TMM\$(7,3)	7)=TMM\$(7,3)	
TMM\$(1,5)	TMM\$(7,4)	7)=TMM\$(7,4)	
TMM\$(1,5)	TMM\$(7,5)	7)=TMM\$(7,5)	
TMM\$(1,5)	TMM\$(7,6)	7)=TMM\$(7,6)	
TMM\$(1,5)	TMM\$(7,7)	7)=TMM\$(7,7)	
TMM\$(1,5)	TMM\$(7,8)	7)=TMM\$(7,8)	
TMM\$(1,5)	TMM\$(7,9)	7)=TMM\$(7,9)	
TMM\$(1,5)	TMM\$(7,10)	7)=TMM\$(7,10)	
TMM\$(1,5)	TMM\$(7,11)	7)=TMM\$(7,11)	
TMM\$(1,5)	TMM\$(7,12)	7)=TMM\$(7,12)	
TMM\$(1,5)	TMM\$(7,13)	7)=TMM\$(7,13)	
TMM\$(1,5)	TMM\$(7,14)	7)=TMM\$(7,14)	
TMM\$(1,5)	TMM\$(7,15)	7)=TMM\$(7,15)	
TMM\$(1,5)	TMM\$(7,16)	7)=TMM\$(7,16)	
TMM\$(1,5)	TMM\$(7,17)	7)=TMM\$(7,17)	
TMM\$(1,5)	TMM\$(7,18)	7)=TMM\$(7,18)	
TMM\$(1,5)	TMM\$(7,19)	7)=TMM\$(7,19)	
TMM\$(1,5)	TMM\$(7,20)	7)=TMM\$(7,20)	
TMM\$(1,5)	TMM\$(7,21)	7)=TMM\$(7,21)	
TMM\$(1,5)	TMM\$(7,22)	7)=TMM\$(7,22)	
TMM\$(1,5)	TMM\$(7,23)	7)=TMM\$(7,23)	
TMM\$(1,5)	TMM\$(7,24)	7)=TMM\$(7,24)	
TMM\$(1,5)	TMM\$(7,25)	7)=TMM\$(7,25)	
TMM\$(1,5)	TMM\$(7,26)	7)=TMM\$(7,26)	
TMM\$(1,5)	TMM\$(7,27)	7)=TMM\$(7,27)	
TMM\$(1,5)	TMM\$(7,28)	7)=TMM\$(7,28)	
TMM\$(1,5)	TMM\$(7,29)	7)=TMM\$(7,29)	
TMM\$(1,5)	TMM\$(7,30)	7)=TMM\$(7,30)	
TMM\$(1,5)	TMM\$(7,31)	7)=TMM\$(7,31)	
TMM\$(1,5)	TMM\$(7,32)	7)=TMM\$(7,32)	
TMM\$(1,5)	TMM\$(7,33)	7)=TMM\$(7,33)	
TMM\$(1,5)	TMM\$(7,34)	7)=TMM\$(7,34)	
TMM\$(1,5)	TMM\$(7,35)	7)=TMM\$(7,35)	
TMM\$(1,5)	TMM\$(7,36)	7)=TMM\$(7,36)	
TMM\$(1,5)	TMM\$(7,37)	7)=TMM\$(7,37)	
TMM\$(1,5)	TMM\$(7,38)	7)=TMM\$(7,38)	
TMM\$(1,5)	TMM\$(7,39)	7)=TMM\$(7,39)	
TMM\$(1,5)	TMM\$(7,40)	7)=TMM\$(7,40)	
TMM\$(1,5)	TMM\$(7,41)	7)=TMM\$(7,41)	
TMM\$(1,5)	TMM\$(7,42)	7)=TMM\$(7,42)	
TMM\$(1,5)	TMM\$(7,43)	7)=TMM\$(7,43)	
TMM\$(1,5)	TMM\$(7,44)	7)=TMM\$(7,44)	
TMM\$(1,5)	TMM\$(7,45)	7)=TMM\$(7,45)	
TMM\$(1,5)	TMM\$(7,46)	7)=TMM\$(7,46)	
TMM\$(1,5)	TMM\$(7,47)	7)=TMM\$(7,47)	
TMM\$(1,5)	TMM\$(7,48)	7)=TMM\$(7,48)	
TMM\$(1,5)	TMM\$(7,49)	7)=TMM\$(7,49)	
TMM\$(1,5)	TMM\$(7,50)	7)=TMM\$(7,50)	
TMM\$(1,5)	TMM\$(7,51)	7)=TMM\$(7,51)	
TMM\$(1,5)	TMM\$(7,52)	7)=TMM\$(7,52)	
TMM\$(1,5)	TMM\$(7,53)	7)=TMM\$(7,53)	
TMM\$(1,5)	TMM\$(7,54)	7)=TMM\$(7,54)	
TMM\$(1,5)	TMM\$(7,55)	7)=TMM\$(7,55)	
TMM\$(1,5)	TMM\$(7,56)	7)=TMM\$(7,56)	
TMM\$(1,5)	TMM\$(7,57)	7)=TMM\$(7,57)	
TMM\$(1,5)	TMM\$(7,58)	7)=TMM\$(7,58)	
TMM\$(1,5)	TMM\$(7,59)	7)=TMM\$(7,59)	
TMM\$(1,5)	TMM\$(7,60)	7)=TMM\$(7,60)	
TMM\$(1,5)	TMM\$(7,61)	7)=TMM\$(7,61)	
TMM\$(1,5)	TMM\$(7,62)	7)=TMM\$(7,62)	
TMM\$(1,5)	TMM\$(7,63)	7)=TMM\$(7,63)	
TMM\$(1,5)	TMM\$(7,64)	7)=TMM\$(7,64)	
TMM\$(1,5)	TMM\$(7,65)	7)=TMM\$(7,65)	
TMM\$(1,5)	TMM\$(7,66)	7)=TMM\$(7,66)	
TMM\$(1,5)	TMM\$(7,67)	7)=TMM\$(7,67)	
TMM\$(1,5)	TMM\$(7,68)	7)=TMM\$(7,68)	
TMM\$(1,5)	TMM\$(7,69)	7)=TMM\$(7,69)	
TMM\$(1,5)	TMM\$(7,70)	7)=TMM\$(7,70)	
TMM\$(1,5)	TMM\$(7,71)	7)=TMM\$(7,71)	
TMM\$(1,5)	TMM\$(7,72)	7)=TMM\$(7,72)	
TMM\$(1,5)	TMM\$(7,73)	7)=TMM\$(7,73)	
TMM\$(1,5)	TMM\$(7,74)	7)=TMM\$(7,74)	
TMM\$(1,5)	TMM\$(7,75)	7)=TMM\$(7,75)	
TMM\$(1,5)	TMM\$(7,76)	7)=TMM\$(7,76)	
TMM\$(1,5)	TMM\$(7,77)	7)=TMM\$(7,77)	
TMM\$(1,5)	TMM\$(7,78)	7)=TMM\$(7,78)	
TMM\$(1,5)	TMM\$(7,79)	7)=TMM\$(7,79)	
TMM\$(1,5)	TMM\$(7,80)	7)=TMM\$(7,80)	
TMM\$(1,5)	TMM\$(7,81)	7)=TMM\$(7,81)	
TMM\$(1,5)	TMM\$(7,82)	7)=TMM\$(7,82)	
TMM\$(1,5)	TMM\$(7,83)	7)=TMM\$(7,83)	



```

CONST4=0.94375D0
CMTM$(1,1,1,3)=D1*CONST4
CMTM$(1,1,1,4)=0.D0
CMTM$(1,1,1,5)=0.D0
CMTM$(2,1,1,3)=(D1+2.D0*D2)*CONST4
CMTM$(2,1,1,4)=(2.D0*D1+D2)*CONST4
CMTM$(2,1,1,5)=(D1+D2)*CONST4
CMTM$(3,1,1,3)=0.D0
CMTM$(3,1,1,4)=D2*CONST4
CMTM$(3,1,1,5)=0.D0
CMTM$(4,1,1,3)=(D2+D3)*CONST4
CMTM$(4,1,1,4)=(D2+2.D0*D3)*CONST4
CMTM$(4,1,1,5)=(2.D0*D2+D3)*CONST4
CMTM$(5,1,1,3)=0.D0
CMTM$(5,1,1,4)=0.D0
CMTM$(5,1,1,5)=D3*CONST4
CMTM$(6,1,1,3)=(D1+2.D0*D3)*CONST4
CMTM$(6,1,1,4)=(D1+D3)*CONST4
CMTM$(6,1,1,5)=(2.D0*D1+D3)*CONST4
CMTM$(7,1,1,3)=E1*CONST4
CMTM$(7,1,1,4)=0.D0
CMTM$(7,1,1,5)=0.D0
CMTM$(8,1,1,3)=(E1+2.D0*E2)*CONST4
CMTM$(8,1,1,4)=(E1+D0*E1+E2)*CONST4
CMTM$(8,1,1,5)=(E1+E2)*CONST4
CMTM$(9,1,1,3)=0.D0
CMTM$(9,1,1,4)=E2*CONST4
CMTM$(9,1,1,5)=0.D0
CMTM$(10,1,1,3)=(E2+E3)*CONST4
CMTM$(10,1,1,4)=(E2+2.D0*E3)*CONST4
CMTM$(10,1,1,5)=(E2+E3)*CONST4
CMTM$(11,1,1,3)=0.D0
CMTM$(11,1,1,4)=0.D0
CMTM$(11,1,1,5)=E3*CONST4
CMTM$(12,1,1,3)=(E1+2.D0*E3)*CONST4
CMTM$(12,1,1,4)=(E1+E3)*CONST4
CMTM$(12,1,1,5)=(2.D0*E1+E3)*CONST4
CMTM$(12,1,1,6)=3.2981D0
CMTM$(1,1,1,6)=1.D0*CONST2
CMTM$(2,1,1,7)=1.D0*CONST2
CMTM$(3,1,1,8)=1.D0*CONST2
CMTM$(4,1,1,9)=1.D0*CONST2
CMTM$(5,1,1,20)=1.D0*CONST2
CMTM$(6,1,1,21)=1.D0*CONST2

```

ALPHA1= ALPHA/ VISCOSITY

ALPHA1=9.4454D-05

```

FLSS6670
FLSS6680
FLSS6690
FLSS6700
FLSS6710
FLSS6720
FLSS6730
FLSS6740
FLSS6750
FLSS6760
FLSS6770
FLSS6780
FLSS6790
FLSS6800
FLSS6810
FLSS6820
FLSS6830
FLSS6840
FLSS6850
FLSS6860
FLSS6870
FLSS6880
FLSS6890
FLSS6900
FLSS6910
FLSS6920
FLSS6930
FLSS6940
FLSS6950
FLSS6960
FLSS6970
FLSS6980
FLSS6990
FLSS7000
FLSS7010
FLSS7020
FLSS7030
FLSS7040
FLSS7050
FLSS7060
FLSS7070
FLSS7080
FLSS7090
FLSS7100
FLSS7110
FLSS7120
FLSS7130
FLSS7140

```









```

N(11)=N11
N(12)=N12
N(13)=N13
N(14)=N14
N(15)=N15
N(16)=N16
N(17)=N17
N(18)=N18
N(19)=N19
N(20)=N20
N(21)=N21
DO 200 I$=1,21
I=N(I$)
DO 200 J$=1,21
J=N(J$)
TM(I,J)=TM(I,J)+TM$(I$,J$)
200 CONTINUE
300 IF(NNQXY.EQ.0) GO TO 310
DO 310 I=1,NNQXY
RHS(NQS(I))=RHS(NQS(I))+Q(NQS(I))+65.962D0
RHS(NQS(I)+NN)=RHS(NQS(I)+NN)+Q(NQS(I)+NN)
310 CONTINUE
310 IF(NNQZC.EQ.0) GO TO 312
DO 312 I=1,NNQZC
RHS(NQS(NNQXY+I)+2*NN)=RHS(NQS(NNQXY+I))+2*NN)+Q(NQS(NNQXY+I))+2*NN)
312 CONTINUE
312 IF(NNQZ.EQ.0) GO TO 311
DO 311 I=1,NNQZ
RHS(NQS(NNQXY+NNQZC+I)+MM)=RHS(NQS(NNQXY+NNQZC+I)+MM)+
1Q(NQS(NNQXY+NNQZC+I)+MM)
311 CONTINUE

MODIFICATION OF RHS FOR TM BOUNDARY CONDITIONS

DO 315 I=1,MMM
DO 315 J=1,NTOTVP
JX=NVIS(J)
RHS(I)=RHS(I)-TM(I,JX)*X(JX)
TM(I,JX)=0.D0
TM(JX,I)=0.D0
TM(JX,JX)=1.D0
RHS(JX)=X(JX)
315 CONTINUE
M=1
ND=117
IA=117
IDGT=0

```

```

FLSSS7630
FLSSS7640
FLSSS7650
FLSSS7660
FLSSS7670
FLSSS7680
FLSSS7690
FLSSS7700
FLSSS7710
FLSSS7720
FLSSS7730
FLSSS7740
FLSSS7750
FLSSS7760
FLSSS7770
FLSSS7780
FLSSS7790
FLSSS7800
FLSSS7810
FLSSS7820
FLSSS7830
FLSSS7840
FLSSS7850
FLSSS7860
FLSSS7870
FLSSS7880
FLSSS7890
FLSSS7900
FLSSS7910
FLSSS7920
FLSSS7930
FLSSS7940
FLSSS7950
FLSSS7960
FLSSS7970
FLSSS7980
FLSSS7990
FLSSS8000
FLSSS8010
FLSSS8020
FLSSS8030
FLSSS8040
FLSSS8050
FLSSS8060
FLSSS8070
FLSSS8080
FLSSS8090
FLSSS8100

```





```

CALL LEQT2F (TM,M,ND,IA,RHS,IDGT,WKAREA,IER)
WRITE(NWRITE,2000)
DO 322 J=1,MMM
TDIFF=DABS(T1(J)-X(J))
EPSLN=1.D-06
IF(TDIFF-EPSLN) 322,324,324
322 CONTINUE
324 CONTINUE
DO 360 I=1,MMM
X(I)=RHS(I)
T1(I)=X(I)
WRITE (NWRITE,2005) I,X(I)
360 CONTINUE
IF(J.NE.MMM) GO TO 177
WRITE(NWRITE,2021)
WRITE(NWRITE,2025)
WRITE(NWRITE,2030)
WRITE(NWRITE,2035)
WRITE(NWRITE,2040)
WRITE(NWRITE,2045)
WRITE(NWRITE,2050)
500 FORMAT(I10)
600 FORMAT(I10,18X,'STEADY STATE FLUID MECHANICS PROBLEM',////)
1005 FORMAT(3I10)
1006 FORMAT(6X,A4,A4,I10,2F10.0)
1010 FORMAT(7I10)
1015 FORMAT(6X,A4,A4,I10,F10.0)
1016 FORMAT(6X,A4,I10)
1020 FORMAT(I10,2F10.0)
1025 FORMAT(6X,A4,I10,F10.0)
1030 FORMAT(6X,A4,2I10,2F10.0)
1034 FORMAT(5X,'NNQZC=',I3,/)
1035 FORMAT(5X,'NO. OF NODES=',I3,/)
1036 FORMAT(5X,'NO. OF CORNERS=',I3,/)
1037 FORMAT(5X,'NNVELS=',I3,/)
1038 FORMAT(5X,'NNQXY=',I3,/)
1039 FORMAT(5X,'NNPS=',I3,/)
1040 FORMAT(5X,'NNTS=',I3,/)
1041 FORMAT(5X,'NNQZ=',I3,/)
1045 FORMAT(5X,'SUMMARY OF NODAL COORDINATES',/,/,
17X,I,12X,X(I),13X,Y(I),/,/)
1050 FORMAT(5X,I3,2(7X,F10.3))
1055 FORMAT(5X,I3,10X,6(5X,I3))
1060 FORMAT(5X,I3,7X,'NODES WHERE VELOCITIES ARE SPECIFIED',/,/,
17X,I,5X,'NODE',5X,'U VELOCITY',5X,'V VELOCITY',/,/)
1065 FORMAT(2X,2(4X,I3),3X,F12.3,3X,F12.3)

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FLSS8110
FLSS8120
FLSS8130
FLSS8140
FLSS8150
FLSS8160
FLSS8170
FLSS8180
FLSS8190
FLSS8200
FLSS8210
FLSS8220
FLSS8230
FLSS8240
FLSS8250
FLSS8260
FLSS8270
FLSS8280
FLSS8290
FLSS8300
FLSS8310
FLSS8330
FLSS8340
FLSS8350
FLSS8360
FLSS8370
FLSS8380
FLSS8390
FLSS8400
FLSS8410
FLSS8420
FLSS8430
FLSS8440
FLSS8450
FLSS8460
FLSS8470
FLSS8480
FLSS8490
FLSS8500
FLSS8510
FLSS8520
FLSS8540
FLSS8550
FLSS8560
FLSS8570
FLSS8580

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```

1070 FORMAT(//,5X,'NODES WHERE QX AND QY ARE SPECIFIED',
1071         ,2X,'NODE',1X,'QX',10X,'QY',//)
1075 FORMAT(5X,I3,2(10X,F12.3))
1080 FORMAT(//,5X,'NODES WHERE PRESSURE IS SPECIFIED',
1081         ,9X,I',4X,NODE',13X,'PRESSURE',//)
1081 FORMAT(//,5X,'NODES WHERE TEMPERATURE IS SPECIFIED',
1082         ,9X,I',4X,NODE',12X,'TEMPERATURE',//)
1082 FORMAT(//,5X,'NODES WHERE HEAT FLUX QZ IS SPECIFIED',
1083         ,9X,I',4X,NODE',12X,'HEAT FLUX',//)
1083 FORMAT(//,5X,'NODES WHERE QZC IS SPECIFIED',
1084         ,9X,I',4X,NODE',15X,'QZC',//)
1085 FORMAT(7X,I3,3X,I3,10X,F12.3)
1090 FORMAT(//,5X,'NODAL VARIABLE IS THE U-VELOCITY AT NODES 1 - 35',
1091         ,5X,'THE V-VELOCITY AT NODES 36 - 70',
1092         ,5X,'AND THE PRESSURE AT NODES 71 - 82',
1093         ,5X,'THE TEMPERATURE AT NODES 83 - 117',//)
1091 FORMAT(//,5X,'THE FIRST SEQUENCE OF 117 NODAL VARIABLES',
1092         ,5X,'REPRESENTS A LINEAR, STEADY STATE SYSTEM',
1093         ,5X,'WHILE THE SECOND SET OF THE 117 VALUES CORRESPONDS',
1094         ,5X,'TO A NONLINEAR ANALYSIS OF THE SOLUTION DOMAIN',//)
1095 FORMAT(5X,3(3X,I3),2(5X,F12.3))
2000 FORMAT(//,5X,'NODE NO.',6X,'NODE VARIABLE',//)
2005 FORMAT(9X,I3,5X,D17.10,/)
2015 FORMAT(1H1,21X,'THIS IS A 3-D AXISYMMETRIC PROBLEM',//)
2020 FORMAT(21X,'THIS IS A 2-D NONLINEAR PROBLEM',//)
2021 FORMAT(//,5X,10X,'THE KINEMATIC VISCOSITY OF FLUID 50-HB-3520 AT 20
1DEGREES C.',//)
2025 FORMAT(//,5X,10X,'THE DENSITY OF 50-HB-3520 AT 20 DEGREES C. = 1.059',
2026         ,10X,'SQ.CM/SEC',//)
2030 FORMAT(//,5X,10X,'THE COEFF. OF THERMAL EXPANSION OF 50-HB-3520 AT 20
16 GM/CC',//)
2035 FORMAT(//,5X,10X,'THE THERMAL DIFFUSIVITY OF 50-HB-3520 AT 20 DEGREE',
2036         ,10X,'DEGREES C.',//)
2040 FORMAT(//,5X,10X,'THE GRASHOF NUMBER (GR(L)) = (G*B*L**3*(TH-TC))/V**',
2041         ,10X,'SQ.CM/SEC',//)
2045 FORMAT(//,5X,10X,'THE U VELOCITY FORCING FUNCTION, G*B*T(INITIAL), =
12 = 946.4',//)
2050 FORMAT(//,5X,10X,'THE SPECIFIED WALL PRESSURES',
2051         ,165.962 CM/SQ.SEC',//)
2055 FORMAT(//,5X,10X,'THE SPECIFIED WALL PRESSURES',
2056         ,1X,'5X,'ARE NORMALIZED TO ONE (1) ATMOSPHERE',
2057         ,3X,'5X,'THAT IS, 1014000 DYNES/SQ.CM',
2058         ,3X,'5X,'(ALL PARAMETER VALUES ARE IN CGS UNITS)',//)
2059 STOP
2060 END

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FLSS8590  
FLSS8600  
FLSS8610  
FLSS8620  
FLSS8630  
FLSS8640  
FLSS8650  
FLSS8660  
FLSS8670  
FLSS8680  
FLSS8690  
FLSS8700

FLSS8710  
FLSS8720  
FLSS8730  
FLSS8740  
FLSS8750  
FLSS8760  
FLSS8770

FLSS8790  
FLSS8800  
FLSS8810

FLSS8830

FLSS8850  
FLSS8860  
FLSS8870  
FLSS8880

FLSS8910  
FLSS8920



# TIME-DEPENDENT FLUID MECHANICS PROBLEM

```

IMPLICIT REAL*8(A-H,O-Z,$)
DATA NREAD/5/

THE U VELOCITY IS IN THE FIRST NN POSITIONS OF X(I)
THE V VELOCITY IS IN THE SECOND NN POSITIONS OF X I.E. X(I+NN)
THE P PRESSURE IS IN THE NN+NN+I POSITIONS OF X I.E. X(I+NN+NN)
THE T TEMPERATURE IS IN THE NN+NN+NNCN+I POSITIONS OF X I.E.
X(I+NN+NN+NNCN)
THERE ARE NNCN PRESSURE NODES (NNCN=NUMBER OF CORNER NODES)
TM MUST BE DIMENSIONED 3*NN+NNCN X 3*NN+NNCN

DATA NWRITE/6/
DATA STOP/STOP//
DIMENSION XC(125),YC(125),NODE(125,6),NVS(125),NCN(125)
DIMENSION X(117),NVIS(117),NCP(125),NPS(125),Q(117)
DIMENSION TMS(125),TI(117)
DIMENSION TMS(21,21),N(21),NQIS(117)
DIMENSION RPS(6),ZPS(6)
DIMENSION XCS(3),YCS(3)
DIMENSION CDS(21,21)
DIMENSION Y(7,117),W(117,132)
COMMON CD(117,117),TM(117,117),RHS(117)

SPECIFY WHETHER TWO DIMENSIONAL, INCLUDING NON-LINEAR TERMS,
(NCASE=1) OR AXISYMMETRIC (NCASE=2)

      READ(NREAD,500)NCASE
      IF(NCASE.EQ.1)GO TO 5
      WRITE(NWRITE,2015)
      GO TO 6
      5 WRITE(NWRITE,2020)
      6 CONTINUE

TIME00001
TIME00002
TIME00003
TIME00004
TIME00005
TIME00006
TIME00007
TIME00008
TIME00009
TIME00010
TIME00011
TIME00012
TIME00013
TIME00014
TIME00015
TIME00016
TIME00017
TIME00018
TIME00019
TIME00020
TIME00021
TIME00022
TIME00023
TIME00024
TIME00025
TIME00026
TIME00027
TIME00028
TIME00029
TIME00030
TIME00031
TIME00032
TIME00033
TIME00034
TIME00035
TIME00036

```

THE FIRST PART OF THE PROGRAM CAN BE CONSIDERED AS AN INPUT ROUTINE







IN WHICH LINES 049 TO 0303 ARE INPUT VERIFICATION OF ALL DATA.  
SUCH A SECTION WOULD BE PART OF ANY FINITE ELEMENT PROGRAM.

READ IN NUMBER OF NODES AND ELEMENTS AND NO. OF CORNER NODES

READ(NREAD,1005)NN,NE,NNCN

INITIALIZE ALL PARAMETERS

```

MM=2*NN+NNCN
MMM=3*NN+NNCN
DO 50 I=1,MMM
  XC(I)=0.00
  YC(I)=0.00
  NVS(I)=0
  NCP(I)=0
  NPS(I)=0
  NQS(I)=0
  DO 50 J=1,6
    NODE(I,J)=0
  CONTINUE
  DO 51 I=1,MMM
    TI(I)=0.00
    Y(I)=0.00
    X(I)=0.00
    NVIS(I)=0
    Q(I)=0.00
    NQIS(I)=0.00
    RHS(I)=0.00
    DO 51 J=1,MMM
      TM(I,J)=0.00
      CC(I,J)=0.00
    CONTINUE
  DO 52 I=1,21
    N(I)=0
    DO 52 J=1,21
      TM$(I,J)=0.00
      CD$(I,J)=0.00
    CONTINUE
  DO 53 I=1,6
    RP$(I)=0.00
    ZP$(I)=0.00
  CONTINUE
  DO 54 I=1,3
    XC$(I)=0.00
    YC$(I)=0.00

```

50

51

52

53

TIME0038  
TIME0040  
TIME0041  
TIME0042  
TIME0043  
TIME0044  
TIME0045  
TIME0046  
TIME0047  
TIME0048  
TIME0049  
TIME0050  
TIME0051  
TIME0052  
TIME0053  
TIME0054  
TIME0055  
TIME0056  
TIME0057  
TIME0058  
TIME0059  
TIME0060  
TIME0061  
TIME0062  
TIME0063  
TIME0064  
TIME0065  
TIME0066  
TIME0067  
TIME0068  
TIME0069  
TIME0070  
TIME0071  
TIME0072  
TIME0073  
TIME0074  
TIME0075  
TIME0076  
TIME0077  
TIME0078  
TIME0079  
TIME0080  
TIME0081  
TIME0082  
TIME0083  
TIME0084  
TIME0085  
TIME0086



```

54 CONTINUE
READ NODE NUMBER AND COORDINATES
DO 100 J=1,NN
  READ(NREAD,1006)WORD,I,XC(I),YC(I)
  IF(WORD.EQ.STOP) GO TO 101
  NCN(J)=I
  CONTINUE
100 NNCN=J-1
  THE ARRAY NCP(J) GENERATES THE GLOBAL PRESSURE INDICES (P1,P2,ETC.)
  THUS PRESSURE NODES ARE LABELED AS CORNER NODES AND ARE INPUTED
  WHEN CNE INPUTS A GLOBAL CORNER NODE FOR J
DO 107 J=1,NNCN
  NCP(NCN(J))=J+NN+NN
107 CONTINUE
READ SYSTEM TOPOLOGY (ELEMENT NO. AND NODE NUMBERS IN
COUNTERCLOCKWISE FASHION STARTING AT THE UPPER LEFT
HAND CORNER NODE).
DO 105 I=1,NE
  READ(NREAD,1010)J,NODE(J,1),NODE(J,2),NODE(J,3),
  1 NODE(J,4),NODE(J,5),NODE(J,6)
105 CONTINUE
  MAXDIF=0
  DO 108 I=1,NE
    DO 108 J=1,6
      DO 108 K=1,6
        LL=IABS(NODE(I,J)-NODE(I,K))
        IF(LL.GT.MAXDIF) MAXDIF=LL
        IBAND=2*(MAXDIF+1)
        NEQ=3*NN+NNCN
      CONTINUE
108 WRITE(NWRITE,1017)IBAND,NEQ
1017 FORMAT(5X,'IBAND=',I3,'/',5X,'NEQ =',I3,/)
  READ NODES WHERE BOTH U AND V VELOCITY IS SPECIFIED
DO 110 I=1,MM
  READ(NREAD,1006)WORD,NVELS,VELU,VELV
  IF(WORD.EQ.STOP) GO TO 111
  NVS(I)=NVELS
  X(NVS(I))=VELU
  X(NVS(I))+NN=VELV
110 CONTINUE

```

```

TIME0087
TIME0088
TIME0089
TIME0090
TIME0091
TIME0092
TIME0093
TIME0094
TIME0095
TIME0096
TIME0097
TIME0098
TIME0099
TIME0100
TIME0101
TIME0102
TIME0103
TIME0104
TIME0105
TIME0106
TIME0107
TIME0108
TIME0109
TIME0110
TIME0111
TIME0112
TIME0113
TIME0114
TIME0115
TIME0116
TIME0117
TIME0118
TIME0119
TIME0120
TIME0121
TIME0122
TIME0123
TIME0124
TIME0125
TIME0126
TIME0127
TIME0128
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TIME0131
TIME0132
TIME0133
TIME0134

```



TIME0135  
TIME0136  
TIME0137  
TIME0138  
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TIME0179  
TIME0180  
TIME0181  
TIME0182

COUNT NODES HAVING SPECIFIED VELOCITIES

111 NNVELS=I-1

READ QX AND QY VALUES AT INTERNAL NODES

```
DO 125 I=1,NN
  READ(NREAD,1006)WORD,NQXY,QXNS,QYNS
  IF(WORD.EQ.STOP) GO TO 126
  NQS(I)=NQXY
  Q(NQS(I))=QXNS
  Q(NQS(I)+NN)=QYNS
125 CONTINUE
```

COUNT NODES HAVING SPECIFIED QX AND QY

126 NNQXY=I-1

READ NOCE NUMBER AND PRESSURE WHERE SPECIFIED

```
DO 130 I=1,NN
  READ(NREAD,1025)WORD,NP,PNP
  IF(WORD.EQ.STOP)GO TO 135
  NPS(I)=NP
  X(NCP(NPS(I)))=PNP
130 CONTINUE
```

COUNT BOUNDARY NODES WHERE PRESSURE IS SPECIFIED

135 NNPS=I-1

READ NOCE NUMBER AND TEMPERATURE WHERE SPECIFIED

```
DO 140 I=1,MM
  READ(NREAD,1025) WORD,NTEMP,TNT
  IF(WORD.EQ.STOP) GO TO 145
  NVS(I+NNVELS)=NTEMP
  X(NVS(I+NNVELS)+MM)=TNT
140 CONTINUE
```

COUNT NODES HAVING SPECIFIED TEMPERATURES

145 NNTS=I-1

READ NOCE NUMBERS AND QZC WHERE SPECIFIED

DO 141 I=1,MM





```

      REAC(NREAD,1025) WORD,NQZC,QZCNS
      IF(WORD.EQ.STOP) GO TO 146
      NQS(NNQXY+I)=NQZC
      Q(NQS(NNQXY+I)+2*NN)=QZCNS
      141 CONTINUE

```

```

      COUNT NODES WHERE QZC IS SPECIFIED

```

```

      146 NNQZC=I-1

```

```

      READ NODE NUMBERS AND HEAT FLUX QZ WHERE SPECIFIED

```

```

      DO 142 I=1,MM
      READ(NREAD,1025) WORD,NQZ,QZNS
      IF(WORD.EQ.STOP) GO TO 147
      NQS(NNQXY+NNQZC+I)=NQZ
      Q(NQS(NNQXY+NNQZC+I)+MM)=QZNS
      142 CONTINUE

```

```

      COUNT NODES WHERE HEAT FLUX QZ IS SPECIFIED

```

```

      147 NNQZ=I-1

```

```

      NQIS IS A LIST OF INDICES OF KNOWN QX,QY,QZC AND QZ

```

```

      DO 1140 I=1,NNQXY
      NQIS(I)=NQS(I)
      NQIS(I+NNQXY)=NQS(I)+NN
      1140 CONTINUE

```

```

      DO 1141 I=1,NNQZC
      NQIS(2*NNQXY+I)=NQS(NNQXY+I)+2*NN
      1141 CONTINUE

```

```

      DO 1145 I=1,NNQZ
      NQIS(2*NNQXY+NNQZC+I)=NQS(NNQXY+NNQZC+I)+MM
      1145 CONTINUE

```

```

      NVIS IS A LIST OF KNOWN VELOCITY, PRESSURE, AND TEMPERATURE INDICES

```

```

      DO 1150 I=1,NNVELS
      NVIS(I)=NVS(I)
      NVIS(I+NNVELS)=NVS(I)+NN
      1150 CONTINUE

```

```

      DO 1155 J=1,NNPS
      NVIS(2*NNVELS+J)=NCP(NPS(J))
      1155 CONTINUE

```

```

      DO 1160 K=1,NNTS
      NVIS(2*NNVELS+NNPS+K)=NVS(K+NNVELS)+MM
      1160 CONTINUE

```

```

TIME0183
TIME0184
TIME0185
TIME0186
TIME0187
TIME0188
TIME0189
TIME0190
TIME0191
TIME0192
TIME0193
TIME0194
TIME0195
TIME0196
TIME0197
TIME0198
TIME0199
TIME0200
TIME0201
TIME0202
TIME0203
TIME0204
TIME0205
TIME0206
TIME0207
TIME0208
TIME0209
TIME0210
TIME0211
TIME0212
TIME0213
TIME0214
TIME0215
TIME0216
TIME0217
TIME0218
TIME0219
TIME0220
TIME0221
TIME0222
TIME0223
TIME0224
TIME0225
TIME0226
TIME0227
TIME0228
TIME0229
TIME0230

```





T I M E 0 2 3 1  
 T I M E 0 2 3 2  
 T I M E 0 2 3 3  
 T I M E 0 2 3 4  
 T I M E 0 2 3 5  
 T I M E 0 2 3 6  
 T I M E 0 2 3 7  
 T I M E 0 2 3 8  
 T I M E 0 2 3 9  
 T I M E 0 2 4 0  
 T I M E 0 2 4 1  
 T I M E 0 2 4 2  
 T I M E 0 2 4 3  
 T I M E 0 2 4 4  
 T I M E 0 2 4 5  
 T I M E 0 2 4 6  
 T I M E 0 2 4 7  
 T I M E 0 2 4 8  
 T I M E 0 2 4 9  
 T I M E 0 2 5 0  
 T I M E 0 2 5 1  
 T I M E 0 2 5 2  
 T I M E 0 2 5 3  
 T I M E 0 2 5 4  
 T I M E 0 2 5 5  
 T I M E 0 2 5 6  
 T I M E 0 2 5 7  
 T I M E 0 2 5 8  
 T I M E 0 2 5 9  
 T I M E 0 2 6 0  
 T I M E 0 2 6 1  
 T I M E 0 2 6 2  
 T I M E 0 2 6 3  
 T I M E 0 2 6 4  
 T I M E 0 2 6 5  
 T I M E 0 2 6 6  
 T I M E 0 2 6 7  
 T I M E 0 2 6 8  
 T I M E 0 2 6 9  
 T I M E 0 2 7 0  
 T I M E 0 2 7 1  
 T I M E 0 2 7 2  
 T I M E 0 2 7 3  
 T I M E 0 2 7 4  
 T I M E 0 2 7 5  
 T I M E 0 2 7 6  
 T I M E 0 2 7 7  
 T I M E 0 2 7 8

NNHC=NUMBER OF NODES WHERE HEAT TRANSFER COEFFICIENT IS SPECIFIED

NNHC=0

NTOTQ=TCTAL NUMBER OF KNOWN QX,QY,QZC, AND QZ

NTOTQ=2\*NNQXY+NNQZC+NNQZ

NTOTVP=TOTAL NUMBER OF KNOWN VELOCITIES, PRESSURES, AND TEMPERATURES

NTOTVP=2\*NNVELS+NNPS+NNNTS

PRINT ALL INPUT DATA

WRITE(NWRITE,1035)NN,NE,NNCN

WRITE(NWRITE,1036)NNVELS

WRITE(NWRITE,1037)NNQXY

WRITE(NWRITE,1038)NNPS

WRITE(NWRITE,1039)NNNTS

WRITE(NWRITE,1034)NNQZC

WRITE(NWRITE,1041)

DO 150 I=1,NNCN

WRITE(NWRITE,1045)NCN(I),XC(NCN(I)),YC(NCN(I))

CONTINUE

150 WRITE(NWRITE,1050)

DO 155 I=1,NE

WRITE(NWRITE,1055)I,NODE(I,1),NODE(I,2),NODE(I,3),

155 CGTINUE

WRITE(NWRITE,1060)

DO 160 I=1,NNVELS

WRITE(NWRITE,1065)I,NVS(I),X(NVS(I)),X(NVS(I))+NN)

CONTINUE

160 WRITE(NWRITE,1070)

DO 165 I=1,NNQXY

WRITE(NWRITE,1065)I,NQS(I),Q(NQS(I)),Q(NQS(I))+NN)

CONTINUE

165 WRITE(NWRITE,1080)

DO 170 I=1,NNPS

WRITE(NWRITE,1085)I,NPS(I),X(NCP(NPS(I)))

CONTINUE

170 WRITE(NWRITE,1081)

DO 171 I=1,NNNTS

WRITE(NWRITE,1085)I,NVS(I+NNVELS),X(NVS(I+NNVELS))+MM)

CONTINUE

171 WRITE(NWRITE,1083)



```

DO 173 I=1, NNQZC
WRITE(NWRITE,1085) I, NQS(I+NNQXY), Q(NQS(I+NNQXY)+2*NN)
CONTINUE
173 WRITE(NWRITE,1082)
DO 172 I=1, NNQZ
WRITE(NWRITE,1085) I, NQS(I+NNQXY+NNQZC), Q(NQS(I+NNQXY+NNQZC)+MM)
CONTINUE
172 T=0.00
DO CONTINUE
177 CONTINUE
DO 178 I=1, MMM
RHS(I)=0.00
DO 178 J=1, MMM
TM(I,J)=0.00
CONTINUE
178 IF(T.GT.0.00) GO TO 180
DO 179 I=1, MMM
CC(I,J)=0.00
CONTINUE
179 CONTINUE
DO 181 I=1, 21
DO 181 J=1, 21
TM$(I,J)=0.00
CC$(I,J)=0.00
CONTINUE
181

```

END OF INPUT AND VERIFICATION ROUTINE

```

DO 300 K=1, NE
N1=NODE(K,1)
N2=NODE(K,2)
N3=NODE(K,3)
N4=NODE(K,4)
N5=NODE(K,5)
N6=NODE(K,6)
N7=NODE(K,1)+NN
N8=NODE(K,2)+NN
N9=NODE(K,3)+NN
N10=NODE(K,4)+NN
N11=NODE(K,5)+NN
N12=NODE(K,6)+NN
N13=NCP(NODE(K,1))
N14=NCP(NODE(K,3))
N15=NCP(NODE(K,5))
N16=NODE(K,1)+MM
N17=NODE(K,2)+MM
N18=NODE(K,3)+MM
N19=NODE(K,4)+MM

```

TIME0279  
TIME0280  
TIME0281  
TIME0282  
TIME0283  
TIME0284  
TIME0285  
TIME0286  
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TIME0290  
TIME0291  
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TIME0296  
TIME0297  
TIME0298  
TIME0299  
TIME0300  
TIME0301  
TIME0302  
TIME0303  
TIME0304  
TIME0305  
TIME0306  
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TIME0318  
TIME0319  
TIME0320  
TIME0321  
TIME0322  
TIME0323  
TIME0324  
TIME0325  
TIME0326



ME03278	ME03279	ME03280	ME03281	ME03282	ME03283	ME03284	ME03285	ME03286	ME03287	ME03288	ME03289	ME03290	ME03291	ME03292	ME03293	ME03294	ME03295	ME03296	ME03297	ME03298	ME03299	ME03300	ME03301	ME03302	ME03303	ME03304	ME03305	ME03306	ME03307	ME03308	ME03309	ME03310	ME03311	ME03312	ME03313	ME03314	ME03315	ME03316	ME03317	ME03318	ME03319	ME03320	ME03321	ME03322	ME03323	ME03324	ME03325	ME03326	ME03327	ME03328	ME03329	ME03330	ME03331	ME03332	ME03333	ME03334	ME03335	ME03336	ME03337	ME03338	ME03339	ME03340	ME03341	ME03342	ME03343	ME03344	ME03345	ME03346	ME03347	ME03348	ME03349	ME03350	ME03351	ME03352	ME03353	ME03354	ME03355	ME03356	ME03357	ME03358	ME03359	ME03360	ME03361	ME03362	ME03363	ME03364	ME03365	ME03366	ME03367	ME03368	ME03369	ME03370	ME03371	ME03372	ME03373	ME03374
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```

N20=NODE(K,5)+MM
N21=NODE(K,6)+MM
XC$(1)=XC(NODE(K,1))
XC$(2)=XC(NODE(K,3))
XC$(3)=XC(NODE(K,5))
YC$(1)=YC(NODE(K,1))
YC$(2)=YC(NODE(K,3))
YC$(3)=YC(NODE(K,5))
A=1.DO
ZBAR=(YC$(1)+YC$(2)+YC$(3))/3.DO
RBAR=(XC$(1)+XC$(2)+XC$(3))/3.DO
IF(NCASE.EQ.2)A=RBAR
AA=1.DO
IF(NCASE.EQ.2)AA=2.DO*3.14159DO*(RBAR)
A1=XC$(2)*YC$(3)-XC$(3)*YC$(2)
A2=XC$(3)*YC$(1)-XC$(1)*YC$(3)
A3=XC$(1)*YC$(2)-XC$(2)*YC$(1)
B1=YC$(2)-YC$(3)
B2=YC$(3)-YC$(1)
B3=XC$(3)-XC$(2)
C1=XC$(3)-XC$(2)
C2=XC$(1)-XC$(3)
C3=XC$(2)-XC$(1)
DEL=DABS(0.5DO*(XC$(1)+XC$(2)+XC$(3)-YC$(1)-YC$(2)-YC$(3)))
1+XC$(3))*YC$(1)-YC$(2))*A/(3.DO*DEL))*AA
CNST=10.90DO*(1.DO*A/(3.DO*DEL))*AA
D1=-B1/6.DO
D2=-B2/6.DO
D3=-B3/6.DO
E1=-C1/6.DO
E2=-C2/6.DO
E3=-C3/6.DO
F1=B1/2520.DO
F2=B2/2520.DO
F3=B3/2520.DO
G1=C1/2520.DO
G2=C2/2520.DO
G3=C3/2520.DO
U1=TI(N1)
U2=TI(N2)
U3=TI(N3)
U4=TI(N4)
U5=TI(N5)
U6=TI(N6)
V1=TI(N7)
V2=TI(N8)
V3=TI(N9)
V4=TI(N10)

```





```

V5=TL(N11)
V6=TL(N12)
TM$(1,1)=0.75D0*(B1*B1+C1*C1)*CONST
TM$(1,2)=(B1*B2+C1*C2)*CONST
TM$(1,3)=-TM$(1,2)*0.25D0
TM$(1,4)=0
TM$(1,6)=(B1*B3+C1*C3)*CONST
TM$(1,5)=-TM$(1,6)*0.25D0
TM$(3,3)=.75D0*(B2*B2+C2*C2)*CONST
TM$(3,4)=(B2*B3+C2*C3)*CONST
TM$(3,5)=-TM$(3,4)*0.25D0
TM$(2,1)=TM$(1,2)
TM$(2,2)=TM$(1,2)
TM$(2,3)=8.D0/3.D0*(TM$(1,1)+TM$(3,3))+2.D0*TM$(1,2)
TM$(2,4)=2.D0*TM$(1,6)+TM$(3,4)+TM$(1,2)+4.D0/3.D0*TM$(3,3)
TM$(2,5)=0
TM$(2,6)=TM$(1,6)+2.D0*TM$(3,4)+TM$(1,2)+4.D0/3.D0*TM$(1,1)
TM$(3,1)=TM$(1,3)
TM$(3,2)=TM$(1,3)
TM$(3,6)=0
TM$(5,5)=.75D0*(B3*B3+C3*C3)*CONST
TM$(4,1)=TM$(1,4)
TM$(4,2)=TM$(2,4)
TM$(4,3)=TM$(3,4)
TM$(4,4)=8.D0/3.D0*(TM$(3,3)+TM$(5,5))+2.D0*TM$(3,4)
TM$(4,5)=TM$(3,4)
TM$(4,6)=TM$(1,6)+TM$(3,4)+2.D0*TM$(1,2)+4.D0/3.D0*TM$(5,5)
TM$(6,3)=TM$(3,6)
TM$(5,1)=TM$(1,5)
TM$(5,2)=TM$(2,5)
TM$(5,3)=TM$(3,5)
TM$(5,4)=TM$(4,5)
TM$(5,6)=TM$(1,6)
TM$(6,1)=TM$(1,6)
TM$(6,2)=TM$(2,6)
TM$(6,4)=TM$(4,6)
TM$(6,5)=TM$(5,6)
TM$(6,6)=8.D0/3.D0*(TM$(5,5)+TM$(1,1))+2.D0*TM$(1,6)
IF(NCASE.NE.1) GO TO 3000

BEGIN INPUT OF NON-LINEAR TERMS

TM$(1,1)=TM$(1,1)
1-(78.D0*U1+48.D0*U2-9.D0*U3+12.D0*U4-9.D0*U5+48.D0*U6)*F1
2-(78.D0*V1+48.D0*V2-9.D0*V3+12.D0*V4-9.D0*V5+48.D0*V6)*F1
3*G1
TM$(2,1)=TM$(2,1)
1-(48.D0*U1+160.D0*U2-32.D0*U3+16.D0*U4-20.D0*U5+80.D0*U6)*F1

```

```

TIME0375
TIME0376
TIME0377
TIME0378
TIME0379
TIME0380
TIME0381
TIME0382
TIME0383
TIME0384
TIME0385
TIME0386
TIME0387
TIME0388
TIME0389
TIME0390
TIME0391
TIME0392
TIME0393
TIME0394
TIME0395
TIME0396
TIME0397
TIME0398
TIME0399
TIME0400
TIME0401
TIME0402
TIME0403
TIME0404
TIME0405
TIME0406
TIME0407
TIME0408
TIME0409
TIME0410
TIME0411
TIME0412
TIME0413
TIME0414
TIME0415
TIME0416
TIME0417
TIME0418
TIME0419
TIME0420
TIME0421
TIME0422

```









TM\$(6,2)=TM\$(6,2)  
 1-(-16.D0\*U1+128.D0\*U2-16.D0\*U3+128.D0\*U4-16.D0\*U5+128.D0\*U6)\*F1  
 2-(-16.D0\*V1+128.D0\*V2-16.D0\*V3+128.D0\*V4-16.D0\*V5+128.D0\*V6)\*G1  
 3D0\*U5+384.D0\*U6)\*F2  
 4D0\*U5+384.D0\*U6)\*F2  
 5128.D0\*V4-32.D0\*V5+384.D0\*V6)\*G2  
 TM\$(1,3)=TM\$(1,3)  
 1-(-18.D0\*U1-32.D0\*U2-9.D0\*U3-20.D0\*U4+11.D0\*U5-16.D0\*U6)\*F2  
 2-(-18.D0\*V1-32.D0\*V2-9.D0\*V3-20.D0\*V4+11.D0\*V5-16.D0\*V6)\*G2  
 36)\*G2  
 TM\$(2,3)=TM\$(2,3)  
 1-(-32.D0\*U1+160.D0\*U2+48.D0\*U3+80.D0\*U4-20.D0\*U5+16.D0\*U6)\*F2  
 2-(-32.D0\*V1+160.D0\*V2+48.D0\*V3+80.D0\*V4-20.D0\*V5+16.D0\*V6)\*G2  
 3TM\$(3,3)=TM\$(3,3)  
 1-(-9.D0\*U1+48.D0\*U2+78.D0\*U3+48.D0\*U4-9.D0\*U5+12.D0\*U6)\*F2  
 2-(-9.D0\*V1+48.D0\*V2+78.D0\*V3+48.D0\*V4-9.D0\*V5+12.D0\*V6)\*G2  
 3TM\$(4,3)=TM\$(4,3)  
 1-(-20.D0\*U1+80.D0\*U2+48.D0\*U3+160.D0\*U4-32.D0\*U5+16.D0\*U6)\*F2  
 2-(-20.D0\*V1+80.D0\*V2+48.D0\*V3+160.D0\*V4-32.D0\*V5+16.D0\*V6)\*G2  
 3TM\$(5,3)=TM\$(5,3)  
 1-(-11.D0\*U1-20.D0\*U2-9.D0\*U3-32.D0\*U4-18.D0\*U5-16.D0\*U6)\*F2  
 2-(-11.D0\*V1-20.D0\*V2-9.D0\*V3-32.D0\*V4-18.D0\*V5-16.D0\*V6)\*G2  
 3TM\$(6,3)=TM\$(6,3)  
 1-(-16.D0\*U1+16.D0\*U2+12.D0\*U3+16.D0\*U4-16.D0\*U5-96.D0\*U6)\*F2  
 2-(-16.D0\*V1+16.D0\*V2+12.D0\*V3+16.D0\*V4-16.D0\*V5-96.D0\*V6)\*G2  
 3TM\$(1,4)=TM\$(1,4)  
 1-(-24.D0\*U1-16.D0\*U2+4.D0\*U3-48.D0\*U4-16.D0\*U5-32.D0\*U6)\*F2  
 2-(-24.D0\*V1-16.D0\*V2+4.D0\*V3-48.D0\*V4-16.D0\*V5-32.D0\*V6)\*G2  
 3TM\$(2,4)=TM\$(2,4)  
 1-(-16.D0\*U1+128.D0\*U2-16.D0\*U3+128.D0\*U4-16.D0\*U5+128.D0\*U6)\*F2  
 2-(-16.D0\*V1+128.D0\*V2-16.D0\*V3+128.D0\*V4-16.D0\*V5+128.D0\*V6)\*G2  
 3D0\*U5+128.D0\*U6)\*F3  
 48.D0\*U5+128.D0\*U6)\*F3  
 5192.D0\*V4-48.D0\*V5+128.D0\*V6)\*G3  
 TM\$(3,4)=TM\$(3,4)  
 1-(-4.D0\*U1-16.D0\*U2+24.D0\*U3-32.D0\*U4+16.D0\*U5-48.D0\*U6)\*F2  
 2-(-4.D0\*V1-16.D0\*V2+24.D0\*V3-32.D0\*V4+16.D0\*V5-48.D0\*V6)\*G2  
 3D0\*U5-16.D0\*U6)\*F3  
 4D0\*U5-16.D0\*U6)\*F3  
 548.D0\*V4-16.D0\*V5-16.D0\*V6)\*G3

TME04771  
 TME04772  
 TME04773  
 TME04774  
 TME04775  
 TME04776  
 TME04777  
 TME04778  
 TME04779  
 TME04780  
 TME04811  
 TME04812  
 TME04823  
 TME04845  
 TME04866  
 TME04887  
 TME04888  
 TME04889  
 TME04900  
 TME04911  
 TME04923  
 TME04934  
 TME04945  
 TME04956  
 TME04967  
 TME04978  
 TME04999  
 TME05000  
 TME05011  
 TME05023  
 TME05034  
 TME05045  
 TME05056  
 TME05067  
 TME05078  
 TME05089  
 TME05100  
 TME05112  
 TME05123  
 TME05134  
 TME05145  
 TME05156  
 TME05167  
 TME05178



```

TM$(4,4)=TM$(4,4)
1-(-48.D0*U1+128.D0*U2-32.D0*U3+384.D0*U4+48.D0*U5+192.D0*U6)*F2
2-(-48.D0*V1+128.D0*V2-32.D0*V3+384.D0*V4+48.D0*V5+192.D0*V6)*G2
3D0*U5+128.D0*U6)*F3
42.D0*U4-32.D0*V5+128.D0*V6)*G3
5+384.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*F2
TM$(5,4)=TM$(5,4)
1-(-16.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*F2
2*V6)*G2
3*U5-16.D0*U6)*F3
4D0*V4+24.D0*V5-16.D0*V6)*G3
5.D0*V6)*G3
TM$(6,4)=TM$(6,4)
1-(-32.D0*U1+128.D0*U2-48.D0*U3+192.D0*U4+48.D0*U5+384.D0*U6)*F2
2-(-32.D0*V1+128.D0*V2-48.D0*V3+192.D0*V4+48.D0*V5+384.D0*V6)*G2
3D0*U5+128.D0*U6)*F3
46.D0*U4-16.D0*V5+128.D0*V6)*G3
5+128.D0*U1-16.D0*U2+11.D0*V3-20.D0*V4-9.D0*V5-32.D0*V6)*G3
TM$(1,5)=TM$(1,5)
1-(-18.D0*U1-16.D0*U2+11.D0*U3-20.D0*U4-9.D0*U5-32.D0*U6)*F3
2-(-18.D0*V1-16.D0*V2+11.D0*V3-20.D0*V4-9.D0*V5-32.D0*V6)*G3
3TM$(2,5)=TM$(2,5)
1-(-16.D0*U1-96.D0*U2-16.D0*U3+16.D0*U4+12.D0*U5+16.D0*U6)*F3
2-(-16.D0*V1-96.D0*V2-16.D0*V3+16.D0*V4+12.D0*V5+16.D0*V6)*G3
3TM$(3,5)=TM$(3,5)
1-(-11.D0*U1-16.D0*U2-18.D0*U3-32.D0*U4-9.D0*U5-20.D0*U6)*F3
2-(-11.D0*V1-16.D0*V2-18.D0*V3-32.D0*V4-9.D0*V5-20.D0*V6)*G3
3TM$(4,5)=TM$(4,5)
1-(-20.D0*U1+16.D0*U2-32.D0*U3+160.D0*U4+48.D0*U5+80.D0*U6)*F3
2-(-20.D0*V1+16.D0*V2-32.D0*V3+160.D0*V4+48.D0*V5+80.D0*V6)*G3
3TM$(5,5)=TM$(5,5)
1-(-9.D0*U1+12.D0*U2-9.D0*U3+48.D0*U4+78.D0*U5+48.D0*U6)*F3
2-(-9.D0*V1+12.D0*V2-9.D0*V3+48.D0*V4+78.D0*V5+48.D0*V6)*G3
3TM$(6,5)=TM$(6,5)
1-(-32.D0*U1+16.D0*U2-20.D0*U3+80.D0*U4+48.D0*U5+160.D0*U6)*F3
2-(-32.D0*V1+16.D0*V2-20.D0*V3+80.D0*V4+48.D0*V5+160.D0*V6)*G3
3TM$(1,6)=TM$(1,6)
1-(-24.D0*U1-16.D0*U2+4.D0*U3-48.D0*U4-16.D0*U5-32.D0*U6)*F1
2-(-24.D0*V1-16.D0*V2+4.D0*V3-48.D0*V4-16.D0*V5-32.D0*V6)*G1
3D0*U5+48.D0*U6)*F3
4D0*V4-16.D0*V5+48.D0*V6)*G3
516.D0*V4-16.D0*V5+48.D0*V6)*G3

```





```

TM$(2,6)=TM$(2,6)
1-(-16.D0*U1+128.D0*U2-16.D0*U3+128.D0*U4-16.D0*U5+128.D0*U6)*F1
2-(-16.D0*V1+128.D0*V2-16.D0*V3+128.D0*V4-16.D0*V5+128.D0*V6)*G1
3D0*U5+192.D0*U6)*F3
4D0*V4-48.D0*V5+192.D0*V6)*G3
5128.D0*U1-16.D0*U2+24.D0*U3-32.D0*U4-16.D0*U5+128.D0*U6)*F3
TM$(3,6)=TM$(3,6)
1-(-16.D0*U1-16.D0*U2+24.D0*U3-32.D0*U4-16.D0*U5+128.D0*U6)*F3
2-(-16.D0*V1-16.D0*V2+24.D0*V3-32.D0*V4-48.D0*V5+128.D0*V6)*G3
3D0*U5-48.D0*U6)*F3
4D0*V4+4.D0*V5-48.D0*V6)*G3
516.D0*U1+128.D0*U2-32.D0*U3+384.D0*U4+48.D0*U5+192.D0*U6)*F1
TM$(4,6)=TM$(4,6)
1-(-48.D0*U1+128.D0*U2-32.D0*U3+384.D0*U4+48.D0*U5+192.D0*U6)*F1
2-(-48.D0*V1+128.D0*V2-32.D0*V3+384.D0*V4+48.D0*V5+192.D0*V6)*G1
3D0*U5+128.D0*U6)*F3
46.D0*U4-16.D0*U5+128.D0*V5+128.D0*V6)*G3
5+128.D0*V4-16.D0*V5+128.D0*V6)*G3
TM$(5,6)=TM$(5,6)
1-(-16.D0*U1-16.D0*U2-16.D0*U3+48.D0*U4+120.D0*U5+48.D0*U6)*F1
2-(-16.D0*V1-16.D0*V2-16.D0*V3+48.D0*V4+120.D0*V5+48.D0*V6)*G1
3D0*U5-32.D0*U6)*F3
40*U5-32.D0*V5-32.D0*V6)*G3
56.D0*U4+24.D0*U5+128.D0*U6)*F3
TM$(6,6)=TM$(6,6)
1-(-32.D0*U1+128.D0*U2-48.D0*U3+192.D0*U4+48.D0*U5+384.D0*U6)*F1
2-(-32.D0*V1+128.D0*V2-48.D0*V3+192.D0*V4+48.D0*V5+384.D0*V6)*G1
3D0*U5+384.D0*U6)*F3
4D0*U5+384.D0*V6)*G3
5128.D0*U4-32.D0*U5+384.D0*V6)*G3

THIS ENDS ADDITION OF NON-LINEAR TERMS TO THE LOCAL ARRAY

3000 CONTINUE
TM$(7,7)=TM$(1,1)
TM$(7,8)=TM$(1,2)
TM$(7,9)=TM$(1,3)
TM$(7,10)=TM$(1,4)
TM$(7,11)=TM$(1,5)
TM$(7,12)=TM$(1,6)
TM$(8,7)=TM$(2,1)
TM$(8,8)=TM$(2,2)
TM$(8,9)=TM$(2,3)
TM$(8,10)=TM$(2,4)
TM$(8,11)=TM$(2,5)
TM$(8,12)=TM$(2,6)
TM$(9,7)=TM$(3,1)
TM$(9,8)=TM$(3,2)

```







```

9)=0.0.0
1,5)=E2+E3
1,10)=E2+2.00*E3
1,13)=E2+E2+E3
1,14)=E2+E2+E3
1,15)=0.0.0
1,16)=0.0.0
1,17)=E3
1,18)=E1+E3
1,19)=E1+E3
1,20)=2.00*E1+E3
1,21)=0.9437500
1,22)=D1*CONST4
1,23)=0.0.0
1,24)=0.0.0
1,25)=0.0.0
1,26)=D1+2.00*D2)*CONST4
1,27)=D1+2.00*D1+D2)*CONST4
1,28)=D1+D2)*CONST4
1,29)=0.0.0
1,30)=D2*CONST4
1,31)=D2+D3)*CONST4
1,32)=D2+2.00*D3)*CONST4
1,33)=D2+D3)*CONST4
1,34)=D2+D3)*CONST4
1,35)=0.0.0
1,36)=D3*CONST4
1,37)=D1+2.00*D3)*CONST4
1,38)=D1+D3)*CONST4
1,39)=D2+D3)*CONST4
1,40)=D2+D3)*CONST4
1,41)=D1*CONST4
1,42)=0.0.0
1,43)=0.0.0
1,44)=E1+2.00*E2)*CONST4
1,45)=E1+2.00*E1+E2)*CONST4
1,46)=E1+E2)*CONST4
1,47)=0.0.0
1,48)=E2*CONST4
1,49)=0.0.0
1,50)=E2+E3)*CONST4
1,51)=E2+2.00*E3)*CONST4
1,52)=E2+E2+E3)*CONST4
1,53)=0.0.0
1,54)=0.0.0
1,55)=E3*CONST4
1,56)=E1+2.00*E3)*CONST4
1,57)=E1+E3)*CONST4
1,58)=E1+E3)*CONST4
1,59)=E1+E3)*CONST4
1,60)=E1+E3)*CONST4
1,61)=E1+E3)*CONST4
1,62)=E1+E3)*CONST4
1,63)=E1+E3)*CONST4
1,64)=E1+E3)*CONST4
1,65)=E1+E3)*CONST4
1,66)=E1+E3)*CONST4
1,67)=E1+E3)*CONST4
1,68)=E1+E3)*CONST4
1,69)=E1+E3)*CONST4
1,70)=E1+E3)*CONST4
1,71)=E1+E3)*CONST4
1,72)=E1+E3)*CONST4
1,73)=E1+E3)*CONST4
1,74)=E1+E3)*CONST4
1,75)=E1+E3)*CONST4
1,76)=E1+E3)*CONST4
1,77)=E1+E3)*CONST4
1,78)=E1+E3)*CONST4
1,79)=E1+E3)*CONST4
1,80)=E1+E3)*CONST4
1,81)=E1+E3)*CONST4
1,82)=E1+E3)*CONST4
1,83)=E1+E3)*CONST4
1,84)=E1+E3)*CONST4
1,85)=E1+E3)*CONST4
1,86)=E1+E3)*CONST4
1,87)=E1+E3)*CONST4
1,88)=E1+E3)*CONST4
1,89)=E1+E3)*CONST4
1,90)=E1+E3)*CONST4
1,91)=E1+E3)*CONST4
1,92)=E1+E3)*CONST4
1,93)=E1+E3)*CONST4
1,94)=E1+E3)*CONST4
1,95)=E1+E3)*CONST4
1,96)=E1+E3)*CONST4
1,97)=E1+E3)*CONST4
1,98)=E1+E3)*CONST4
1,99)=E1+E3)*CONST4
1,100)=E1+E3)*CONST4
1,101)=E1+E3)*CONST4
1,102)=E1+E3)*CONST4
1,103)=E1+E3)*CONST4
1,104)=E1+E3)*CONST4
1,105)=E1+E3)*CONST4
1,106)=E1+E3)*CONST4
1,107)=E1+E3)*CONST4
1,108)=E1+E3)*CONST4
1,109)=E1+E3)*CONST4
1,110)=E1+E3)*CONST4
1,111)=E1+E3)*CONST4
1,112)=E1+E3)*CONST4
1,113)=E1+E3)*CONST4
1,114)=E1+E3)*CONST4
1,115)=E1+E3)*CONST4
1,116)=E1+E3)*CONST4
1,117)=E1+E3)*CONST4
1,118)=E1+E3)*CONST4
1,119)=E1+E3)*CONST4
1,120)=E1+E3)*CONST4
1,121)=E1+E3)*CONST4
1,122)=E1+E3)*CONST4
1,123)=E1+E3)*CONST4
1,124)=E1+E3)*CONST4
1,125)=E1+E3)*CONST4
1,126)=E1+E3)*CONST4
1,127)=E1+E3)*CONST4
1,128)=E1+E3)*CONST4
1,129)=E1+E3)*CONST4
1,130)=E1+E3)*CONST4
1,131)=E1+E3)*CONST4
1,132)=E1+E3)*CONST4
1,133)=E1+E3)*CONST4
1,134)=E1+E3)*CONST4
1,135)=E1+E3)*CONST4
1,136)=E1+E3)*CONST4
1,137)=E1+E3)*CONST4
1,138)=E1+E3)*CONST4
1,139)=E1+E3)*CONST4
1,140)=E1+E3)*CONST4
1,141)=E1+E3)*CONST4
1,142)=E1+E3)*CONST4
1,143)=E1+E3)*CONST4
1,144)=E1+E3)*CONST4
1,145)=E1+E3)*CONST4
1,146)=E1+E3)*CONST4
1,147)=E1+E3)*CONST4
1,148)=E1+E3)*CONST4
1,149)=E1+E3)*CONST4
1,150)=E1+E3)*CONST4
1,151)=E1+E3)*CONST4
1,152)=E1+E3)*CONST4
1,153)=E1+E3)*CONST4
1,154)=E1+E3)*CONST4
1,155)=E1+E3)*CONST4
1,156)=E1+E3)*CONST4
1,157)=E1+E3)*CONST4
1,158)=E1+E3)*CONST4
1,159)=E1+E3)*CONST4
1,160)=E1+E3)*CONST4
1,161)=E1+E3)*CONST4
1,162)=E1+E3)*CONST4
1,163)=E1+E3)*CONST4
1,164)=E1+E3)*CONST4
1,165)=E1+E3)*CONST4
1,166)=E1+E3)*CONST4
1,167)=E1+E3)*CONST4
1,168)=E1+E3)*CONST4
1,169)=E1+E3)*CONST4
1,170)=E1+E3)*CONST4
1,171)=E1+E3)*CONST4
1,172)=E1+E3)*CONST4
1,173)=E1+E3)*CONST4
1,174)=E1+E3)*CONST4
1,175)=E1+E3)*CONST4
1,176)=E1+E3)*CONST4
1,177)=E1+E3)*CONST4
1,178)=E1+E3)*CONST4
1,179)=E1+E3)*CONST4
1,180)=E1+E3)*CONST4
1,181)=E1+E3)*CONST4
1,182)=E1+E3)*CONST4
1,183)=E1+E3)*CONST4
1,184)=E1+E3)*CONST4
1,185)=E1+E3)*CONST4
1,186)=E1+E3)*CONST4
1,187)=E1+E3)*CONST4
1,188)=E1+E3)*CONST4
1,189)=E1+E3)*CONST4
1,190)=E1+E3)*CONST4
1,191)=E1+E3)*CONST4
1,192)=E1+E3)*CONST4
1,193)=E1+E3)*CONST4
1,194)=E1+E3)*CONST4
1,195)=E1+E3)*CONST4
1,196)=E1+E3)*CONST4
1,197)=E1+E3)*CONST4
1,198)=E1+E3)*CONST4
1,199)=E1+E3)*CONST4
1,200)=E1+E3)*CONST4
1,201)=E1+E3)*CONST4
1,202)=E1+E3)*CONST4
1,203)=E1+E3)*CONST4
1,204)=E1+E3)*CONST4
1,205)=E1+E3)*CONST4
1,206)=E1+E3)*CONST4
1,207)=E1+E3)*CONST4
1,208)=E1+E3)*CONST4
1,209)=E1+E3)*CONST4
1,210)=E1+E3)*CONST4
1,211)=E1+E3)*CONST4
1,212)=E1+E3)*CONST4
1,213)=E1+E3)*CONST4
1,214)=E1+E3)*CONST4
1,215)=E1+E3)*CONST4
1,216)=E1+E3)*CONST4
1,217)=E1+E3)*CONST4
1,218)=E1+E3)*CONST4
1,219)=E1+E3)*CONST4
1,220)=E1+E3)*CONST4
1,221)=E1+E3)*CONST4
1,222)=E1+E3)*CONST4
1,223)=E1+E3)*CONST4
1,224)=E1+E3)*CONST4
1,225)=E1+E3)*CONST4
1,226)=E1+E3)*CONST4
1,227)=E1+E3)*CONST4
1,228)=E1+E3)*CONST4
1,229)=E1+E3)*CONST4
1,230)=E1+E3)*CONST4
1,231)=E1+E3)*CONST4
1,232)=E1+E3)*CONST4
1,233)=E1+E3)*CONST4
1,234)=E1+E3)*CONST4
1,235)=E1+E3)*CONST4
1,236)=E1+E3)*CONST4
1,237)=E1+E3)*CONST4
1,238)=E1+E3)*CONST4
1,239)=E1+E3)*CONST4
1,240)=E1+E3)*CONST4
1,241)=E1+E3)*CONST4
1,242)=E1+E3)*CONST4
1,243)=E1+E3)*CONST4
1,244)=E1+E3)*CONST4
1,245)=E1+E3)*CONST4
1,246)=E1+E3)*CONST4
1,247)=E1+E3)*CONST4
1,248)=E1+E3)*CONST4
1,249)=E1+E3)*CONST4
1,250)=E1+E3)*CONST4
1,251)=E1+E3)*CONST4
1,252)=E1+E3)*CONST4
1,253)=E1+E3)*CONST4
1,254)=E1+E3)*CONST4
1,255)=E1+E3)*CONST4
1,256)=E1+E3)*CONST4
1,257)=E1+E3)*CONST4
1,258)=E1+E3)*CONST4
1,259)=E1+E3)*CONST4
1,260)=E1+E3)*CONST4
1,261)=E1+E3)*CONST4
1,262)=E1+E3)*CONST4
1,263)=E1+E3)*CONST4
1,264)=E1+E3)*CONST4
1,265)=E1+E3)*CONST4
1,266)=E1+E3)*CONST4
1,267)=E1+E3)*CONST4
1,268)=E1+E3)*CONST4
1,269)=E1+E3)*CONST4
1,270)=E1+E3)*CONST4
1,271)=E1+E3)*CONST4
1,272)=E1+E3)*CONST4
1,273)=E1+E3)*CONST4
1,274)=E1+E3)*CONST4
1,275)=E1+E3)*CONST4
1,276)=E1+E3)*CONST4
1,277)=E1+E3)*CONST4
1,278)=E1+E3)*CONST4
1,279)=E1+E3)*CONST4
1,280)=E1+E3)*CONST4
1,281)=E1+E3)*CONST4
1,282)=E1+E3)*CONST4
1,283)=E1+E3)*CONST4
1,284)=E1+E3)*CONST4
1,285)=E1+E3)*CONST4
1,286)=E1+E3)*CONST4
1,287)=E1+E3)*CONST4
1,288)=E1+E3)*CONST4
1,289)=E1+E3)*CONST4
1,290)=E1+E3)*CONST4
1,291)=E1+E3)*CONST4
1,292)=E1+E3)*CONST4
1,293)=E1+E3)*CONST4
1,294)=E1+E3)*CONST4
1,295)=E1+E3)*CONST4
1,296)=E1+E3)*CONST4
1,297)=E1+E3)*CONST4
1,298)=E1+E3)*CONST4
1,299)=E1+E3)*CONST4
1,300)=E1+E3)*CONST4
1,301)=E1+E3)*CONST4
1,302)=E1+E3)*CONST4
1,303)=E1+E3)*CONST4
1,304)=E1+E3)*CONST4
1,305)=E1+E3)*CONST4
1,306)=E1+E3)*CONST4
1,307)=E1+E3)*CONST4
1,308)=
```





TM\$(1,16)	=1	.DO*	CONST2
TM\$(2,17)	=1	.DO*	CONST2
TM\$(3,18)	=1	.DO*	CONST2
TM\$(4,19)	=1	.DO*	CONST2
TM\$(5,20)	=1	.DO*	CONST2
TM\$(6,21)	=1	.DO*	CONST2

**ALPHA1=ALPHA/VISCOSITY**

ALPHA1=9.4454D-05

ALPHA1=9.4454D-05  
ALPHA1=ALPHA1  
CONST1=16,17,18,19,20,21,22,23,24,25,26,27,28,29,30,31,32,33,34,35,36,37,38,39,40,41,42,43,44,45,46,47,48,49,50,51,52,53,54,55,56,57,58,59,60,61,62,63,64,65,66,67,68,69,70,71,72,73,74,75,76,77,78,79,80,81,82,83,84,85,86,87,88,89,90,91,92,93,94,95,96,97,98,99,100,101,102,103,104,105,106,107,108,109,110,111,112,113,114,115,116,117,118,119,120,121,122,123,124,125,126,127,128,129,130,131,132,133,134,135,136,137,138,139,140,141,142,143,144,145,146,147,148,149,150,151,152,153,154,155,156,157,158,159,160,161,162,163,164,165,166,167,168,169,170,171,172,173,174,175,176,177,178,179,180,181,182,183,184,185,186,187,188,189,190,191,192,193,194,195,196,197,198,199,200,201,202,203,204,205,206,207,208,209,210,211,212,213,214,215,216,217,218,219,220,221,222,223,224,225,226,227,228,229,230,231,232,233,234,235,236,237,238,239,240,241,242,243,244,245,246,247,248,249,250,251,252,253,254,255,256,257,258,259,260,261,262,263,264,265,266,267,268,269,270,271,272,273,274,275,276,277,278,279,280,281,282,283,284,285,286,287,288,289,290,291,292,293,294,295,296,297,298,299,300,301,302,303,304,305,306,307,308,309,310,311,312,313,314,315,316,317,318,319,320,321,322,323,324,325,326,327,328,329,330,331,332,333,334,335,336,337,338,339,340,341,342,343,344,345,346,347,348,349,350,351,352,353,354,355,356,357,358,359,360,361,362,363,364,365,366,367,368,369,370,371,372,373,374,375,376,377,378,379,380,381,382,383,384,385,386,387,388,389,390,391,392,393,394,395,396,397,398,399,400,401,402,403,404,405,406,407,408,409,410,411,412,413,414,415,416,417,418,419,420,421,422,423,424,425,426,427,428,429,430,431,432,433,434,435,436,437,438,439,440,441,442,443,444,445,446,447,448,449,450,451,452,453,454,455,456,457,458,459,460,461,462,463,464,465,466,467,468,469,470,471,472,473,474,475,476,477,478,479,480,481,482,483,484,485,486,487,488,489,490,491,492,493,494,495,496,497,498,499,500,501,502,503,504,505,506,507,508,509,510,511,512,513,514,515,516,517,518,519,520,521,522,523,524,525,526,527,528,529,530,531,532,533,534,535,536,537,538,539,540,541,542,543,544,545,546,547,548,549,550,551,552,553,554,555,556,557,558,559,560,561,562,563,564,565,566,567,568,569,570,571,572,573,574,575,576,577,578,579,580,581,582,583,584,585,586,587,588,589,590,591,592,593,594,595,596,597,598,599,600,601,602,603,604,605,606,607,608,609,610,611,612,613,614,615,616,617,618,619,620,621,622,623,624,625,626,627,628,629,630,631,632,633,634,635,636,637,638,639,640,641,642,643,644,645,646,647,648,649,650,651,652,653,654,655,656,657,658,659,660,661,662,663,664,665,666,667,668,669,670,671,672,673,674,675,676,677,678,679,680,681,682,683,684,685,686,687,688,689,690,691,692,693,694,695,696,697,698,699,700,701,702,703,704,705,706,707,708,709,710,711,712,713,714,715,716,717,718,719,720,721,722,723,724,725,726,727,728,729,730,731,732,733,734,735,736,737,738,739,740,741,742,743,744,745,746,747,748,749,750,751,752,753,754,755,756,757,758,759,760,761,762,763,764,765,766,767,768,769,770,771,772,773,774,775,776,777,778,779,780,781,782,783,784,785,786,787,788,789,790,791,792,793,794,795,796,797,798,799,800,801,802,803,804,805,806,807,808,809,810,811,812,813,814,815,816,817,818,819,820,821,822,823,824,825,826,827,828,829,830,831,832,833,834,835,836,837,838,839,840,841,842,843,844,845,846,847,848,849,850,851,852,853,854,855,856,857,858,859,860,861,862,863,864,865,866,867,868,869,870,871,872,873,874,875,876,877,878,879,880,881,882,883,884,885,886,887,888,889,890,891,892,893,894,895,896,897,898,899,900,901,902,903,904,905,906,907,908,909,910,911,912,913,914,915,916,917,918,919,920,921,922,923,924,925,926,927,928,929,930,931,932,933,934,935,936,937,938,939,940,941,942,943,944,945,946,947,948,949,950,951,952,953,954,955,956,957,958,959,960,961,962,963,964,965,966,967,968,969,970,971,972,973,974,975,976,977,978,979,980,981,982,983,984,985,986,987,988,989,990,991,992,993,994,995,996,997,998,999,1000,1001,1002,1003,1004,1005,1006,1007,1008,1009,1010,1011,1012,1013,1014,1015,1016,1017,1018,1019,1020,1021,1022,1023,1024,1025,1026,1027,1028,1029,1030,1031,1032,1033,1034,1035,1036,1037,1038,1039,1040,1041,1042



```

CONST3=-DEL/180.D0
CD$(1,1)=6.D0*CONST3
CD$(1,2)=0.D0
CD$(1,3)=-CONST3
CD$(1,4)=-4.D0*CONST3
CD$(1,5)=-CONST3
CD$(1,6)=0.D0
CD$(2,1)=0.D0
CD$(2,2)=32.D0*CONST3
CD$(2,3)=0.D0
CD$(2,4)=16.D0*CONST3
CD$(2,5)=16.D0*CONST3
CD$(2,6)=16.D0*CONST3
CD$(3,1)=0.D0
CD$(3,2)=0.D0
CD$(3,3)=6.D0*CONST3
CD$(3,4)=0.D0
CD$(3,5)=-CONST3
CD$(3,6)=-4.D0*CONST3
CD$(4,1)=16.D0*CONST3
CD$(4,2)=0.D0
CD$(4,3)=32.D0*CONST3
CD$(4,4)=0.D0
CD$(4,5)=16.D0*CONST3
CD$(5,1)=-CONST3
CD$(5,2)=-4.D0*CONST3
CD$(5,3)=-CONST3
CD$(5,4)=0.D0
CD$(5,5)=6.D0*CONST3
CD$(5,6)=0.D0
CD$(6,1)=0.D0
CD$(6,2)=16.D0*CONST3
CD$(6,3)=16.D0*CONST3
CD$(6,4)=16.D0
CD$(6,5)=32.D0*CONST3
CD$(6,6)=CD$(1,2)
CD$(7,7)=CD$(1,3)
CD$(7,8)=CD$(1,4)
CD$(7,9)=CD$(1,5)
CD$(7,10)=CD$(1,6)
CD$(7,11)=CD$(2,1)
CD$(7,12)=CD$(2,2)
CD$(8,8)=CD$(2,3)
CD$(8,9)=CD$(2,4)
CD$(8,10)=CD$(2,5)
CD$(8,11)=CD$(2,6)

```

```

TIME0759
TIME0760
TIME0761
TIME0762
TIME0763
TIME0764
TIME0765
TIME0766
TIME0767
TIME0768
TIME0769
TIME0770
TIME0771
TIME0772
TIME0773
TIME0774
TIME0775
TIME0776
TIME0777
TIME0778
TIME0779
TIME0780
TIME0781
TIME0782
TIME0783
TIME0784
TIME0785
TIME0786
TIME0787
TIME0788
TIME0789
TIME0790
TIME0791
TIME0792
TIME0793
TIME0794
TIME0795
TIME0796
TIME0797
TIME0798
TIME0799
TIME0800
TIME0801
TIME0802
TIME0803
TIME0804
TIME0805
TIME0806

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```

300 CONTINUE
   IF(NNQXY.EQ.0) GO TO 310
   DO 310 I=1,NNQXY
     RHS(NQS(I))=RHS(NQS(I))+Q(NQS(I))+65.962D0
   RHS(NQS(I))+NN)=RHS(NQS(I)+NN)+Q(NQS(I)+NN)
   CONTINUE
310 IF(NNQZC.EQ.0) GO TO 312
   DO 312 I=1,NNQZC
     RHS(NQS(NNQXY+I)+2*NN)=RHS(NQS(NNQXY+I)+2*NN)+Q(NQS(NNQXY+I)+2*NN)
   CONTINUE
312 IF(NNQZ.EQ.0) GO TO 311
   DO 311 I=1,NNQZ
     RHS(NQS(NNQXY+NNQZC+I)+MM)=RHS(NQS(NNQXY+NNQZC+I)+MM)+
     1Q(NQS(NNQXY+NNQZC+I)+MM)
   CONTINUE
311 CONTINUE

MODIFICATION OF RHS FOR TM AND CD BOUNDARY CONDITIONS

DO 315 I=1,MMM
DO 315 J=1,NTOTVP
  JX=NVIS(J)
  RHS(I)=RHS(I)-TM(I,JX)*X(JX)
  TM(I,JX)=0.D0
  TM(JX,I)=0.D0
CONTINUE
315 IF(T.GT.0.D0) GO TO 321
DO 320 I=1,NTOTVP
  K=NVIS(I)
  Y(1,K)=X(K)
DO 316 J=1,MMM
  CD(J,K)=0.D0
  CD(K,J)=0.D0
  CD(K,K)=1.D0
  RHS(K)=0.D0
CONTINUE
316 NY=117
NL=0
M=70
JSKF=0
MAXDER=6
IPRT=1
HMIN=1.D-12
HMAX=5.D-02
RMSEPS=1.D-03
IF(T-0.D0) 323,323,321
323 GO TO 325

```

```

TIME0903
TIME0904
TIME0905
TIME0906
TIME0907
TIME0908
TIME0909
TIME0910
TIME0911
TIME0912
TIME0913
TIME0914
TIME0915
TIME0916
TIME0917
TIME0918
TIME0919
TIME0920
TIME0921
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TIME0937
TIME0938
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TIME0940
TIME0941
TIME0942
TIME0943
TIME0944
TIME0945
TIME0946
TIME0947
TIME0948
TIME0949
TIME0950

```



```

321 TEND=T
325 CONTINUE
      CALL SDESOL(Y,YL,T,TEND,NY,NL,M,JSKF,MAXDER,IPRT,H,HMIN,HMAX,
1      IRMSEPS,W)
      IF(T.GT.5.D-02) GO TO 324
      DO 322 J=1,MMM
      TDIFF=DABS(T1(J)-Y(1,J))
      T1(J)=Y(1,J)
      EPSLN=1.D-06
      IF(TDIFF-EPSLN) 322,177,177
322 CONTINUE
324 WRITE(NWRITE,2000)
      DO 360 I=1,MMM
      WRITE(NWRITE,2005) I,Y(1,I)
360 CONTINUE
      WRITE(NWRITE,2021)
      WRITE(NWRITE,2025)
      WRITE(NWRITE,2030)
      WRITE(NWRITE,2035)
      WRITE(NWRITE,2040)
      WRITE(NWRITE,2045)
      WRITE(NWRITE,2050)
      WRITE(NWRITE,2055)
      FCRMAT(110)
500 FCRMAT(111),18X,'TIME-DEPENDENT FLUID MECHANICS PROBLEM',////)
600 FCRMAT(3110)
1005 FCRMAT(6X,A4,I10,2F10.0)
1006 FCRMAT(7110)
1010 FCRMAT(6X,A4,I10,F10.0)
1015 FCRMAT(6X,A4,I10)
1016 FCRMAT(6X,A4,I10)
1020 FCRMAT(110,2F10.0)
1025 FCRMAT(6X,A4,I10,F10.0)
1030 FCRMAT(6X,A4,2110,2F10.0)
1034 FCRMAT(5X,'NNQZC=',I3,/)
1035 FCRMAT(5X,'NO. OF CORNERS=',I3,/)
1036 FCRMAT(5X,'NNVELS=',I3,/)
1037 FCRMAT(5X,'NNQXY=',I3,/)
1038 FCRMAT(5X,'NNPS=',I3,/)
1039 FCRMAT(5X,'NNTS=',I3,/)
1040 FCRMAT(5X,'NNQZ=',I3,/)
1041 FCRMAT(5X,'SUMMARY OF NODAL COORDINATES',/,/,
17X,I,12X,X(I),13X,Y(I),/)
1045 FCRMAT(5X,I3,13,2(7X,F10.3))
1050 FCRMAT(5X,'LISTING OF SYSTEM TOPOLOGY',/,/,5X
1,ELEMENT NUMBER,20X,'NODE NUMBERS',/)
1055 FCRMAT(5X,I3,10X,6(5X,I3))

```

```

TIME0951
TIME0952
TIME0953
TIME0954
TIME0955
TIME0956
TIME0957
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TIME0960
TIME0961
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TIME0988
TIME0989
TIME0990
TIME0991
TIME0992
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TIME0994
TIME0995
TIME0996
TIME0997

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```

80 BLOCK OF AUXILLARY STORAGE, AND OBTAIN INITIAL VALUES OF
90 DERIVATIVES.
100 THE CALLING SEQUENCE FOR SDESOL IS
110
120 CALL SDESOL(Y,YL,T,TEND,NY,NL,M,JSKF,MAXDER,IPRT,H,HMIN,HMAX,RMSEPS,W)
130
140 WHERE THE PARAMETERS ARE DEFINED AS FOLLOWS.
150
160 Y - ARRAY DIMENSIONED (7,NY). THIS ARRAY CONTAINS THE
170 DEPENDENT VARIABLES AND THEIR SCALED DERIVATIVES.
180 Y(J+1,I) CONTAINS THE J-TH DERIVATIVE OF THE I-TH VAR
190 IABLE TIMES H**J/J-FACTORIAL, WHERE H IS THE CURRENT
200 STEPSIZE. ON FIRST ENTRY THE CALLER SUPPLIES THE
210 INITIAL VALUES OF EACH VARIABLE IN Y(1,I). ON SUB-
220 SEQUENT ENTRIES IT IS ASSUMED THE ARRAY HAS NOT
230 BEEN CHANGED. TO INTERPOLATE TO NON-MESH POINTS,
240 THESE VALUES CAN BE USED AS FOLLOWS. IF H IS THE
250 CURRENT STEPSIZE AND VALUES AT TIME T+E ARE
260 NEEDED, LET S = E/H AND THEN
270
280 I-TH VARIABLE AT T+E IS SUM Y(J+1,I)*S**J
290 J=0
300
310 THE VALUE OF JS IS OBTAINED IN THE CALLING PROGRAM
320 BY JS = IABS(JSKF/10), WHICH APPEAR LINEARLY.
330 ARRAY OF NL VARIABLES WHICH APPEAR LINEARLY.
340 - CURRENT VALUE OF THE INDEPENDENT VARIABLE (TIME)
350 - END TIME OF DIFFERENTIAL EQUATIONS AND NONLINEAR
360 - VARIABLES.
370 - NUMBER OF LINEAR VARIABLES
380 - NUMBER OF VARIABLES INCLUDED IN THE ERROR TEST
390 - AN INDICATOR USED BOTH ON INPUT AND OUTPUT
400 - ON INPUT, JSKF = -1 INDICATES A RESTART CALL TO
410 SDESOL. JSKF = 0 INDICATES AN INITIAL CALL TO THE
420 SDESOL. JSKF > 0 INDICATES A CONTINUATION OF THE
430 PREVIOUS CALL TO SDESOL. JSKF < -1 MAY HAVE RETURNS
440 FROM THE SOL. CALL TO SDESOL. JSKF < -1 MAY HAVE RETURNS
450 FROM THE SOL. CALL TO SDESOL. JSKF < -1 MAY HAVE RETURNS
460 RESULTS IN TERMINATION.
470 APPROPRIATE COMMENT.
480 ON OUTPUT, JSKF CONSISTS OF TWO DIGITS AND SIGN,
490 + OR - QP. Q IS THE ORDER OF THE FORMULA CURRENTLY
500 BEING USED. P INDICATES THE TYPE OF RETURN, AS
510 FOLLOWS.
520 JSKF > 0, P = 1 IS THE NORMAL RETURN
530 JSKF < 0, P IS AN ERROR RETURN, WITH THE FOLLOWING
540
550

```



```

MEANINGS.
P = 1      ERROR TEST FAILED FOR H > HMIN
P = 3      CORRECTOR FAILED TO CONVERGE FOR H > HMIN
P = 4      CORRECTOR FAILED TO CONVERGE FOR FIRST
          ORDER METHOD
P = 5      ERROR RETURN FROM SUBROUTINE NUTSL
P = 6      ERROR RETURN FROM SUBROUTINE Derval
          METHOD. IT MUST BE NO GREATER THAN SIX.
          INTERNAL PRINT NO CONTROL INDICATOR FOR LCASUB.
          IPRT = 0
          IPRT > 0
          PRINT COUNTERS, STEPSIZE, CURRENT TIMES
          AND VALUES OF DEPENDENT VARIABLES AT
          EACH STEP.
          CURRENT STEPSIZE. THE ONE INITIAL VALUE MUST BE SUPPLIED
          BUT NEED NOT BE THOSE A SMALLER ONE IF NECESSARY TO
          SUBROUTINE WILL PER STEP TO UNDERSTIMATE THE INITIAL
          KEEP THE ERROR BETTER TO UNDERSTIMATE THE INITIAL
          VALUE. IT IS BETTER TO UNDERSTIMATE IT. THE STEPSIZE IS
          STEPSIZE NOT CHANGED BY THE USER.
          MINIMUM STEPSIZE ALLOWED
          MAXIMUM STEPSIZE ALLOWED
          THE SINGLE TEST ERROR TO CURRENT ESTIMATES, ER(I), DIVIDED BY
          YMAX(I) = (MAXIMUM THE STEPSIZE AND/OR THE ORDER
          LESS THAN EPS. ACHIEVE THIS.
          ARE VARIED TO STORAGE ARRAY. MUST BE AT LEAST 13*NY + 5*NL
          SCRATCH STORAGE PLUS THOSE REQUIRED FOR STORAGE OF THE
          LOCATIONS, (SEE DESCRIPTION OF SUBROUTINE JACMAT).
          MATRIX PW (SEE PW WILL NORMALLY REQUIRE NO MORE THAN
          THE STORAGE OF PW LOCATIONS, AND IF COMPACT STORAGE TECH-
          Niques ARE USED, CAN BE MUCH FEWER.
          -----
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION Y(7,1), YL(1), W(1)
IF (JSKF.GT.0) GO TO 120
IF (JSKF.LT.-1) GO TO 140
N = NY+NL
IF (JSKF.LT.0) GO TO 110
          IF THIS IS THE FIRST ENTRY, OBTAIN VALUES OF THE DERIVATIVES.
          CALL Derval (Y, YL, T, N, NY, W, KRETR)
          IF (KRETR.NE.0) GO TO 130
          NOW SET UP STORAGE BLOCKS IN THE W ARRAY.  THIS NEEDS TO BE DONE

```

```

          SDE 970
          SDE 980
          SDE 1010
          SDE 1020

```





1030 SDE  
1040 SDE  
1050 SDE  
1060 SDE  
1070 SDE  
1080 SDE  
1090 SDE  
1100 SDE  
1110 SDE  
1120 SDE  
1130 SDE

ONLY INITIALLY AND ON RESTARTS.

THE ARRAY SAVE STARTS AT LOCATION  
THE ARRAY YLSV STARTS AT LOCATION  
THE ARRAY YMAX STARTS AT LOCATION  
THE ARRAY ESV STARTS AT LOCATION  
THE ARRAY FI STARTS AT LOCATION  
THE ARRAY DY STARTS AT LOCATION  
THE MATRIX PW STARTS AT LOCATION

110 NSVL = 7\*NY+1  
NYMAX = NSVL+NL  
NER = NYMAX+NY  
NESV = NER+NY  
NFI = NESV+NY  
NDY = NFI+N  
NPW = NDY+N  
JS = JSKF

120 CALL LDASUB (Y,YL,T,TEND,N,NY,M,JS,KE,MAXDER,IPRT,H,HMIN,HMAX,  
1 RMSEPS,W,W(NSVL),W(NYMAX),W(NER),W(NESV),W(NFI),W(NDY),W(NPW))

CODE JSKF ON RETURN FROM LDASUB

1240 SDE  
1250 SDE  
1260 SDE

JSKF = ISIGN(JS\*10+IABS(KF),KF)

RETURN

130 JSKF = -6

RETURN

140 PRINT 1, JSKF

STOP

1330 SDE  
1340 SDE

1 FORMAT ('OIT IS AN ERROR TO ENTER SDESOL WITH JSKF = ',I10//  
1 RUN HAS BEEN TERMINATED.')  
END

SUBROUTINE LDASUB (Y,YL,T,TEND,N,NY,M,JS,KE,MAXDER,IPRT,H,  
1 HMIN,HMAX,RMSEPS,SAVE,YLSV,YMAX,ER,ESV,FI,DY,PW)

30 LDA  
40 LDA  
50 LDA  
60 LDA  
70 LDA  
80 LDA  
90 LDA  
100 LDA  
110 LDA  
120 LDA  
130 LDA

SUBROUTINE LDASUB IS A MODIFICATION OF SUBROUTINE DFASUB  
WHICH IS DUE TO R. L. BROWN AND C. W. GEAR. DFASUB IS DOCUMENTED  
IN THE REPORT

DOCUMENTATION FOR DFASUB--

BY R. L. BROWN AND C. W. GEAR  
REPORT UIUCDCS-R-73-575, JULY 1973  
UNIVERSITY OF ILLINOIS AT URBANA-CHAMPAIGN  
URBANA, ILLINOIS 61801

THIS REPORT IS AVAILABLE FROM THE NATIONAL TECHNICAL INFORMATION  
SERVICE OF THE U. S. DEPARTMENT OF COMMERCE UNDER ACCESSION NUMBER



CC0-1469-225.

THE MODIFICATION HERE IS DOCUMENTED IN THE REPORT  
A PROGRAM FOR THE NUMERICAL SOLUTION OF LARGE SPARSE SYSTEMS OF  
ALGEBRAIC AND IMPLICITLY DEFINED STIFF DIFFERENTIAL EQUATIONS  
BY RICHARD FRANK  
REPORT NPS53FE76051, MAY 1976  
NAVAL POSTGRADUATE SCHOOL  
MONTEREY, CALIFORNIA 93940

-----  
THE CALLING SEQUENCE FOR LDASUB IS

CALL LDASUB(Y, YL, T, TEND, N, NY, M, JSTART, KFLAG, MAXOR, IPRT, H, HMIN,  
HMAX, RMSEPS, SAVE, YLSV, YMAX, ER, ESV, FL, DY, PW)

WHERE THE PARAMETERS ARE DEFINED AS FOLLOWS.  
- ARRAY DIMENSIONED (7, NY). THIS ARRAY CONTAINS THE  
DEPENDENT VARIABLES AND THEIR SCALED DERIVATIVES.  
Y(J+1, I) CONTAINS THE J-TH DERIVATIVE OF THE I-TH VARIABLE  
IABLE SIZE. H\*J/J-FACTORY, WHERE H IS THE CURRENT  
STEPSIZE. ON FIRST ENTRY THE CALLER SUPPLIES THE  
INITIAL VALUES OF EACH VARIABLE OF THE CURRENT  
ESTIMATE OF THE INITIAL VALUES OF THE DERIVATIVES  
IN Y(2, I). ON SUBSEQUENT ENTRIES IT IS ASSUMED THAT  
THE ARRAY Y HAS NOT BEEN CHANGED. TO INTERPOLATE TO  
NON-MESH POINTS, THESE VALUES CAN BE USED AS FOLLOWS.  
IF H IS THE CURRENT STEP SIZE AND VALUES AT TIME T+E  
NEEDED, LET S = E/H AND THEN

I-TH VARIABLE AT T+E IS  
SUM Y(J+1, I)\*S\*\*J  
J=0

THE VALUE OF NQ IS OBTAINED IN THE CALLING PROGRAM  
BY NQ = JSTART.

- YL - ARRAY OF NL = N - NY VARIABLES WHICH APPEAR LINEARLY.
- T - THE USER SUPPLIES INITIAL VALUES FOR THESE VARIABLES.
- TEND - CURRENT VALUE OF THE INDEPENDENT VARIABLE (TIME)
- N - END TIME
- NY - TOTAL NUMBER OF VARIABLES
- M - NUMBER OF DIFFERENTIAL EQUATIONS AND NONLINEAR  
VARIABLES.
- NUMBER OF VARIABLES INCLUDED IN THE ERROR TEST.  
  THIS NUMBER CAN BE NO GREATER THAN NY. IF IT IS  
  GREATER THAN NY, NY VARIABLES ARE USED IN THE ERROR

LDA 140  
LDA 150  
LDA 160  
LDA 170  
LDA 180  
LDA 190  
LDA 200  
LDA 210  
LDA 220  
LDA 230  
LDA 240  
LDA 250  
LDA 260  
LDA 270  
LDA 280  
LDA 290  
LDA 300  
LDA 310  
LDA 320  
LDA 330  
LDA 340  
LDA 350  
LDA 360  
LDA 370  
LDA 380  
LDA 390  
LDA 400  
LDA 410  
LDA 420  
LDA 430  
LDA 440  
LDA 450  
LDA 460  
LDA 470  
LDA 480  
LDA 490  
LDA 500  
LDA 510  
LDA 520  
LDA 530  
LDA 540  
LDA 550  
LDA 560  
LDA 570  
LDA 580  
LDA 590  
LDA 600  
LDA 610





```

JSTART - TEST* AND OUTPUT INDICATOR. FOLLOWING MEANINGS. PREVIOUS
          INPUT ON INPUT <0 THIS INDICATES A TERMINATION FROM A PREVIOUS
          POINT FOLLOWING A TERMINATION OF THE RUN OR
          SOLUTION OF ANOTHER PROBLEM DURING THE SAME
          RUN. PARAMETERS IN THE CALLING SEQUENCE
          MUST HAVE BEEN PRESERVED FROM THE PREVIOUS
          USE, PARTICULARLY THE ARRAYS
          USE, SAVE, YLSV, ES, V, AND PW. AFTER A CALL
          TO SUBROUTINE LDASAV, WHICH ALSO SAVES
          NECESSARY PARAMETERS, INTERNAL TO LDASUB. THE
          INDICATES AN INITIAL CALL TO LDASUB. THE
          ROUTINE INITIALIZES ITSELF, SCALES THE
          DERIVATIVES IN Y(2:I) AND THEN PERFORMS THE
          INTEGRATION UNTIL TEND. TO BE CONTINUED.
          INDICATES THE SOLUTION IS NEITHER
          AFTER THE INITIAL ENTRY IT IS NEITHER
          DESIRABLE NOR NECESSARY TO RE-INITIALIZE
          JSTART = 0, SINCE THIS RE-INITIALIZES
          THE CODE, BEGINNING WITH A FIRST ORDER
          METHOD AGAIN. SET TO THE VALUE OF NQ, THE
          ON OUTPUT, JSTART IS CURRENTLY BEI- & O &
          ORDER OF THE FORMULA INDICATOR, WITH THE FOLLOWING
          THE COMPLETION CODE
          MEANINGS
          +1 THE INTEGRATION WAS SUCCESSFUL
          -1 ERROR TEST FAILED FOR H > HMIN
          -3 CORRECTOR FAILED TO CONVERGE FOR H > HMIN
          -4 ORDER METHOD FAILED TO CONVERGE FOR FIRST
          -5 ORDER METHOD
          MAXOR - ERROR RETURN FROM SUBROUTINE NUTSL
          METHOD - ORDER DERIVATIVE THAT SHOULD BE USED IN THE
          GREATER THAN SIX, NO GREATER THAN SIX. IF IT IS
          INTERNAL PRINT SIX, THE MAXIMUM ORDER USED WILL BE SIX.
          = 0 NO PRINT CONTROL INDICATOR
          > 0 PRINT CONTROL
          AND PRINT COUNTERS, STEPSIZE, CURRENT TIME
          AND VALUES OF DEPENDENT VARIABLES AT
          EACH STEP. AN INITIAL VALUE MUST BE SUPPLIED
          CURRENT STEPSIZE, THE ONE WHICH WILL BE USED, SINCE THE
          SUBROUTINE WILL CHOOSE A SMALLER THAN THE SPECIFIED
          KEEP THE ERROR STEP TO UNDERESTIMATE THE INITIAL
          VALUE. IT IS BETTER TO UNDERESTIMATE THE STEPSIZE IS
          STEPSIZE THAN TO OVERESTIMATE IT.
          NORMALLY NOT CHANGED BY THE USER.

```

```

LDA 620
LDA 630
LDA 640
LDA 650
LDA 660
LDA 670
LDA 680
LDA 690
LDA 700
LDA 710
LDA 720
LDA 730
LDA 740
LDA 750
LDA 760
LDA 770
LDA 780
LDA 790
LDA 800
LDA 810
LDA 820
LDA 830
LDA 840
LDA 850
LDA 860
LDA 870
LDA 880
LDA 890
LDA 900
LDA 910
LDA 920
LDA 930
LDA 940
LDA 950
LDA 960
LDA 970
LDA 980
LDA 990
LDA 1000
LDA 1010
LDA 1020
LDA 1030
LDA 1040
LDA 1050
LDA 1060
LDA 1070
LDA 1080
LDA 1090

```



```

HMIN          MINIMUM STEPSIZE ALLOWED
HMAX          STEPSIZE ALLOWED
RMSEPS       THE SINGLE STEP ERROR ESTIMATES, THE ROOT-MEAN-SQUARE OF
              YMAX(I) = (MAXIMUM OF CURRENT ESTIMATES, ER(I), DIVIDED BY
              LESS THAN RMSEPS. THIS. STEPSIZE AND/OR ORDER ARE
              VARIED TO ACHIEVE THIS.
              AN ARRAY OF LENGTH AT LEAST 7*NY
              AN ARRAY OF LENGTH AT LEAST NL
              A VECTOR OF LENGTH NY ON THE FIRST CALL, THESE WILL
              BE INITIALY SIZED AS YMAX(I) = MAX(1, Y(1, I)),
              A VECTOR OF LENGTH NY
              A VECTOR OF LENGTH NY = NY + NL
              A VECTOR OF LENGTH N = NY + NL
              AN ARRAY IN WHICH THE J MATRIX COMPUTED
              IN SUBROUTINE JACMAT WILL BE STORED. SIZE WHICH
              MUST BE ALLOWED FOR IT, BUT NORMALLY WON'T BE MORE THAN
              N**2 + 2*N LOCATIONS. THE LATTER 2*N BEING REQUIRED
              BY THE LINEAR EQUATION SOLVER.
-----
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION Y(7,1), YL(1), SAVE(7,1), YMAX(1), ER(1), YLSV(1), F1(1)
PERT(6,3), COF(21), ESV(1), PW(1), SAV(1), A(29)
1 EQUIVALENCE (A(8),BND), (A(9),BR), (A(10),E), (A(11),EDWN),
2 (A(12),ENQ1), (A(13),ENQ2), (A(14),ENQ3), (A(15),EPS), (A(16),EUP)
3 (A(17),HNEW), (A(18),PEPSH), (A(19),IDCUB), (A(20),IWEVAL),
4 (A(21),K), (A(22),LCOPYL), (A(23),LCOPYR), (A(24),MAXDER),
  (A(25),M1), (A(26),NL), (A(27),NQ), (A(28),NS), (A(29),NW)
-----

```

```

LDA 1410
LDA 1420
LDA 1430
LDA 1440
LDA 1450
LDA 1460
LDA 1470
-----
THE COEFFICIENTS IN THE PERT ARRAY ARE USED FOR ERROR TESTING AND
CHANGING STEPSIZE AND NEED TO BE ACCURATE TO ONLY A FEW DIGITS.
-----
DATA PERT/4.0,9.0,16.0,25.0,36.0,49.0,9.0,16.0,25.0,36.0
1,49.0,64.0,1.0,1.0,1.0,2.5,2.7889D-2,1.70569D-3,6.83929D-5/
-----

```

```

LDA 1500
LDA 1510
LDA 1520
LDA 1530
LDA 1540
LDA 1550
LDA 1560
-----
THE ENTRIES IN THE COF ARRAY ARE THE COEFFICIENTS FOR THE STIFFLY
STABLE METHODS USED IN THIS PROGRAM AND ARE TO BE THE MACHINE
PRECISION EQUIVALENTS OF THE FOLLOWING CONSTANTS.
-----

```

-1











```

C      E IS A TEST FOR ERRORS OF THE CURRENT ORDER NQ      LDA 2020
C      EUP IS TO TEST FOR INCREASING THE ORDER, EDWN FOR DECREASING THE      LDA 2030
C      ORDER.      LDA 2040
C      LDA 2050
140  K = NQ*(NQ-1)/2
      CALL COPYZ (A(2),COF(K+1),NQ)
      K = NQ+1
      IDOUB = NQ
      ENQ1 = .5D0/NQ
      ENQ2 = .5D0/K
      ENQ3 = .5D0/(NQ+2)
      PEP3H = EPS**2
      E = PERT(NQ,1)*PEPSH
      EUP = PERT(NQ,2)*PEPSH
      ECWN = PERT(NQ,3)*PEPSH
      BND = (EPS*ENQ3)**2
      IWEVAL = 1
      GO TO IRET, (190,200,490,570)
150  IF (H.EQ.HNEW) GO TO 190
      IF CALLER HAS CHANGED H, RESCALE DERIVATIVES TO REFLECT THAT HNEW
      WAS USED ON THE LAST CALL.
      R = H/HNEW
      ASSIGN 190 TO IRET
      GC TO 610
C      LDA 2210
C      LDA 2220
C      LDA 2230
C      LDA 2240
160  JSTART = NQ
      HNEW = H
      RETURN
170  NS = NS+1
      IF (IPRT.LE.0) GO TO 180
      PRINT DATA IF DESIRED BY USER
C      LDA 2370
C      LDA 2380
C      LDA 2390
180  PRINT 1, NS,NW,NQ,H,T,(Y(1,I),I=1,NY)
      IF (NL.GT.0) PRINT 2, (YL(I),I=1,NL)
      CONTINUE
      IF (KFLAG.LT.0) GO TO 160
      IF (T.GE.TEND) GO TO 160
      TAKE ANOTHER STEP IF T < TEND
      JSTART = 1
C      LDA 2450
C      LDA 2460
C      LDA 2470
C      LDA 2490

```



```

C      SAVE DATA FOR TRIAL WITH A SMALLER TIMESTEP IF THIS STEP FAILS      LDA 2500
C      -----LDA 2510
C      190 CALL COPYZ (SAVE,Y,LCOPYY)
C      CALL COPYZ (YLSV,YL,LCOPYL)
C      RACUM = 1.00
C      KFLAG = 1
C      HOLD = H
C      NCOLD = NQ
C      TOLD = T
C      T = T+H
C      200 HINV = 1.00/H
C      -----
C      COMPUTE PREDICTED VALUES BY EFFECTIVELY MULTIPLYING DERIVATIVE
C      VECTOR BY PASCAL TRIANGLE MATRIX
C      -----
C      DO 210 J=2,K
C      J3 = K+J-1
C      LDA 2610
C      LDA 2620
C      LDA 2630
C      LDA 2640
C      LDA 2650
C
C      DO 210 J1=J,K
C      J2 = J3-J1
C      LDA 2680
C
C      DO 210 I=1,NY
C      210 Y(J2,I) = Y(J2,I)+Y(J2+1,I)
C      LDA 2710
C
C      DO 220 I=1,NY
C      220 ER(I) = 0.00
C      LDA 2740
C      LDA 2750
C
C      DO UP TO THREE CORRECTOR ITERATIONS. CONVERGENCE IS OBTAINED WHEN
C      CHANGES ARE LESS THAN BND WHICH IS DEPENDENT ON THE ERROR TEST
C      CONSTANT. THE SUM OF CORRECTIONS IS ACCUMULATED IN ER(I). IT IS
C      EQUAL TO THE K-TH DERIVATIVE OF Y TIMES H**K/(K-FACTORIAL*A(K)),
C      AND THUS IS PROPORTIONAL TO THE ACTUAL ERRORS TO THE LOWEST POWER
C      OF H PRESENT, WHICH IS H**K.
C      -----
C      LDA 2780
C      LDA 2790
C      LDA 2800
C      LDA 2810
C      LDA 2820
C      LDA 2830
C      LDA 2840
C      LDA 2850
C      LDA 2860
C      LDA 2870
C
C      DO 270 L=1,3
C      CALL DIFFUN (Y,YL,T,HINV,DY)
C      IF (IWEVAL.LT.1) GO TO 230
C      -----
C      IF THERE HAS BEEN A CHANGE OF ORDER OR THERE HAS BEEN TROUBLE
C      WITH CONVERGENCE, PW IS RE-EVALUATED PRIOR TO STARTING THE
C      CORRECTOR ITERATION. IWEVAL IS THEN SET TO -1 AS AN INDICATOR
C      THAT IT HAS BEEN DONE. NEWPW IS SET NONZERO TO INDICATE TO
C      SUBROUTINE NUTSL THAT A NEW PW HAS BEEN PROVIDED.
C      -----
C      LDA 2910
C      LDA 2920
C      LDA 2930
C      LDA 2940
C      LDA 2950
C      LDA 2960
C      LDA 2970

```









```

GO TO 170
-----LDA 3470
C THE CORRECTOR CONVERGED, SO NOW THE ERROR TEST IS MADE.-----LDA 3480
C-----LDA 3490
330 D = 0.00
C
C DO 340 I=1,M1
C YM = DMAX1(DABS(Y(1,I)),YMAX(I))
C 340 D = D+(ER(I)/YM)**2
C
C IWEVAL = 0
C IF (D.GT.E) GO TO 380
C
C-----LDA 3580
C THE ERROR TEST IS OKAY, SO THE STEP IS ACCEPTED. IF IDOUB
C NOW BECOMES NEGATIVE, A TEST IS MADE TO SEE IF THE STEP SIZE
C CAN BE INCREASED AT THIS ORDER OR ONE HIGHER OR ONE LOWER.
C THE CHANGE IS MADE ONLY IF THE STEP CAN BE INCREASED BY AT
C LEAST 10%. IDOUB IS SET TO NQ TO PREVENT FURTHER TESTING
C FOR A WHILE. IF NO CHANGE IS MADE, IDOUB IS SET TO 9.
C-----LDA 3640
C IF (K.LT.3) GO TO 360
C-----LDA 3650
C
C DO 350 J=3,K
C
C DO 350 I=1,NY
C 350 Y(J,I) = Y(J,I)+A(J)*ER(I)
C
C KFLAG = 1
C IDOUB = IDOUB-1
C IF (IDOUB) 410,370,510
C 370 CALL COPYZ (ESV,ER,M1)
C GO TO 510
C
C-----LDA 3780
C THE ERROR TEST FAILED. IF JSTART = 0, THE DERIVATIVES IN THE
C SAVE ARRAY ARE UPDATED. TESTS ARE THEN MADE TO FIX THE STEPSIZE
C AND PERHAPS REDUCE THE ORDER. AFTER RESTORING AND SCALING THE
C Y VARIABLES, THE STEP IS RETRIED.
C-----LDA 3790
C-----LDA 3800
C-----LDA 3810
C-----LDA 3820
C-----LDA 3830
C 380 IF (JSTART.GT.0) GO TO 400
C
C DO 390 I=1,NY
C 390 SAVE(2,I) = Y(2,I)
C
C KFLAG = KFLAG-2
C IF (H.LE.HMIN) GO TO 550
C 400
C IF (KFLAG.LE.-5) GO TO 530
C 410 PR2 = (D/E)**ENQ2*1.200

```





```

C
L = 0
IF (NQ.LE.1) GO TO 430
D = 0.D0
LDA 3970

C
DO 420 J=1,M1
YM = DMAX1(DABS(Y(1,J)), YMAX(J))
420 D = D+(Y(K,J)/YM)**2
LDA 4010

C
PR1 = (D/EDWN)**ENQ1*1.3D0
IF (PR1.GE.PR2) GO TO 430
PR2 = PR1
L = -1
430 IF (KFLAG.LT.0.OR.NQ.GE.MAXDER) GO TO 450
D = 0
LDA 4080

C
DO 440 J=1,M1
YM = DMAX1(DABS(Y(1,J)), YMAX(J))
440 D = D+((ER(J)-ESV(J))/YM)**2
LDA 4120

C
PR1 = (D/EUP)**ENQ3*1.4D0
IF (PR1.GE.PR2) GO TO 450
PR2 = PR1
L = 1
450 R = 1.D0/DMAX1(PR2,1.D-5)
IF (KFLAG.LT.0.OR.R.GE.1.1D0) GO TO 460
IDUB = 9
GO TO 510
460 NEWQ = NQ+L
K = NEWQ+1
IF (NEWQ.LE.NQ) GO TO 480
R1 = A(NEWQ)/DFLOAT(NEWQ)

C
DO 470 J=1,NY
470 Y(K,J) = ER(J)*R1
LDA 4250

C
480 CONTINUE
LDA 4280

C
IF THE STEP WAS OKAY, SCALE THE Y VARIABLES IN ACCORDANCE
WITH THE NEW VALUE OF H. IF KFLAG < 0, HOWEVER, USE THE
4300 LDA 4310
4310 LDA 4320
4320 LDA 4330
4330 LDA 4340
4340 LDA 4350
4350 LDA 4360
-----
IDUB = NQ
IF (NEWQ.EQ.NQ) GO TO 490
NC = NEWQ
490 ASSIGN 490 TO IRET
GO TO 140

```



```

490 IF (KFLAG.GT.0) GO TO 500
    RACUM = RACUM*R
    GO TO 560
500 R = DMAX1(DMIN1(HMAX/H,R),HMIN/H)
    H = H*R
    IWEAL = 1
    ASSIGN 510 TO IRET
    GO TO 610
C
510 DO 520 I=1,M1
520 YMAX(I) = DMAX1(DABS(Y(1,I)),YMAX(I))
C
    GO TO 170
C-----LDA 4500
C THE ERROR TEST HAS NOW FAILED THREE TIMES, SO THE DERIVATIVES ARE
C IN BAD SHAPE. RETURN TO FIRST ORDER METHOD AND TRY AGAIN. OF
C COURSE, IF NQ = 1 ALREADY, THEN THERE IS NO HOPE AND WE EXIT WITH
C KFLAG = -4.-----LDA 4530
C-----LDA 4550
530 IF (NQ.EQ.1) GO TO 540
    NQ = 1
    ICQUB = 1
    ASSIGN 570 TO IRET
    GO TO 140
540 NCOLD = 1
    KFLAG = -4
    GO TO 320
550 KFLAG = -1
    GO TO 170
C-----LDA 4710
C THIS SECTION RESTORES THE SAVED VALUES OF Y AND YL, SCALING THE
C Y DERIVATIVES AS NECESSARY, AND THEN RETURNS TO THE PREDICTOR LOOP.-----LDA 4720
C-----LDA 4730
C-----LDA 4740
560 H = HOLD*RACUM
    H = DMAX1(HMIN,DMIN1(H,HMAX))
570 RACUM = H/HOLD
    R1 = 1.00
C
    DC 580 J=2,K
    R1 = R1*RACUM
C
    DO 580 I=1,NY
580 Y(J,I) = SAVE(J,I)*R1
C
    DO 590 I=1,NY
590 Y(1,I) = SAVE(1,I)
C
    LDA 4790
    LDA 4820
    LDA 4850
    LDA 4860
    LDA 4890

```









```

1 FORMAT (2I5,I2,1P2E10.2,7E14.6/(32X,7E14.6))
2 FORMAT (32X,1P7E14.6)
3 FORMAT (1I,N=,13,1 NL =,13,1 RMSEPS =,1PE9.2,1 TEND =,
4 1E9.2,1 H =,E9.2//) H',8X,'T ',8X,'Y(1,*) AND YL(*)'//)
END
SUBROUTINE COPYZ(S,Y,L)
-----COP
THIS SUBROUTINE COPIES THE ARRAY Y, OF LENGTH L, INTO THE ARRAY S
-----COP
-----COP
-----COP
-----COP

```

C  
C  
C  
C

30  
40  
50  
60  
70

```

IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION S(1),Y(1)
IF(L.LE.0)RETURN
DO 100 J=1,L
  S(J) = Y(J)
100 RETURN
END

```

```

SUBROUTINE Derval (Y, YL, T, N, NY, W, KERET)
THIS SUBROUTINE SUPPLIES THE INITIAL VALUES OF THE DERIVATIVES
OF THE NODAL PARAMETERS TAKEN FROM STEADY STATE SYSTEM ANALYSIS.

```

C  
C

```

IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION Y(7,1), YL(1), W(1)
Y(2,7)=-9.428681D01
Y(2,8)=-9.079442D01
Y(2,9)=-9.428681D01
Y(2,12)=-1.012731D02
Y(2,13)=-2.894024D02
Y(2,14)=-1.012731D02
Y(2,17)=-1.057558D02
Y(2,18)=-3.616863D01
Y(2,19)=-1.057558D02
Y(2,22)=-7.0622577D01
Y(2,23)=-3.162976D02
Y(2,24)=-7.062277D01
Y(2,27)=-1.322337D02
Y(2,28)=-3.825550D01
Y(2,29)=-1.322337D02
Y(2,42)=0.D0
Y(2,43)=0.D0
Y(2,44)=0.D0
Y(2,47)=0.D0
Y(2,48)=0.D0
Y(2,49)=0.D0
Y(2,52)=0.D0
Y(2,53)=0.D0
Y(2,54)=0.D0

```













```

RETURN
END
SUBROUTINE DIFFUN(Y, YL, T, HINV, DY)
THIS SUBROUTINE EVALUATES THE SYSTEM'S GOVERNING SET OF
EQUATIONS AT A GIVEN TIME AND GIVEN VALUES OF THE NODAL
PARAMETER AND ITS DERIVATIVE.
IMPLICIT REAL*8(A-H,O-Z,$)
DIMENSION Y(7,1), YL(1), DY(1)
COMMON CD(117,117), TM(117,117), C(117)
DO 200 I=1,117
  DY(I)=-C(I)
DO 100 J=1,117
  DY(I)=DY(I)+CD(I,J)*Y(2,J)*HINV+TM(I,J)*Y(1,J)
CONTINUE
RETURN
END
SUBROUTINE JACMAT(Y, YL, T, HINV, A2, N, NY, EPS, DY, F1, PW)
THIS SUBROUTINE EVALUATES THE JACOBIAN MATRIX AT THE GIVEN TIME
AND AT CURRENT VALUES OF THE DEPENDENT VARIABLES, ORDER,
AND STEP SIZE.
IMPLICIT REAL*8(A-H,O-Z,$)
DIMENSION Y(7,1), PW(117,1)
COMMON CD(117,117), TM(117,117), C(117)
AH=-A2*HINV
DO 200 I=1,117
  DO 200 J=1,117
    PW(I,J)=TM(I,J)+AH*CD(I,J)
  RETURN
END

```

C C C

C C C





## LIST OF REFERENCES

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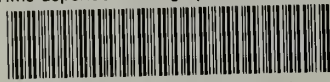


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